

Brookfield Properties Community Heat Pump Study: Chiller Plant Optimization

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Brookfield Properties Community Heat Pump Study: Chiller Plant Optimization

Final Report

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Abstract

This report presents the preliminary findings of a Smith Engineering study commissioned by Brookfield Properties and NYSERDA to perform a community heat pump feasibility study at the 250 Vesey St facility in New York, NY.

Keywords

thermal energy network, district energy, heat recovery, electrification, decarbonization, feasibility study, lifecycle cost analysis

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Acronyms and Abbreviations

ΔP	differential pressure
ΔT	differential temperature
\$/klbs	dollars per kilo-pounds
%Sp	% speed
$^{\circ}C$	degrees Celsius
$^{\circ}F$	degrees Fahrenheit
A	amps/area
AAV	automatic air vent
ABV CLG	above finished ceiling
ACU	air conditioning unit
AFF	above finished floor
AHU	air handling unit
AP	access panel
BAS	building automation system
BD	balancing damper
BFF	below finished floor
BMS	burner management system
BTU	British thermal units
BTUH	BTU per hour
BYP	bypass
CAC	control air compressor
CD	ceiling diffuser
CF	cubic feet
CFH	cubic feet per hour
CFM	cubic feet per minute
CHW	chilled water
CHWP	chilled water pump
CHWR	chilled water return
CHWS	chilled water supply
CO ₂	carbon dioxide
CO ₂ e	carbon dioxide emissions

COND	condensate
Cond	condenser
CR	cold room
CU	condensing unit; copper
CV	coefficient of valve
CW	condenser water
CWP	condenser water pump
CWR	condenser water return
CWS	condenser water supply
CWST	condenser water supply temperature
DAT	discharge air temperature
DB	dry-bulb
DDC	direct digital controls
DG	door grille
DIA	diameter
Dmd	demand
Dp	dew point
DP	differential pressure
Dpr	damper
DTW	dual temperature water
DTWR	dual temperature water return
DTWS	dual temperature water supply
EA	each
EAT	entering air temperature
EC	evaporative cooler
EDH	electric duct heater
EF	exhaust fan
Eff	efficiency
EG	exhaust grille
EH	exhaust hood
EMCS	energy management control system
ER	exhaust register
ESP	external static pressure
Evap	evaporator
EWT	entering water temperature
F	flow
FCU	fan coil unit
FD	fire damper
FG	fire grille

FL DR	floor drain
FPM	feet per minute
FT WG	feet of water, gauge
Ft	feet
FTU	fan terminal unit
FW	feed water
G	glycol
GA	gauge
GAL	gallons
GALV	galvanized
GPH	gallons per hour
GPM	gallons per minute
H	enthalpy
H ₂ O	water
HG	hot gas
HHW	heating hot water
HHWP	heating hot water pump
HHWR	heating hot water return
HHWST	heating hot water supply temperature
HP	heat pump; horsepower
HR	hour
HVAC	heating, ventilation, and air conditioning
HW	hot water
HX	heat exchanger
I.D.	inside diameter
IN. WG	inches of water, gauge
In.	inches
klbs	kilo pounds
kW	kilowatt
kW/ton	kilowatt per ton
kWh	kilowatt hour
kWh/ton	kilowatt per ton
LAT	leaving air temperature
LB	pound
LD	linear diffuser
LPS	low-pressure steam
LWT	leaving water temperature
MA	mixed air
MAT	mixed air temperature

MBH	thousand BTUH
MCF	thousands of Cubic Feet
MD	motorized damper
MT	megatons
N.O.	normally open
NC	normally closed
NIC	not in contract
No.	number
NPLV	nominal part load value
NPSHa	net positive suction head available
NPSHr	net positive suction head required
NTS	not to scale
OA	outside air
OAL	outdoor air louver
OAT	outdoor air temperature
OC	on center
OD	outside diameter
PF	power factor
PG	process glycol
PH	phase
Po	position
Press	pressure
PSI	pounds per square inch
PSIG	pounds per square inch gauge
RA	return air
RAG	return air grille
RAR	return air register
RAT	return air temperature
RD	round diffuser
ReH	ReHeat
RH	relative humidity
RL	refrigerant liquid
RPM	revolutions per minute
RS	refrigerant suction
RV	roof vent
RW	river water
SA	supply air
SAR	supply air register
SAT	supply air temperature

SD	smoke damper
SF	supply fan; square feet
SG	soffit grille
SIM	similar
SP	static pressure
SPEC	specification
St	status
STD	standard
STL	steel
Stm	steam
TDH	total dynamic head
TEMP	temperature
TES	thermal energy storage
TG	transfer grille
TSP	total static pressure
TYP	typical
UC	undercut door
UH	unit heater
V	valve; volts
VAV	variable air volume
VFD	variable frequency drive
VFM	venturi flow meter
VVU	variable volume unit
WB	wet-bulb
WPD	water pressure drop

Executive Summary

ES.1 Subject and Purpose

This report presents the preliminary findings of a Smith Engineering study commissioned by Brookfield Properties and NYSERDA to perform a community heat pump feasibility study at the 250 Vesey St. facility in New York, NY. The purpose of the heat pump system is to reduce the on-site steam usage of the 250 Vesey St. property in conjunction with New York City's energy usage goals. The heat pump system is intended to reduce the amount of steam usage by using the internal and external heat gains of the building which are inherently transferred to the heating, ventilation, and air conditioning (HVAC) chilled water system. Along with the new heat pumps, a new heating hot water distribution system would be provided to direct the heat toward existing HVAC and plumbing heating systems. The energy for this heat pump system would come from the electrical grid of New York City, which is in the process of becoming a more green energy source. In the end, the heat pump system will allow an overall greenhouse gas emission reduction at the building as shown by the tables and data within this report.

ES.2 Conclusion

The complex has a large amount of simultaneous heating and cooling, creating an opportunity for a heat pump application. The central heat pump option initially looked attractive since it could effectively leverage the site's year-round cooling loads. The site's potential carbon reduction by using the internal and solar heat gain is substantial. As electricity carbon content drops, the measures will improve the annual reduction and result in a much higher percentage of reduction of the total site.

Smith Engineering recommends proceeding with the following task: determining the system costs and completing Task 3 as outlined in the project scope.

Table ES-1 presents a financial summary of the proposed modifications, highlighting the projected costs and benefits of the heat pump system.

Table ES-1. Financial Summary of Modifications

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.1	Central Heat Pump	113,650	-4,739,842	4,800	-1,370	3,430	\$ (35,657,579)	\$ -	\$ (35,657,579)	\$ 2,971,526	12.0	\$ 919,313	9.2	\$ 39,892,695
M.2	Tower A Heat Pump	5,954	-271,945	251	-79	173	\$ (3,741,971)	\$ -	\$ (3,741,971)	\$ 118,944	31.5	\$ 46,334	22.6	\$ (532,686)
M.3	Tower B Heat Pump	11,324	-467,814	478	-135	343	\$ (3,210,609)	\$ -	\$ (3,210,609)	\$ 267,538	12.0	\$ 91,940	8.9	\$ 3,769,543
M.4	Tower C Heat Pump	2,974	-64,663	126	-19	107	\$ (3,400,390)	\$ -	\$ (3,400,390)	\$ 89,458	38.0	\$ 28,659	28.7	-1,106,849.5
M.5	Tower D Heat Pump	4,578	-104,168	193	-30	163	\$ (3,158,404)	\$ -	\$ (3,158,404)	\$ 135,183	23.4	\$ 43,750	17.6	316,027.0
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

1 Introduction

1.1 Subject and Purpose

Smith Engineering conducted a study that Brookfield Properties and the New York State Energy Research and Development Authority (NYSERDA) commissioned to evaluate the feasibility of heat pump systems at the 250 Vesey St. facility in New York, NY. The study examines two primary strategies for electrifying the heating system:

- Installing a centralized heat pump in the Central Plant
- Installing decentralized heat pumps in individual buildings

1.2 Scope of Work

The contracted scope of work includes the following tasks:

- Task 1: Survey the existing equipment and infrastructure
- Task 2: Energy model of proposed system operation
- Task 3: Economic model and summary
- Task 4: Schematic design
- Task 5: Prepare the final report

1.2.1 Task 1: Survey the Existing Equipment and Infrastructure

1. Collect equipment information
 - Gather available and relevant photos of equipment nameplates
 - Collect available and relevant equipment submittals and cut sheets
 - Assemble available and relevant drawing sheets with equipment schedules
 - Obtain available and relevant existing equipment performance test reports
2. Collect system information
 - Gather process and instrumentation drawings or one-line drawings
3. Collect controls data:
 - Capture photos/images of existing control system
 - Assemble existing sequence of operations
 - Gather available and relevant trend data
4. Collect utility information
 - Compile available and relevant utility bills and rate structure information
5. Develop an evaluation matrix of viable technology
6. Consolidate and present a summary of baseline energy use and life-cycle costs to NYSERDA, including the following deliverables

- Prepare the final report, including survey results within Task 5E, F, and G
- Produce an evaluation matrix summarizing four studied technologies, outlining positives, negatives, and scores for each technology; the matrix will determine which technology advances in the study

Table 1 lists the manufacturer and model numbers considered.

Table 1. List of Manufacturers and Models

Manufacturer	Model Name
JCI	OM
JCI	CYK
JCI	Frick
Vilter	—

1.2.2 Task 2: Energy Model of Proposed System Operation

1. Summarize analytical methods used for analysis
 - Describe the methods applied to model and compare system performance
2. Develop proposed model
 - Model each proposed system modification and compare the operation to the existing system operation model (base model)
 - Include the following system addition, with each measure requiring one model run and a result report) as defined below
 - Central heat pump
 - Perform model quality assurance and quality control
3. Submit the deliverables, which include the following
 - Weather-normalize 8,760 load profiles to TMY3 data
 - Chilled water load (heat pump evaporator load)
 - Heating hot water (HHW) load per building (heat pump condenser load)
 - Develop 8,760 temperature profiles
 - River water temperature profile (low temp for the heat pump lift)
 - Heating hot water temperature reset strategy for each building (high temp for the heat pump lift)
 - Calculate 8,760 system metrics
 - Steam reduction
 - Electrical energy consumption of the heat pump

- Perform cost analysis
 - Compare the cost of additional steam savings for the upper house with the cost of routing heating hot water piping to the upper house of each building
- Analyze heating hot water load distribution
 - Use available data to assess existing heating hot water load distribution between the upper house and lower house of the buildings

1.2.3 Task 3: Economic Model and Summary

1. Develop cost estimates for proposed system modification
 - Develop cost estimates using the following methodologies
 - Previous project data
 - Contractor quotes
 - Material quotes
2. Summarize the following per proposed modification; the total impact to the site may include:
 - Energy reduction per energy source
 - Demand reduction per energy source
 - Energy cost reduction per energy source
 - Demand cost reduction per energy source
 - Total energy and demand cost reduction
 - Demand response payment increase
 - Carbon reduction per energy source
 - Carbon tax reduction
 - Simple payback
3. Submit the deliverables, which include the following
 - Prepare cost-benefit analysis of the heat pump
 - This milestone will also include a deliverable of the cost-benefit analysis of smaller packaged heat pumps versus extending the large-scale heat pump distribution up the building to handle the upper house, including reasons why the chosen scale of technology is better for this application
 - As a part of this milestone, a first cost estimate will be generated for the project to move forward, and an energy calculation will be made for the energy savings and potential carbon emission reduction

1.2.4 Task 4: Schematic Design

1. Prepare schematic level mechanical and electrical drawing set for project concept
 - Scope: heat pump
 - Identifying existing/demolition plan views (major equipment only)

- Create new work plan views (major equipment only)
 - Develop mechanical one-line drawings
- Plant, piping distribution, and connection to the building
 - Develop electrical one-line drawings
- 2. Submit the deliverables, including
 - Analyze space requirements for each heat pump size and include findings in the Conclusions section of the report
 - Evaluate the existing underground tunnel network at the World Financial Center buildings that holds steam and chilled water (CHW) piping coming from the Central Plant
 - Assess requirements for adding additional heating hot water piping in these tunnels and determine how heating hot water will be pumped to each building

1.2.5 Task 5 Prepare the Final Report

1. Create a presentation, including:
 - Executive summary: Summarize options analyzed, economic analysis, and results
 - Purpose: Define the purpose of the report
 - Scope: Outline the scope of the project
 - Introduction: Provide an overview of the project
 - Existing facility description: Present detailed equipment information, system description, existing equipment curves, and operational data
 - Existing operation: Describe the existing load profile and existing equipment efficiency maps
 - Field observations: Document observations made during fieldwork
 - System modification options: Describe proposed modification description, economic analysis, feasibility
 - Conclusion and recommendations: Summarize conclusions and provide recommendations
 - Appendix: Include results impact and lessons learned
2. Submit the deliverables, including:
 - Provide the final step to proving the feasibility of this project before it moves forward
 - Submit a draft version of the Final Report to NYSERDA's Project Manager by the date specified in the Milestone Schedule of the NYSERDA Agreement for this step
 - NYSERDA will review the draft within 60 working days after receipt
 - Prepare and submit a final version of the report incorporating NYSERDA's comments within 30 working days after receiving feedback

2 Existing Infrastructure Summary

2.1 Building

This study focuses on the Brookfield Properties chilled water system, with a central chiller plant at 250 Vesey St., New York, NY. This analysis targets the chilled water system with an installed capacity of approximately 15,000 Tons. The central plant supplies chilled water to Tower A, Tower B, Tower C, Tower D, Winter Garden, and retail areas.

Figure 1. Site Image



2.2 Central Chilled Water Plant

The chilled water plant at 250 Vesey St. has 10 chillers, one chiller with 1,581 tons and the other 9 with 1,496 tons each, for a total plant capacity of 15,045 tons. The detailed existing system information is broken up into four sections: generating equipment, distribution equipment, utilization equipment, and system controls.

2.3 Chilled Water Generation

The chilled water generating equipment consists of water-cooled chillers and river water heat rejection. The chillers are tied to a closed loop condenser water system. This system is connected to 11 plate and frame heat exchangers that reject heat to the adjacent Hudson River.

Table 2 provides the water-cooled chiller specifications, including model numbers, nominal capacity, refrigerant type, year manufactured, efficiency metrics, and fluid operating conditions.

Table 2. Chiller Specifications

Water Cooled Chillers											
Chiller ID		CH-1	CH-2	CH-3	CH-4	CH-5	CH-6	CH-7	CH-8	CH-9	CH-10
Location		250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant
Service		CHW	CHW	CHW	CHW	CHW	CHW	CHW	CHW	CHW	CHW
Make		Carrier	Carrier	Carrier	Carrier	Carrier	Carrier	Carrier	Carrier	Carrier	Carrier
Model Number		19FA583	19FA561-B-500-5929-L-EC	19XR-8987E65ENB64	19XR-8987E65ENB64	19XR-8987E65ENB64	19FA561-B-500-5929-L-EC	19XR-8987E65ENB64	19XR-8987E65ENB64	19XR-8987E65ENB64	19XR-8987E65ENB64
Serial Number		34501	34502	-	-	-	35759	-	-	-	-
Nominal Capacity	Tons	1,581	1,581	1,440	1,440	1,440	1,581	1,440	1,440	1,440	1,440
Specified Net Capacity	Tons	1,572	1,581	1,435	1,435	1,435	1,581	1,435	1,435	1,435	1,435
Original Refrigerant		R-500	R-500	R-500	R-500	R-500	R-500	R-500	R-500	R-500	R-500
Current Refrigerant		R-134A	R-134A	R-134A	R-134A	R-134A	R-134A	R-134A	R-134A	R-134A	R-134A
Year Manufactured		-	1984	2018	2018	2018	1984	2018	2018	2018	2018
Compressor											
Type		Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal
Model		19FA583	19FA561BEC	19XR-E65	19XR-E65	19XR-E65	19FA561BEC	19XR-E65	19XR-E65	19XR-E65	19XR-E65
Stages		2	2	2	2	2	2	2	2	2	2
Driver - Electric Motor											
Make		-	-	-	-	-	-	-	-	-	-
Motor Control Type		Starter	Starter	VFD	VFD	VFD	Starter	VFD	VFD	VFD	VFD
Input Power	kW	970	1,083	757	757	757	1,083	757	757	757	757
Motor Nameplate Power	HP	-	-	-	-	-	-	-	-	-	-
Voltage	V	460	460	460	460	460	460	460	460	460	460
Speed	RPM	3,571	9,440	-	-	-	9,440	-	-	-	-
Full Load Efficiency	kW/Ton	0.62	0.68	0.53	0.53	0.53	0.68	0.53	0.53	0.53	0.53
Refrigerant Evaporator											
Fluid		Water	Water	Water	Water	Water	Water	Water	Water	Water	Water
Fluid Flow	GPM	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750
Fluid Pressure Drop	Ft of H2O	18.8	18.6	20.5	20.5	20.5	18.6	20.5	20.5	20.5	20.5
Entering Fluid Temp.	°F	52.72	55.80	54.52	54.52	54.52	55.80	54.52	54.52	54.52	54.52
Leaving Fluid Temp.	°F	39.00	42.00	42.00	42.00	42.00	42.00	42.00	42.00	42.00	42.00
Fluid Passes	No.	2	2	2	2	2	2	2	2	2	2
Approach	°F	-	-	-	-	-	-	-	-	-	-
Design Working Pressure	PSIG	250	250	250	250	250	250	250	250	250	250
Refrigerant Condenser											
Fluid		Water	Water	Water	Water	Water	Water	Water	Water	Water	Water
Fluid Flow	GPM	4,285	4,285	5,035	5,035	5,035	4,285	5,035	5,035	5,035	5,035
Fluid Pressure Drop	Ft of H2O	25.3	24.2	44.3	44.3	44.3	24.2	44.3	44.3	44.3	44.3
Entering Fluid Temp.	°F	79.00	79.00	79.00	79.00	79.00	79.00	79.00	79.00	79.00	79.00
Leaving Fluid Temp.	°F	89.44	89.60	86.89	86.89	86.89	89.60	86.89	86.89	86.89	86.89
Fluid Passes	No.	2	2	2	2	2	2	2	2	2	2
Approach	°F	-	-	-	-	-	-	-	-	-	-
Design Working Pressure	PSIG	250	250	250	250	250	250	250	250	250	250

2.4 Thermal Energy Storage

The thermal energy storage (TES) includes thirteen tanks with a total storage capacity of 3.3 million gallons of chilled water. Four heat exchangers manage the charging and discharging the thermal energy storage tanks. Table 3 provides specifications for the thermal energy storage tanks. Table 4 provides specifications for the thermal storage tank pumps. Table 5 provides specifications for the heat exchangers serving the thermal storage tanks.

Table 3. Thermal Energy Storage Tank Specifications

TES Tanks														
Fan ID	Tank 1	Tank 2	Tank 3	Tank 4	Tank 5	Tank 6	Tank 7	Tank 8	Tank 9	Tank 10	Tank 11	Tank 12	Tank 13	
Location	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	
Service	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	Thermal Storage	
Capacity	Gallons	284,493	262,788	262,788	258,973	264,647	264,647	269,075	283,830	283,830	283,753	277,247	272,137	270,859

Table 4. Thermal Energy Storage Pump Specifications

Pumps										
Pump ID			P-25P	P-26P	P-27P	P-38P	P-39P	P-40P	P-41P	
Location			250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	
Service			Open Loop TES CHW	Open Loop TES CHW	Open Loop TES CHW	Closed Loop TES CHW	Closed Loop TES CHW	Closed Loop TES CHW	Closed Loop TES CHW	
Make			Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	
Model Number			10FN / 4310	10FN / 4310	10FN / 4310	10G / 4310	10G / 4310	10G / 4310	10G / 4310	
Serial Number			113428	113429	-	113408	113409	113410	113411	
Design	Pump									
	Type		Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal
	Style		Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line
	Fluid		Water	Water	Water	Water	Water	Water	Water	Water
	Impeller Diameter	Inches	10.6	10.6	10.6	12.0	12.0	12.0	12.0	12.0
	Stages	No.	1	1	1	1	1	1	1	1
	Design Working Pressure	PSIG	175	175	175	175	175	175	175	175
	Fluid Flow	GPM	3,030	3,030	3,030	3,225	3,225	3,225	3,225	3,225
	Head (TDH)	Ft of H2O	80	80	80	45	45	45	45	45
	NPSHr	Ft of H2O	16.0	16.0	16.0	19.0	19.0	19.0	19.0	19.0
	Design Point Efficiency	%	82.0%	82.0%	82.0%	81.0%	81.0%	81.0%	81.0%	81.0%
	Driver - Electric Motor									
	Make		Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon
	Motor Control Type		VFD	VFD	VFD	Starter	Starter	Starter	Starter	Starter
	Motor Insulation Class		F	F	F	B	B	B	B	B
Overloading	Yes/No	No	No	No	No	No	No	No	No	
Motor Nameplate Power	HP	100	100	100	50	50	50	50	50	
Voltage	V	460	460	460	460	460	460	460	460	
Speed	RPM	1,175	1,175	1,175	1,185	1,185	1,185	1,185	1,185	
Nominal Efficiency	%	93.0%	93.0%	93.0%	92.4%	92.4%	92.4%	92.4%	92.4%	

Table 5. Thermal Energy Storage Heat Exchanger Specifications

Plate and Frame Heat Exchanger						
Heat Exchanger ID		PHX-11	PHX-12	PHX-13	PHX-14	
Location		250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	
Service		TES HX	TES HX	TES HX	TES HX	
Make		Alfa Laval	Alfa Laval	Alfa Laval	Alfa Laval	
Model Number		A20-BFG	A20-BFG	A20-BFG	A20-BFG	
Serial Number		-	-	-	-	
Specified Net Capacity	Tons	1,342	1,342	1,342	1,342	
Specified Net Capacity	MBH	16,100	16,100	16,100	16,100	
Design	Plates					
	Material		304 SS	304 SS	304 SS	304 SS
	Plate Thickness	Inches	0.02	0.02	0.02	0.02
	Installed Quantity	No.	684	684	684	684
	Maximum Quantity	No.	684	684	684	684
	Cold Side					
	Fluid		Water	Water	Water	Water
	Fluid Flow	GPM	2,300	2,300	2,300	2,300
	Fluid Pressure Drop	Ft of H2O	3.6	3.6	3.6	3.6
	Entering Fluid Temp.	°F	42.00	42.00	42.00	42.00
	Leaving Fluid Temp.	°F	56.00	56.00	56.00	56.00
	Approach	°F	7	7	7	7
	Design Working Pressure	PSIG	200	200	200	200
	Capacity	MBH	16,100	16,100	16,100	16,100
	Hot Side					
	Fluid		Water	Water	Water	Water
	Fluid Flow	GPM	3,230	3,230	3,230	3,230
	Fluid Pressure Drop	Ft of H2O	7.1	7.1	7.1	7.1
	Entering Fluid Temp.	°F	59.00	59.00	59.00	59.00
	Leaving Fluid Temp.	°F	49.00	49.00	49.00	49.00
Design Working Pressure	PSIG	200	200	200	200	
Capacity	MBH	16,150	16,150	16,150	16,150	
Min. Flow	GPM	-	-	-	-	
Max. Flow	GPM	-	-	-	-	

2.5 Cooling System Distribution

The distribution equipment consists of condenser water pumps, chilled water pumps, river water pumps, and thermal energy storage pumps.

2.5.1 Condenser Water

The condenser water is delivered to the chillers through heat exchangers by 10 condenser water pumps. Table 6 provides specifications on the condenser water pumps. Table 7 provides specifications on the river water pumps.

Table 6. Condenser Water Pump Specifications

Pumps												
Pump ID		P-1P	P-2P	P-3P	P-4P	P-5P	P-6P	P-7P	P-7A	P-7B	P-7C	
Location		250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	
Service		CW	CW	CW	CW	CW	CW	CW	CW	CW	CW	
Make		Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	
Model Number		10G / 4310	10G / 4310	10G / 4310	10G / 4310	10G / 4310	10G / 4310	10G / 4310	4300 12x12x13	4300 12x12x13	4300 12x12x13	
Serial Number		113417	113418	-	113420	113421	119319	119318	140892	140893	146972	
Pump												
Design	Type		Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	
	Style		Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	
	Fluid		Water	Water	Water	Water	Water	Water	Water	Water	Water	
	Impeller Diameter	Inches	12.0	12.0	12.0	12.0	12.0	12.0	12.0	12.4	12.4	12.4
	Stages	No.	1	1	1	1	1	1	1	1	1	1
	Design Working Pressure	PSIG	175	175	175	175	175	175	175	-	-	-
	Fluid Flow	GPM	4,285	4,285	4,285	4,285	4,285	4,285	4,285	4,285	4,285	4,285
	Head (TDH)	Ft of H2O	120	120	120	120	120	120	120	125	125	125
	NPSHr	Ft of H2O	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0	19.0
	Design Point Efficiency	%	83.0%	83.0%	83.0%	83.0%	83.0%	83.0%	83.0%	83.5%	83.5%	83.5%
	Driver - Electric Motor											
	Make		Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon
	Motor Control Type		Starter	Starter	Starter	Starter	Starter	Starter	VFD	Starter	VFD	Starter
	Motor Insulation Class		F	F	F	B	B	B	B	B	F	B
	Overloading	Yes/No	No	No	No	No	No	No	No	No	No	No
Motor Nameplate Power	HP	175	175	175	175	175	175	175	175	175	175	
Voltage	V	460	460	460	460	460	460	460	460	460	460	
Speed	RPM	1,775	1,775	1,780	1,780	-	1,775	1,780	1,780	1,780	1,775	
Nominal Efficiency	%	94.1%	94.1%	94.1%	91.4%	-	94.1%	91.4%	91.4%	94.1%	94.1%	

Table 7. River Water Pump Specifications

Pumps										
Pump ID		P-28P	P-29P	P-30P	P-31P	P-32P	P-33P	P-50P	P-51P	
Location		250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	
Service		River Water	River Water	River Water	River Water	River Water	River Water	River Water	River Water	
Make		Johnston	Johnston	Johnston	Johnston	Johnston	Johnston	Johnston	Johnston	
Model Number		22NMC	22NMC	22NMC	22NMC	22NMC	22NMC	22NMC	22NMC	
Serial Number		-	-	-	-	-	-	-	-	
Design	Pump									
	Type		-	-	-	-	-	-	-	
	Style		Vertical Turbine	Vertical Turbine	Vertical Turbine	Vertical Turbine	Vertical Turbine	Vertical Turbine	Vertical Turbine	
	Fluid		River Water	River Water	River Water	River Water	River Water	River Water	River Water	
	Impeller Diameter	Inches	14.0	14.0	14.0	14.0	14.0	14.0	14.0	
	Stages	No.	2	2	2	2	2	2	2	
	Design Working Pressure	PSIG	-	-	-	-	-	-	-	
	Fluid Flow	GPM	5,000	5,000	5,000	5,000	5,000	5,000	5,000	
	Head (TDH)	Ft of H2O	100	100	100	100	100	100	100	
	NPSHr	Ft of H2O	19.0	19.0	19.0	19.0	19.0	19.0	19.0	
	Design Point Efficiency	%	87.0%	87.0%	87.0%	87.0%	87.0%	87.0%	87.0%	
	Driver - Electric Motor									
	Make		U.S. Motor	U.S. Motor	U.S. Motor	U.S. Motor	U.S. Motor	U.S. Motor	U.S. Motor	U.S. Motor
	Motor Control Type		VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD
	Motor Insulation Class		F	F	F	F	F	F	F	F
	Overloading	Yes/No	Yes	Yes	Yes	Yes	Yes	Yes	Yes	Yes
	Motor Nameplate Power	HP	155	155	155	155	150	155	155	155
	Voltage	V	460	460	460	460	460	460	460	460
	Speed	RPM	1,180	1,180	1,180	1,180	1,185	1,185	1,180	1,180
Nominal Efficiency	%	93.0%	93.0%	93.0%	93.0%	93.0%	93.0%	93.0%	93.0%	

2.5.2 Chilled Water

The chilled water is circulated via a primary loop for the chillers and a secondary loop that provides chilled water to Towers A, B, C, and D; the Winter Garden; and retail areas. Table 8 and Table 9 provide specifications for the chilled water pumps.

Table 8. Primary Chilled Water Pump Specifications

Pumps												
Pump ID	P-8P	P-9P	P-10P	P-11P	P-12P	P-13P	P-14P	P-14A	P-14B	P-14C		
Location	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant		
Service	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW	Primary CHW		
Make	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong		
Model Number	10FN / 4310	10FN / 4310	10FN / 4310	10FN / 4310	10FN / 4310	10FN / 4310	10FN / 4310	4300 12x12x13	4300 12x12x13	4300 12x12x13		
Serial Number	113412	113413	113414	-	113416	119320	119321	140894	140895	146973		
Design	Pump											
	Type	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	
	Style	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	
	Fluid	Water	Water	Water	Water	Water	Water	Water	Water	Water	Water	
	Impeller Diameter	Inches	9.5	9.5	9.5	9.5	9.5	9.5	9.5	9.4	9.4	9.4
	Stages	No.	1	1	1	1	1	1	1	1	1	1
	Design Working Pressure	PSIG	175	175	175	175	175	175	175	-	-	-
	Fluid Flow	GPM	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750	2,750
	Head (TDH)	Ft of H2O	65	65	65	65	65	65	65	65	65	65
	NPSHr	Ft of H2O	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Design Point Efficiency	%	80.0%	80.0%	80.0%	80.0%	80.0%	80.0%	80.0%	80.0%	80.0%	80.0%
	Driver - Electric Motor											
	Make	Marathon	Marathon	Marathon	-	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon
	Motor Control Type	Starter	Starter	VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD
	Motor Insulation Class	B	B	B	-	B	B	B	B	B	B	B
	Overloading	Yes/No	No	No	No	No	No	No	No	No	No	No
	Motor Nameplate Power	HP	75	75	75	75	75	75	75	75	75	75
Voltage	V	460	460	460	460	460	460	460	460	460	460	
Speed	RPM	1,775	1,775	1,775	-	1,775	1,775	1,775	1,775	1,775	1,775	
Nominal Efficiency	%	91.7%	91.7%	91.7%	-	91.7%	91.7%	91.7%	91.7%	91.7%	91.7%	

Table 9. Secondary Chilled Water Pump Specifications

Pumps												
Pump ID	P-15P	P-16P	P-17P	P-18P	P-19P	P-20P	P-21P	P-22P	P-23P	P-24P		
Location	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant	250 Vesey Central Plant		
Service	Tower D CHW	Tower D CHW	Tower D CHW	Tower C CHW	Tower C CHW	Tower C CHW	Tower A & B CHW	Tower A & B CHW	Tower A & B CHW	Tower A & B CHW		
Make	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong	Armstrong		
Model Number	8GN / 4300	8GN / 4300	8GN / 4300	10GN / 4310	10GN / 4310	10GN / 4310	10GN / 4310	10GN / 4310	10GN / 4310	10GN / 4310		
Serial Number	119322	119323	-	113422	-	113424	119325	113425	113426	113427		
Design	Pump											
	Type	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	Centrifugal	
	Style	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	Vertical In-Line	
	Fluid	Water	Water	Water	Water	Water	Water	Water	Water	Water	Water	
	Impeller Diameter	Inches	11.5	11.5	11.5	10.8	10.8	10.8	12.3	11.8	11.8	11.8
	Stages	No.	1	1	1	1	1	1	1	1	1	1
	Design Working Pressure	PSIG	175	175	175	175	175	175	175	175	175	175
	Fluid Flow	GPM	1,935	1,935	1,935	3,100	3,100	3,100	3,675	3,675	3,675	3,675
	Head (TDH)	Ft of H2O	100	100	100	110	110	110	125	125	125	125
	NPSHr	Ft of H2O	17.5	17.5	17.5	18.0	18.0	18.0	20.0	20.0	20.0	20.0
	Design Point Efficiency	%	76.0%	76.0%	76.0%	81.0%	81.0%	81.0%	83.5%	83.5%	83.5%	83.5%
	Driver - Electric Motor											
	Make	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	Marathon	
	Motor Control Type	VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD	Starter	VFD	VFD
	Motor Insulation Class	B	B	B	B	B	B	B	B	B	B	B
Overloading	Yes/No	No	No	No	No	No	No	No	No	No	No	
Motor Nameplate Power	HP	75	75	75	125	125	125	150	150	150	150	
Voltage	V	460	460	460	460	460	460	460	460	460	460	
Speed	RPM	1,775	1,775	1,775	1,780	1,780	1,780	1,780	1,780	1,780	1,780	
Nominal Efficiency	%	91.7%	91.7%	91.7%	93.6%	93.6%	93.6%	93.6%	93.6%	93.6%	93.6%	

2.6 Chilled Water Use

The chilled water system serves Towers A, B, C, and D; the Winter Garden; and retail areas, primarily for space cooling through air handling units (AHUs) and each tower's heat exchangers. A network of piping delivers the chilled water to these towers, with access through an underground tunnel. Table 10 through Table 16 provide the specifications of the heat exchangers and air handling units for each tower, the Winter Garden, and the Central Plant.

2.6.1 Building Primary to Secondary Chilled Water Heat Exchangers

See Table 10 for the existing chilled water heat exchanger specifications.

Table 10. Towers A, B, C, and D, and Winter Garden: Heat Exchanger Specifications

Plate and Frame Heat Exchanger									
Heat Exchanger ID		PX-1A	PX-2A	PHX-1B	PHX-2B	PX-1C	PX-2C	PHX-1D	PHX-2D
Location		11th Flr MER	11th Flr MER	10th Flr MER	10th Flr MER	15th Flr MER	15th Flr MER	17th Floor MER	17th Floor MER
Service		Chilled Water	Chilled Water	Chilled Water	Chilled Water	SCHW	PCHW	Secondary CHW Loop	Secondary CHW Loop
Make		Alfa-Laval	Alfa-Laval	Alfa Laval	Alfa Laval	Alfa-Laval	Alfa-Laval	Alfa Laval	Alfa Laval
Model Number		A-20-BHA	A-20-BHA	AX30-FD	AX30-FD	A - 20 - BFD	A - 20 - BFD	A20-BFD	A20-BFD
Serial Number		-	-	3010067020	3010067021	3010065030	3010065029	30100-66339	30100-66881
Specified Net Capacity	Tons	1,714	1,714	2,124	2,124	2,188	2,188	1,503	1,503
Specified Net Capacity	MBH	20,570	20,570	25,484	25,484	26,256	26,256	18,038	18,038
Plates									
Material		-	-	AISI 304	AISI 304	-	-	304 SS	304 SS
Plate Thickness	Inches	-	-	0.04	0.04	-	-	-	-
Installed Quantity	No.	-	-	678	678	-	-	634	634
Maximum Quantity	No.	-	-	-	-	-	-	634	634
Cold Side									
Fluid		Water	Water	Water	Water	Water	Water	Water	Water
Fluid Flow	GPM	2,420	2,420	2,755	2,755	3,282	3,282	1,850	1,850
Fluid Pressure Drop	Ft of H2O	9.0	9.0	9.1	9.1	24.0	24.0	29.6	29.6
Entering Fluid Temp.	°F	42.00	42.00	42.00	42.00	42.00	42.00	42.00	42.00
Leaving Fluid Temp.	°F	59.00	59.00	60.50	60.50	58.00	58.00	61.50	61.50
Approach	°F	3.6	3.6	3.0	3.0	3.0	3.0	3.0	3.0
Design Working Pressure	PSIG	-	-	300	300	250	250	250	250
Capacity	MBH	20,570	20,570	25,484	25,484	26,256	26,256	18,000	18,000
Min. Flow	GPM	-	-	-	-	-	-	-	-
Max. Flow	GPM	-	-	-	-	-	-	-	-
Hot Side									
Fluid		Water	Water	Water	Water	Water	Water	Water	Water
Fluid Flow	GPM	2,110	2,110	2,755	2,755	3,282	3,282	1,900	1,900
Fluid Pressure Drop	Ft of H2O	11.7	11.7	9.1	9.1	24.0	24.0	30.0	30.0
Entering Fluid Temp.	°F	65.10	65.10	63.50	63.50	61.00	61.00	64.00	64.00
Leaving Fluid Temp.	°F	45.60	45.60	45.00	45.00	45.00	45.00	45.00	45.00
Design Working Pressure	PSIG	-	-	300	300	250	250	250	250
Capacity	MBH	20,573	20,573	25,484	25,484	26,256	26,256	18,000	18,000
Min. Flow	GPM	-	-	-	-	-	-	-	-
Max. Flow	GPM	-	-	-	-	-	-	-	-

Design Specifications

2.6.2 Tower A: Air Handling Units

Table 11. Towers A Chilled Water Air Handling Unit Specifications

Air Handling Unit				
AHU ID		AC-1A	AC-2A	
Location		Street Lvl Fan Room	Street Lvl Fan Room	
Service		Lobby	Retail Area	
Make		Joy	Joy	
Supply Air Flow	CFM	60,000	52,500	
Design	Supply Fan			
	Type	Axial	Axial	
	Maximum Fan Speed	RPM	1,770	1,770
	Air Flow	CFM	60,000	52,500
	Total Fan Static	In of H2O	5.5	6.3
	Fan Brake Horsepower	BHP	60	55
	Fan Efficiency	%	87%	94%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make		-	-
	Motor Control Type		VFD	VFD
	Motor Nameplate Power	HP	75	75
	Voltage	V	460	460
	Speed	RPM	1,770	1,770
	Nominal Efficiency	%	-	-
	Quantity	No.	1	1
	Cooling Coil			
	Total Capacity	MBH	3,500	2,610
	Total Capacity	Tons	292	218
	Sensible Capacity	MBH	1,938	1,695
	Fluid		Water	Water
	Fluid Flow	GPM	350	261
	Fluid Pressure Drop	Ft of H2O	10.6	13.1
	Entering Fluid Temp.	°F	42.00	42.00
	Leaving Fluid Temp.	°F	62.00	62.00
	Fluid Rows	No.	8	8
	Number of Coils	No.	-	-
	Air Flow	CFM	60,000	52,500
	Air Pressure Drop	In of H2O	0.7	1.1
	Entering Air Temp. - DB	°F	81.50	81.50
	Entering Air Temp. - WB	°F	68.00	68.00
	Leaving Air Temp. - DB	°F	51.60	51.60
	Leaving Air Temp. - WB	°F	51.50	51.50
Approach	°F	10	10	
Design Working Pressure	PSIG	-	-	

2.6.3 Tower B: Air Handling Units

Table 12. Towers B Chilled Water Air Handling Unit Specifications

AHU ID		Air Handling Unit						
		AC-1B	AC-2B	AC-45B	AC-47B	AC-48B		
Location		Cellar MER Area "C"	Street Lvl MER	Street Lvl MER	3rd Sub-Cellar MER	1st Sub-Cellar MER		
Service		Lobby & Street Lvl	Liberty Street Bridge	Primary Air 3rd to 15th Flrs	2nd & 3rd Sub Cellar Tennant Supply	1st Sub Cellar & Cellar Core Supply		
Make		Trane	Trane	Fanwall	Trane	Trane		
Model Number		Climate Changer	Climate Changer	-	Climate Changer	Climate Changer		
Serial Number		-	-	-	-	-		
Supply Air Flow	CFM	65,000	7,200	-	84,000	30,400		
OA Airflow	CFM	-	-	-	-	-		
Total External Static	In of H2O	6.00	4.50	-	4.75	3.75		
Design		Supply Fan						
		Type	-	-	-	-	-	
		Model	-	-	-	-	-	
		Air Flow	CFM	65,000	7,200	-	84,000	30,400
		Fan Static	In of H2O	6.00	4.50	-	4.75	3.75
		Fan Efficiency	%	-	-	-	-	-
		Quantity	No.	3	1	15	1	1
		Supply Fan Driver - Electric Motor						
		Make	-	-	Toshiba	-	-	
		Motor Control Type	VFD	VFD	VFD	Starter	Starter	
Motor Nameplate Power	HP	100	10	15	50	30		
Voltage	V	480	480	460	480	480		
Speed	RPM	865	1,287	1,770	935	1,116		
Nominal Efficiency	%	-	-	92.4%	-	-		
Quantity	No.	3	1	15	1	1		
Design		Cooling Coil						
		Total Capacity	MBH	3,261	407	8,398	2,989	1,080
		Total Capacity	Tons	-	-	-	-	-
		Sensible Capacity	MBH	2,099	247	-	2,159	781
		Fluid	Water	Water	Water	Water	Water	
		Fluid Flow	GPM	255	32	700	250	90
		Fluid Pressure Drop	Ft of H2O	8.70	10.40	24.90	6.80	2.30
		Entering Fluid Temp.	°F	42.00	42.00	45.00	45.00	45.00
		Leaving Fluid Temp.	°F	67.60	67.60	69.00	69.00	69.00
		Fluid Rows	No.	-	-	-	-	-
		Air Flow	CFM	-	-	-	-	-
		Air Pressure Drop	In of H2O	1.34	1.24	1.23	0.89	0.87
		Entering Air Temp. - DB	°F	81.50	80.80	92.00	80.80	80.80
		Entering Air Temp. - WB	°F	-	-	-	-	-
		Leaving Air Temp. - DB	°F	51.60	49.00	55.00	57.00	57.00
Leaving Air Temp. - WB	°F	-	-	-	-	-		
Approach	°F	10	7	10	12	12		
Design Working Pressure	PSIG	-	-	-	-	-		

2.6.4 Tower C: Air Handling Units

Table 13. Tower C Chilled Water Air Handling Unit Specifications

Air Handling Unit						
AHU ID		AC-1C	AC-2C	AHU-301C	AHU-303C	
Location		3rd Flr Fan Room	3rd Flr Fan Room	3rd Flr Fan Room	3rd Flr Fan Room	
Service		Lobby Levels	Retail Areas	Kitchen	Kitchen	
Make		Trane	Trane	-	-	
Supply Air Flow	CFM	63,000	117,500	-	-	
Design	Supply Fan					
	Type	Centrifugal	Centrifugal	-	-	
	Maximum Fan Speed	RPM	844	834	-	-
	Air Flow	CFM	63,000	58,750	-	-
	Total Fan Static	In of H2O	6.0	6.0	-	-
	Fan Brake Horsepower	BHP	77	72	-	-
	Fan Efficiency	%	78%	77%	-	-
	Quantity	No.	1	2	1	1
	Supply Fan Driver - Electric Motor					
	Make		-	-	-	-
	Motor Control Type		VFD	VFD	Starter	Starter
	Motor Nameplate Power	HP	100	100	-	-
	Voltage	V	460	460	-	-
	Speed	RPM	1,750	1,750	-	-
	Nominal Efficiency	%	-	-	-	-
	Quantity	No.	1	2	1	1
	Cooling Coil					
	Total Capacity	MBH	3,160	5,910	-	-
	Total Capacity	Tons	263	493	-	-
	Sensible Capacity	MBH	2,034	1,897	-	-
	Fluid		Water	Water	Water	Water
	Fluid Flow	GPM	316	591	-	-
	Fluid Pressure Drop	Ft of H2O	15.2	26.7	-	-
	Entering Fluid Temp.	°F	42.00	42.00	-	-
	Leaving Fluid Temp.	°F	62.00	62.00	-	-
	Fluid Rows	No.	8	8	-	-
	Number of Coils	No.	-	-	-	-
	Air Flow	CFM	63,000	117,500	-	-
	Air Pressure Drop	In of H2O	0.8	1.1	-	-
	Entering Air Temp. - DB	°F	81.50	81.50	-	-
	Entering Air Temp. - WB	°F	68.00	68.00	-	-
	Leaving Air Temp. - DB	°F	51.60	51.60	-	-
Leaving Air Temp. - WB	°F	51.50	51.50	-	-	
Approach	°F	10	10	-	-	
Design Working Pressure	PSIG	-	-	-	-	

2.6.5 Tower D: Air Handling Units

Table 14. Towers D Chilled Water Air Handling Unit Specifications

Air Handling Unit				
AHU ID		AC-1D	AC-2DN	AC-2DS
Location		+0'-6" Lvl MER	3rd Flr MER North	3rd Flr MER South
Service		Lobby	Courtyard North & East Area	Courtyard South & West Area
Make		Trane	Trane	Trane
Supply Air Flow	CFM	33,300	51,600	65,600
Design Specifications	Supply Fan			
	Type	Centrifugal	-	-
	Model	40-AFSW-21-10-11	44 AFD	49 AFD
	Design Fan Speed	RPM	1,136	-
	Maximum Fan Speed	RPM	1,283	819
	Air Flow	CFM	33,300	51,600
	Fan Static	In of H2O	3.9	3.3
	Fan Brake Horsepower	BHP	-	46
	Fan Efficiency	%	-	58%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make		-	Marathon
	Motor Control Type		VFD	VFD
	Motor Nameplate Power	HP	40	60
	Voltage	V	460	460
	Speed	RPM	1,725	1,785
	Nominal Efficiency	%	94.1%	95.0%
	Quantity	No.	1	1
	Chilled Water Coil			
	Total Capacity	MBH	-	1,881
	Total Capacity	Tons	-	157
	Sensible Capacity	MBH	-	1,594
	Fluid		Water	Water
	Fluid Flow	GPM	-	198
	Fluid Pressure Drop	Ft of H2O	-	4.3
	Entering Fluid Temp.	°F	-	42.00
	Leaving Fluid Temp.	°F	-	61.00
	Fluid Rows	No.	-	8
	Number of Coils	No.	-	-
	Air Flow	CFM	-	51,600
	Air Pressure Drop	In of H2O	-	0.7
	Entering Air Temp. - DB	°F	-	78.60
	Entering Air Temp. - WB	°F	-	62.80
Leaving Air Temp. - DB	°F	-	50.00	
Leaving Air Temp. - WB	°F	-	49.90	
Approach	°F	-	8	
Design Working Pressure	PSIG	-	-	

2.6.6 Winter Garden: Air Handling Units

Table 15. Winter Garden Chilled Water Air Handling Unit Specifications

Air Handling Unit									
AHU ID		AC-1W	AC-2W	AC-5W	AC-7W	AC-8W	AC-9W	AC-10W	AC-11W
Location		Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Service Lvl MER	Penthouse MER
Service		Winter Garden	Winter Garden	Winter Garden	Winter Garden	Winter Garden	Retail Area	Entrance Lobby East Side	Retail Area
Make		Trane	Trane	Trane	Trane	Trane	Trane	Trane	Trane
Model Number		-	-	-	-	-	-	-	-
Serial Number		-	-	-	-	-	-	-	-
Supply Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Supply Fan									
Type		-	-	-	-	-	-	-	-
Maximum Fan Speed	RPM	912	921	1,078	921	912	1,129	1,114	1,375
Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Total Fan Static	In of H2O	4.0	4.1	4.1	4.1	4.0	5.1	3.5	4.8
Fan Brake Horsepower	BHP	23	24	24	24	23	26	6	9
Fan Efficiency	%	81%	81%	80%	81%	81%	77%	63%	57%
Quantity	No.	1	1	1	1	1	1	1	1
Supply Fan Driver - Electric Motor									
Make		-	-	-	-	-	-	-	-
Motor Control Type		VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD
Motor Nameplate Power	HP	30	30	30	30	30	30	8	10
Voltage	V	460	460	460	460	460	460	460	460
Speed	RPM	1,750	1,750	1,750	1,750	1,750	1,750	1,750	1,750
Nominal Efficiency	%	-	-	-	-	-	-	-	-
Quantity	No.	1	1	1	1	1	1	1	1
Cooling Coil									
Total Capacity	MBH	1,134	1,134	1,133	1,134	1,134	1,241	229	346
Total Capacity	Tons	95	95	94	95	95	103	19	29
Sensible Capacity	MBH	823	823	823	823	823	797	176	223
Fluid		Water	Water	Water	Water	Water	Water	Water	Water
Fluid Flow	GPM	113	113	118	113	113	124	23	43
Fluid Pressure Drop	Ft of H2O	9.5	9.5	10.3	9.5	9.5	17.5	1.6	5.2
Entering Fluid Temp.	°F	42.00	42.00	42.00	42.00	42.00	42.00	42.00	42.00
Leaving Fluid Temp.	°F	62.00	62.00	61.20	62.00	62.00	62.00	62.00	58.00
Fluid Rows	No.	-	-	-	-	-	-	-	-
Number of Coils	No.	-	-	-	-	-	-	-	-
Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Air Pressure Drop	In of H2O	0.8	0.8	0.8	0.8	0.8	1.1	0.9	1.1
Entering Air Temp. - DB	°F	79.50	79.50	79.50	79.50	79.50	83.50	78.00	83.50
Entering Air Temp. - WB	°F	66.20	66.20	66.20	66.20	66.20	69.50	65.00	69.50
Leaving Air Temp. - DB	°F	54.00	54.00	54.00	54.00	54.00	54.00	54.00	54.00
Leaving Air Temp. - WB	°F	53.70	53.70	53.70	53.70	53.70	53.80	53.80	53.80
Approach	°F	12	12	12	12	12	12	12	12
Design Working Pressure	PSIG	-	-	-	-	-	-	-	-

Design

2.6.7 Central Plant: Air Handling Unit

Table 16. Central Plant Chilled Water Air Handling Unit Specifications

Air Handling Unit			
AHU ID	AC-5P		
Location	1st Sub-Cellar		
Service	1st Sub-Cellar Offices		
Make	Trane		
Supply Air Flow	CFM	1,915	
Design	Supply Fan		
	Design Fan Speed	RPM	-
	Maximum Fan Speed	RPM	1,372
	Air Flow	CFM	1,915
	Total Fan Static	In of H2O	2.3
	Fan Brake Horsepower	BHP	1
	Fan Efficiency	%	48%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make	-	
	Motor Control Type	-	
	Motor Nameplate Power	HP	2
	Voltage	V	460
	Speed	RPM	1,750
	Nominal Efficiency	%	-
	Quantity	No.	1
	Cooling Coil		
	Total Capacity	MBH	25
	Total Capacity	Tons	2
	Sensible Capacity	MBH	25
	Fluid	Water	
	Fluid Flow	GPM	7
	Fluid Pressure Drop	Ft of H2O	-
	Entering Fluid Temp.	°F	45.00
	Leaving Fluid Temp.	°F	52.00
	Fluid Rows	No.	6
	Number of Coils	No.	-
	Air Flow	CFM	1,915
	Air Pressure Drop	In of H2O	0.6
	Entering Air Temp. - DB	°F	80.00
	Entering Air Temp. - WB	°F	67.00
	Leaving Air Temp. - DB	°F	68.00
Leaving Air Temp. - WB	°F	57.00	
Approach	°F	23	
Design Working Pressure	PSIG	-	

2.7 Decentralized Steam and Heating Hot Water Use

Steam enters the site through 250 Vesey St. and is distributed through tunnels to the four towers, Central Plant, and the Winter Garden. The steam is used in air handling units, steam-to-hot water heat exchangers, heating and ventilation units, cabinet unit heaters, unit heaters, and steam-to-domestic hot water heaters. Table 17 through Table 50 provide detailed specifications for these systems, for both steam and heating hot water components. These systems are essential for maintaining optimal environmental conditions within the four towers, Winter Garden, and Central Plant. The tables cover a range of equipment designed to efficiently manage heating, cooling, and ventilation requirements across the facility, ensuring reliable operation and comfort.

2.7.1 Tower A

Table 11 through Table 23 provide detailed specifications for the systems within Tower A for both steam and heating hot water systems. These include steam-to-heating hot water heat exchangers, air handling units, heating and ventilating units, and unit heaters.

2.7.1.1 Steam-to-Heating Hot Water Heat Exchangers

Table 17. Tower A Steam-to-Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger									
Heat Exchanger ID		HX-1A	HX-2A	HX-3A	HX-4A	HX-5A	HX-6A	HX-7A	
Location		Tower A	Tower A	Tower A	Tower A	Tower A	Tower A	Tower A	
Service		HW Heater 22nd-40th Flrs	HW AC-41A	HW Heater 3rd-21st Flrs	HW Heater Service, Street, 2nd Flr	HW HV-1A & HV-2A	Condensate Recovery System	HW Heater Standby HX-3A & HX-4A	
Make		ITT	ITT	ITT	ITT	ITT	Yula	ITT	
Model Number		SU 144-2	SU 125-2	SU 144-2	SU 124-2	SU 144-2	HIVC-4J-156B	SU 144-2	
Serial Number		-	-	-	-	-	-	-	
Nominal Capacity	MBH	4,400	7,200	3,900	3,000	4,440	1,430	3,900	
Specified Net Capacity	MBH	4,400	7,260	3,900	3,000	4,400	1,430	3,900	
Shell Side - Steam									
Design Specifications	Fluid	Steam	Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow	LBS/Hr	4,400	7,200	3,900	3,000	4,440	1,430	3,900
	Entering Fluid Temp.	°F	302.10	302.10	302.10	302.10	302.10	302.10	302.10
	Entering Fluid Pressure	PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Entering Fluid Enthalpy	Btu/(Lbm-F)	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30
	Entering Fluid Deg Superheat	°F	83.60	83.60	83.60	83.60	83.60	83.60	83.60
	Leaving Fluid Temp.	°F	218.50	218.50	218.50	218.50	218.50	218.50	218.50
	Leaving Fluid Pressure	PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Leaving Fluid Enthalpy	Btu/(Lbm-F)	186.70	186.70	186.70	186.70	186.70	186.70	186.70
	Fluid Passes	No.	2	2	2	2	2	-	2
	Design Working Pressure	PSIG	-	-	-	-	-	-	-
	Tube Side - Water								
Design Specifications	Fluid	Water	Water	Water	Water	Water	Water	Water	
	Fluid Flow	GPM	440	440	390	300	440	220	390
	Fluid Pressure Drop	Ft of H2O	1.6	3.3	1.0	1.3	1.6	4.0	1.0
	Entering Fluid Temp.	°F	140.00	65.00	140.00	140.00	140.00	80.00	140.00
	Leaving Fluid Temp.	°F	160.00	98.00	160.00	160.00	160.00	93.00	160.00
	Fluid Passes	No.	2	2	2	2	2	4	2
	Approach	°F	-	-	-	-	-	-	-
	Design Working Pressure	PSIG	125	125	175	125	125	250	175
	Min. Flow	GPM	-	-	-	-	-	-	-
	Max. Flow	GPM	-	-	-	-	-	-	-

2.7.1.2 Tower A, Steam: Air Handling Units

Table 18. Tower A Steam Air Handling Unit Specifications

Air Handling Unit				
AHU ID		AC-1A	AC-2A	
Location		Street Lvl Fan Room	Street Lvl Fan Room	
Service		Lobby	Retail Area	
Make		Joy	Joy	
Supply Air Flow	CFM	60,000	52,500	
Design	Supply Fan			
	Type	Axial	Axial	
	Maximum Fan Speed	RPM	1,770	1,770
	Air Flow	CFM	60,000	52,500
	Total Fan Static	In of H2O	5.5	6.3
	Fan Brake Horsepower	BHP	60	55
	Fan Efficiency	%	87%	94%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make	-	-	-
	Motor Control Type	VFD	VFD	VFD
	Motor Nameplate Power	HP	75	75
	Voltage	V	460	460
	Speed	RPM	1,770	1,770
	Nominal Efficiency	%	-	-
	Quantity	No.	1	1
	Pre-Heat Coil			
	Capacity	MBH	3,078	2,805
	Fluid	Steam	Steam	Steam
	Steam Flow	Lbs/Hr	3,240	2,953
	Steam Inlet Pressure	PSIG	-	-
	Steam Temperature	°F	-	-
	Fluid Rows	No.	1	1
	Number of Coils	No.	-	-
	Air Flow	CFM	60,000	52,500
	Air Pressure Drop	In of H2O	0.2	-
	Entering Air Temp. - DB	°F	5.00	5.00
Leaving Air Temp. - DB	°F	55.00	55.00	
Approach	°F	-	-	
Design Working Pressure	PSIG	-	-	

2.7.1.3 Tower A, Steam: Unit Heaters and Cabinet Unit Heaters

Table 19. Tower A Steam Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters					
Unit Heater ID		UH-1A & 2A	UH-3A & 4A	UH-13A	
Location		Street Lvl	Street Lvl	Street Lvl	
Service		Loading Dock	Loading Dock	Street Lvl MER	
Make		Trane	Trane	Trane	
Model Number		60S	336P	60S	
Serial Number		-	-	-	
Quantity of Units		2	2	1	
Unit Heater Capacity		MBH	49	328	51
Design	Air Flow Data				
	Supply Air Flow	CFM	597	5,210	597
	Discharge Air Temp	°F	139.00	119.00	139.00
	Motor Data				
	Make		-	-	-
	Motor Control Type		Starter	Starter	Starter
	Motor Nameplate Power	HP	1/20	3/4	1/20
	Voltage	V	115	460	115
	Speed	RPM	1,100	1,120	1,100
	Nominal Efficiency	%	-	-	-
	Quantity	No.	1	1	1
	Heating Coil				
	Capacity	MBH	49	328	49
	Fluid		Steam	Steam	Steam
	Steam Flow	Lbs/Hr	52	345	52
	Steam Inlet Pressure	PSIG	-	-	-
	Steam Temperature	°F	-	-	-
	Fluid Rows	No.	-	-	-
	Number of Coils	No.	-	-	-
	Air Flow	CFM	597	5,210	597
	Air Pressure Drop	In of H2O	-	-	-
	Entering Air Temp. - DB	°F	-	-	-
	Leaving Air Temp. - DB	°F	139.00	119.00	139.00
Approach	°F	-	-	-	
Design Working Pressure	PSIG	-	-	-	

2.7.1.4 Tower A, Heating Hot Water: Air Handling Unit

Table 20. Tower A Heating Hot Water Air Handling Unit Specifications

Air Handling Unit			
AHU ID		AC-41A	
Location		Penthouse MER	
Service		OA to CUs	
Make		Trane	
Supply Air Flow	CFM	145,000	
Design Specifications	Supply Fan		
	Type	Centrifugal	
	Maximum Fan Speed	RPM	516
	Air Flow	CFM	145,000
	Fan Static	In of H2O	4.5
	Fan Brake Horsepower	BHP	134
	Fan Efficiency	%	77%
	Quantity	No.	-
	Supply Fan Driver - Electric Motor		
	Make	Lincoln	
	Motor Control Type	-	
	Motor Nameplate Power	HP	150
	Voltage	V	460
	Speed	RPM	1,785
	Nominal Efficiency	%	-
	Quantity	No.	-
	Reclaim Coil		
	Capacity	MBH	6,539
	Fluid	Water	
	Fluid Flow	GPM	4,087
	Fluid Pressure Drop	Ft of H2O	27.3
	Entering Fluid Temp.	°F	61.00
	Leaving Fluid Temp.	°F	64.20
	Fluid Rows	No.	6
	Number of Coils	No.	-
	Air Flow	CFM	145,000
	Air Pressure Drop	In of H2O	0.8
Entering Air Temp. - DB	°F	92.00 / 5.00	
Entering Air Temp. - WB	°F	76.00 / -	
Leaving Air Temp. - DB	°F	66.50 / 50.00	
Leaving Air Temp. - WB	°F	65.80 / -	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.1.5 Tower A, Heating Hot Water: Heating and Ventilating Units

Table 21. Tower A Heating Hot Water Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit				
H&V ID		HV-1A	HV-2A	
Location		Service Level MER	Service Level Fan Room	
Service		Garage Ventilation	Service Level Ventilation	
Make		Trane	Trane	
Supply Air Flow	CFM	67,200	3,300	
Design	Supply Fan			
	Type	-	-	
	Maximum Fan Speed	RPM	461	1,150
	Air Flow	CFM	67,200	3,300
	Total Fan Static	In of H2O	2.0	2.0
	Fan Brake Horsepower	BHP	31	2
	Fan Efficiency	%	68%	58%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make	-	-	
	Motor Control Type	Starter	Starter	
	Motor Nameplate Power	HP	40	3
	Voltage	V	460	460
	Speed	RPM	1,750	1,750
	Nominal Efficiency	%	-	-
	Quantity	No.	1	1
	Pre-Heat Coil			
	Capacity	MBH	1,300	190
	Fluid	Water	Water	
	Fluid Flow	GPM	200	19
	Fluid Pressure Drop	Ft of H2O	3.0	0.4
	Entering Fluid Temp.	°F	93.00	160.00
	Leaving Fluid Temp.	°F	80.00	140.00
	Fluid Rows	No.	4	2
	Number of Coils	No.	-	-
	Air Flow	CFM	67,200	3,300
	Air Pressure Drop	In of H2O	0.2	0.2
	Entering Air Temp. - DB	°F	5.00	10.00
	Leaving Air Temp. - DB	°F	70.00	60.00
	Approach	°F	-	-
Design Working Pressure	PSIG	-	-	

2.7.1.6 Tower A, Heating Hot Water: Unit Heaters and Cabinet Unit Heaters

Table 22. Tower A Heating Hot Water Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters							
Unit Heater ID		UH-5A to 12A	CUH-1A to 4A	CUH-5A & 6A	UH-14A to 16A	UH-18A to 25A	
Location		Penthouse Lvl	Street Lvl	Street Lvl	Garage	Gatehouse Penthouse	
Service		Penthouse MER	Secondary Entrance Htg	Exit Corridor Service Area	Garage	Gatehouse Penthouse	
Make		Trane	Trane	Trane	Trane	Trane	
Model Number		90S	04	02	90S	80P	
Serial Number		-	-	-	-	-	
Quantity of Units		8	4	2	3	8	
Unit Heater Capacity	MBH	40	22	14	40	30	
Design	Air Flow Data						
	Supply Air Flow	CFM	1,214	370	145	1,214	858
	Discharge Air Temp	°F	102.30	145.60	141.50	102.30	100.00
	Motor Data						
	Make		-	-	-	-	-
	Motor Control Type		Starter	Starter	Starter	Starter	Starter
	Motor Nameplate Power	HP	1/8	1/30	1/60	1/8	1/20
	Voltage	V	115	115	115	115	115
	Speed	RPM	1,550	700	700	1,550	1,550
	Nominal Efficiency	%	-	-	-	-	-
	Quantity	No.	1	1	1	1	1
	Heating Coil						
	Capacity	MBH	40	22	14	40	30
	Fluid		Water	Water	Water	Water	Water
	Fluid Flow	GPM	4	2	1	4	3
	Fluid Pressure Drop	Ft of H2O	0.8	2.2	0.5	0.8	0.2
	Entering Fluid Temp.	°F	160.00	160.00	160.00	160.00	160.00
	Leaving Fluid Temp.	°F	140.00	140.00	140.00	140.00	140.00
	Number of Coils	No.	-	-	-	-	-
	Air Flow	CFM	1,214	370	145	1,214	858
Leaving Air Temp. - DB	°F	102.30	145.60	141.50	102.30	100.00	
Approach	°F	-	-	-	-	-	
Design Working Pressure	PSIG	-	-	-	-	-	

2.7.2 Tower B

The following tables present detailed specifications for various systems and equipment in Tower B, including both steam and heating hot water-based components. These systems are integral to the heating and ventilation infrastructure of the tower. The tables include specifications for the steam-to-heating hot water heat exchangers, steam-to-domestic hot water heaters, air handling units, heating and ventilating units, unit heaters, and cabinet unit heaters, as well as the corresponding heating hot water systems. Each table outlines the technical details and configurations of these components, providing a comprehensive overview of the Heating, Ventilation, and Air Conditioning (HVAC) systems in Tower B.

2.7.2.1 Tower B, Steam: Steam-to-Heating Hot Water Heat Exchangers

Table 23. Tower B Steam-to-Heating Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger									
Heat Exchanger ID		HX-1B	HX-2B	HX-3B	HX-4B	HX-5B	HX-6B	HX-7B	
Location		Penthouse MER	Penthouse MER	Street Lvl MER	Street Lvl MER	Street Lvl MER	Street Lvl MER	Cellar 1 MER	
Service		HW AC-46B	HW Heater 26th-44th Flrs	HW Heater 3rd-25th Flrs	HW Heater 3rd-25th Flrs	HW FCU Street/Lobby	HW A-B Bridge	HW HV-1B	
Make		B&G	B&G	B&G	B&G	B&G	B&G	B&G	
Model Number		SU 184-2	SU 184-2	SU 204-2	SU 204-2	SU 144-2	SU 46-2	SU 147-2	
Serial Number		Q-59636	-	-	-	Q-59716	-	-	
Nominal Capacity		MBH	-	-	-	-	-	-	
Specified Net Capacity		MBH	5,500	5,500	6,800	6,800	3,800	80	
Shell Side - Steam									
Design Specifications	Fluid		Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow		LBS/Hr	5,670	5,670	7,010	7,010	3,920	8
	Entering Fluid Temp.		°F	302.10	302.10	302.10	302.10	302.10	302.10
	Entering Fluid Pressure		PSIG	2.0	2.0	2.0	2.0	2.0	2.0
	Entering Fluid Enthalpy		Btu/(Lbm-F)	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30
	Entering Fluid Deg Superheat		°F	83.60	83.60	83.60	83.60	83.60	83.60
	Leaving Fluid Temp.		°F	218.50	218.50	218.50	218.50	218.50	218.50
	Leaving Fluid Pressure		PSIG	2.0	2.0	2.0	2.0	2.0	2.0
	Leaving Fluid Enthalpy		Btu/(Lbm-F)	186.70	186.70	186.70	186.70	186.70	186.70
	Fluid Passes		No.	1	1	1	1	1	1
	Design Working Pressure		PSIG	150	150	150	150	150	150
	Tube Side - Water								
Fluid		Water	Water	Water	Water	Water	Water	Water	
Fluid Flow		GPM	550	550	680	680	380	8	
Fluid Pressure Drop		Ft of H2O	1.0	1.0	1.0	1.0	1.0	1.7	
Entering Fluid Temp.		°F	61.00	140.00	140.00	140.00	140.00	180.00	
Leaving Fluid Temp.		°F	81.00	160.00	160.00	160.00	160.00	200.00	
Fluid Passes		No.	2	2	2	2	2	2	
Approach		°F	-	-	-	-	-	-	
Design Working Pressure		PSIG	125	125	125	125	125	125	

2.7.2.2 Tower B, Heating Hot Water: Steam-to-Domestic Hot Water Heaters

Table 24. Tower B Steam-to-Domestic Hot Water Heater Specifications

Hot Water Heaters											
Hot Water Heater ID		H-1	H-2	H-3	H-4	H-5	H-6	H-7	H-8	H-9	
Location		Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	Cellar Level MER	
Service		Dining Terrace Bathrooms	Dining Terrace Bathrooms	Tenant Dining Terrace	Tenant Dining Terrace	Scullery Room	Scullery Room	Precipitators	Precipitators	Precipitators	
Make		AERCO	AERCO	AERCO	AERCO	AERCO	AERCO	AERCO	AERCO	AERCO	
Model Number		B+ 04EC	B+ 04EC	B+ 11EC	B+ 11EC	B+ 07EC	B+ 07EC	B+ 09EC	B+ 09EC	B+ 09EC	
Serial Number		H-12-391	H-12-390	H-12-393	H-12-392	H-12-395	H-12-394	H-12-388	H-12-389	H-12-387	
Nominal Capacity	MBH	-	-	-	-	-	-	-	-	-	
Specified Net Capacity	MBH	800	800	2,500	2,500	1,500	1,500	2,000	2,000	2,000	
Year Manufactured		2012	2012	2012	2012	2012	2012	2012	2012	2,012	
Shell Side - Steam											
Design Specifications	Fluid	Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow	LBS/Hr	802	802	2,506	2,506	1,504	1,504	2,005	2,005	2,005
	Entering Fluid Temp.	°F	303.50	303.50	303.50	303.50	303.50	303.50	303.50	303.50	303.50
	Entering Fluid Pressure	PSIG	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Entering Fluid Enthalpy	Btu/(Lbm-F)	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00
	Entering Fluid Deg Superheat	°F	76.40	76.40	76.40	76.40	76.40	76.40	76.40	76.40	76.40
	Leaving Fluid Temp.	°F	227.10	227.10	227.10	227.10	227.10	227.10	227.10	227.10	227.10
	Leaving Fluid Pressure	PSIG	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0	5.0
	Leaving Fluid Enthalpy	Btu/(Lbm-F)	195.40	195.40	195.40	195.40	195.40	195.40	195.40	195.40	195.40
	Fluid Passes	No.	-	-	-	-	-	-	-	-	-
	Design Working Pressure	PSIG	-	-	-	-	-	-	-	-	-
Tube Side - Water											
Fluid	Water	Water	Water	Water	Water	Water	Water	Water	Water	Water	
Fluid Flow	GPM	20	20	50	50	30	30	40	40	40	
Fluid Pressure Drop	Ft of H2O	-	-	-	1.0	1.0	1.0	1.0	1.0	1.7	
Entering Fluid Temp.	°F	40.00	40.00	40.00	40.00	40.00	40.00	40.00	40.00	40.00	
Leaving Fluid Temp.	°F	120.00	120.00	140.00	140.00	140.00	140.00	140.00	140.00	140.00	
Fluid Passes	No.	1	1	1	1	1	1	1	1	1	
Min. Flow	GPM	-	-	-	-	-	-	-	-	-	
Max. Flow	GPM	-	-	-	-	-	-	-	-	-	

2.7.2.3 Tower B, Steam: Air Handling Units

Table 25. Tower B Steam Air Handling Unit Specifications

Air Handling Unit							
AHU ID		AC-1B	AC-2B	AC-45B	AC-47B	AC-48B	
Location		Cellar MER Area C	Street Lvl MER	Street Lvl MER	2nd Sub-Cellar MER	1st Sub-Cellar MER	
Service		Lobby & Street Lvl	Liberty Street Bridge	OA 3rd to 15th Flrs	Cellar Lvl Tenant Storage	Cellar Lvl Core Areas	
Make		Trane	Trane	Nortek	Trane	Trane	
Supply Air Flow	CFM	65,000	7,200	130,000	84,000	30,400	
OA Airflow	CFM	-	-	130,000	-	-	
External Static	In of H2O	-	-	6.5	-	-	
Design	Supply Fan						
	Type	-	-	Centrifugal	-	-	
	Model	-	-	FW-45-1	-	-	
	Design Fan Speed	RPM	-	-	2,381	-	
	Maximum Fan Speed	RPM	865	1,287	-	621	
	Air Flow	CFM	65,000	7,200	8,667	42,000	
	Total Fan Static	In of H2O	6.0	4.5	-	4.8	
	Fan Brake Horsepower	BHP	86	8	-	39	
	Fan Efficiency	%	71%	61%	-	81%	
	Quantity	No.	1	1	15	2	
	Supply Fan Driver - Electric Motor						
	Make		-	-	Toshiba	-	-
	Motor Control Type		VFD	VFD	VFD	Starter	Starter
	Motor Nameplate Power	HP	100	10	15	50	30
	Voltage	V	460	460	460	460	460
	Speed	RPM	1,750	1,750	1,770	1,750	1,750
	Nominal Efficiency	%	-	-	92.4%	-	-
	Quantity	No.	1	1	15	2	1
	Pre-Heat Coil						
	Capacity	MBH	3,481	236	-	2,508	1,712
	Fluid		Steam	Steam	Steam	Steam	Steam
	Steam Flow	Lbs/Hr	3,664	248	-	2,640	1,802
	Fluid Rows	No.	1	1	-	1	1
	Number of Coils	No.	-	-	-	-	-
	Air Flow	CFM	65,000	7,200	-	84,000	30,400
	Air Pressure Drop	In of H2O	0.2	0.2	-	0.1	0.1
	Entering Air Temp. - DB	°F	5.00	57.00	-	57.00	57.00
	Leaving Air Temp. - DB	°F	55.00	88.00	-	80.00	80.00
Approach	°F	-	-	-	-	-	
Design Working Pressure	PSIG	-	-	-	-	-	

2.7.2.4 Tower B, Steam: Heating and Ventilating Units

Table 26. Tower B Steam Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit				
H&V ID		HV-3B	HV-4B	
Location		Cellar MER	Cellar Storage Adj. Transformer Vaults	
Service		Fresh Air/Smoke Retail Area	Storage/Mech. Space Ventilation	
Make		Trane	Trane	
Model Number		CAHB44C21CA2ESSC10F010	-	
Serial Number		K86G05323	-	
Supply Air Flow		CFM	121,200	
			3,420	
Design	Supply Fan			
	Type		Centrifugal	-
	Model		Master OL3109-0032-44	-
	Design Fan Speed		RPM	-
	Maximum Fan Speed		RPM	986
	Air Flow		CFM	60,600
	Total Fan Static		In of H2O	5.5
	Fan Brake Horsepower		BHP	-
	Fan Efficiency		%	-
	Quantity		No.	2
				1
	Supply Fan Driver - Electric Motor			
	Make		Marathon	-
	Motor Control Type		VFD	Starter
	Motor Nameplate Power		HP	50
	Voltage		V	460
	Speed		RPM	1,775
	Nominal Efficiency		%	93.0%
	Quantity		No.	2
				1
	Pre-Heat Coil			
	Capacity		MBH	6,476
	Fluid			Steam
	Steam Flow		Lbs/Hr	6,817
	Steam Inlet Pressure		PSIG	-
	Steam Temperature		°F	-
	Fluid Rows		No.	1
	Number of Coils		No.	-
Air Flow		CFM	121,200	
Air Pressure Drop		In of H2O	0.2	
Entering Air Temp. - DB		°F	5.00	
Leaving Air Temp. - DB		°F	55.00	
Approach		°F	-	
Design Working Pressure		PSIG	-	

2.7.2.5 Tower B, Steam: Unit Heaters and Cabinet Unit Heaters

Table 27. Tower B Steam Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters					
Unit Heater ID		GEH-1B to 4B	UH-13B to 20B	UH-27B & 28B	UH-29B
Location		Car Ramps	Gatehouse Penthouse	Link MER	Gatehouse Penthouse
Service		Car Ramp Entr. Heating	Gatehouse Penthouse	Link MER Heating	Emerg. Gen. Rm. Htg. Gatehouse
Make		DynaForce	Trane	Trane	Trane
Model Number		96-L/1	42P	42S	42S
Serial Number		-	-	-	-
Quantity of Units		4	8	2	1
Unit Heater Capacity		MBH	447	41	51
Air Flow Data					
Supply Air Flow		CFM	7,580	660	660
Discharge Air Temp		°F	-	124.00	124.00
Motor Data					
Make		-	-	-	-
Motor Control Type		Starter	Starter	Starter	Starter
Motor Nameplate Power		HP	1 1/ 2	1/25	1/25
Voltage		V	460	115	115
Speed		RPM	900	1550	1550
Nominal Efficiency		%	-	-	-
Quantity		No.	2	1	1
Heating Coil					
Capacity		MBH	447	41	41
Fluid			Steam	Steam	Steam
Steam Flow		Lbs/Hr	470	43	43
Steam Inlet Pressure		PSIG	-	-	-
Steam Temperature		°F	-	-	-
Fluid Rows		No.	-	-	-
Number of Coils		No.	-	-	-
Air Flow		CFM	7,580	660	660
Air Pressure Drop		In of H2O	-	-	-
Entering Air Temp. - DB		°F	-	-	-
Leaving Air Temp. - DB		°F	-	124.00	124.00
Approach		°F	-	-	-
Design Working Pressure		PSIG	-	-	-

Design

2.7.2.6 Tower B, Heating Hot Water: Air Handling Unit

Table 28. Tower B Heating Hot Water Air Handling Unit Specifications

Air Handling Unit			
AHU ID		AC-46B	
Location		Penthouse MER	
Service		OA 16th to 44th Flrs	
Make		Trane	
Model Number		CAFB49C12CC	
Serial Number		K86J13488/9	
Supply Air Flow	CFM	135,000	
OA Airflow	CFM	135,000	
External Static	In of H2O	6.50	
Design	Supply Fan		
	Type	Centrifugal	
	Model	490 BAE-DW	
	Design Fan Speed	RPM	961
	Maximum Fan Speed	RPM	-
	Air Flow	CFM	67,500
	Fan Efficiency	%	-
	Quantity	No.	2
	Supply Fan Driver - Electric Motor		
	Make	Marathon	
	Motor Control Type	VFD	
	Motor Nameplate Power	HP	100
	Voltage	V	460
	Speed	RPM	1,788
	Nominal Efficiency	%	95.4%
	Quantity	No.	2
	Reclaim Coil		
	Capacity	MBH	-
	Fluid		Water
	Fluid Flow	GPM	2,313
	Fluid Pressure Drop	Ft of H2O	-
	Entering Fluid Temp.	°F	-
	Leaving Fluid Temp.	°F	-
	Fluid Rows	No.	-
	Number of Coils	No.	3
	Air Flow	CFM	135,000
	Air Pressure Drop	In of H2O	-
Entering Air Temp. - DB	°F	-	
Leaving Air Temp. - DB	°F	-	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.2.7 Tower B, Heating Hot Water: Heating and Ventilating Unit

Table 29. Tower B Heating Hot Water Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit			
H&V ID		HV-1B	HV-2B
Location		1st Sub-Cellar MER	Cellar MER
Service		Garage Ventilation	Cellar Core Ventilation
Make		Trane	Trane
Supply Air Flow	CFM	42,500	2,600
Design	Supply Fan		
	Type		-
	Maximum Fan Speed	RPM	1,176
	Air Flow	CFM	42,500
	Total Fan Static	In of H2O	4.8
	Fan Brake Horsepower	BHP	48
	Fan Efficiency	%	66%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make		Marathon
	Motor Control Type		Starter
	Motor Nameplate Power	HP	60
	Voltage	V	460
	Speed	RPM	1,782
	Nominal Efficiency	%	95.0%
	Quantity	No.	1
	Pre-Heat Coil		
	Capacity	MBH	5,400
	Fluid		Water
	Fluid Flow	GPM	270
	Fluid Pressure Drop	Ft of H2O	4.0
	Entering Fluid Temp.	°F	120.00
	Leaving Fluid Temp.	°F	80.00
	Fluid Rows	No.	-
	Number of Coils	No.	-
	Air Flow	CFM	42,500
	Air Pressure Drop	In of H2O	0.5
	Entering Air Temp. - DB	°F	5.00
	Leaving Air Temp. - DB	°F	65.00
	Approach	°F	-
Design Working Pressure	PSIG	-	

2.7.2.8 Tower B, Heating Hot Water: Unit Heaters and Cabinet Unit Heaters

Table 30. Tower B Heating Hot Water Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters						
Unit Heater ID		CUH-1B & 2B	CUH-3B to 5B	CUH-6B & 7B	UH-7B to 12B	
Location		Street Lvl	Street Lvl	Street Lvl	Penthouse	
Service		Exit Heating	Exit Heating	Exit Heating	Penthouse MER Heating	
Make		Trane	Trane	Trane	Trane	
Model Number		N-AO-04	N-AO-03	N-AO-04	90S	
Serial Number		-	-	-	-	
Quantity of Units		2	3	2	6	
Unit Heater Capacity	MBH	38	29	38	40	
Design	Air Flow Data					
	Supply Air Flow	CFM	400	300	400	1,214
	Discharge Air Temp	°F	136.90	136.80	136.90	102.50
	Motor Data					
	Make		-	-	-	-
	Motor Control Type		Starter	Starter	Starter	Starter
	Motor Nameplate Power	HP	1/30	1/30	1/30	1/8
	Voltage	V	115	115	115	115
	Speed	RPM	1100	1100	1100	1550
	Nominal Efficiency	%	-	-	-	-
	Quantity	No.	1	1	1	1
	Heating Coil					
	Capacity	MBH	38	29	38	40
	Fluid		Water	Water	Water	Water
	Fluid Flow	GPM	4	3	4	4
	Fluid Pressure Drop	Ft of H2O	3.9	1.8	3.9	0.8
	Entering Fluid Temp.	°F	160.00	160.00	160.00	160.00
	Leaving Fluid Temp.	°F	140.00	140.00	140.00	140.00
	Fluid Rows	No.	-	-	-	-
	Number of Coils	No.	-	-	-	-
	Air Flow	CFM	400	300	400	1,214
	Air Pressure Drop	In of H2O	-	-	-	-
	Entering Air Temp. - DB	°F	-	-	-	-
	Leaving Air Temp. - DB	°F	136.90	136.80	136.90	102.50
	Approach	°F	-	-	-	-
	Design Working Pressure	PSIG	-	-	-	-

2.7.3 Tower C

The following tables present detailed specifications for various systems and equipment in Tower C, including both steam and heating hot water-based components. These systems are integral to the heating and ventilation infrastructure of the tower. The tables include specifications for the steam-to-heating hot water heat exchangers, steam-to-domestic hot water heaters, air handling units, heating and ventilating units, unit heaters, and cabinet unit heaters, as well as the corresponding heating hot water systems. Each table outlines the technical details and configurations of these components, providing a comprehensive overview of the HVAC systems in Tower C.

2.7.3.1 Tower C, Steam: Steam-to-Heating Hot Water Heat Exchangers

Table 31. Tower C Steam-to-Heating Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger										
Heat Exchanger ID		HX-1C	HX-2C	HX-3C	HX-4C	HX-5C	HX-6C	HX-7C		
Location		Tower C	Tower C	Tower C	Tower C	Tower C	Tower C	Tower C		
Service		HHW 3rd - 27th Flrs	HHW Service, Street & 2nd Flrs	Condensate Heat Recovery	Aux. Heating AC-52C	HHW 28th - 51st Flrs	HHW HV-1C & 2C	Stdby. for HX-1C & 2C		
Make		ITT	ITT	Yula	ITT	ITT	ITT	ITT		
Model Number		SU 184-2	SU 184-2	HWC-4J-156B	SU 164-2	SU 184-2	SU 104-2	SU 184-2		
Serial Number		-	-	-	-	-	-	-		
Nominal Capacity		MBH	6,000	7,000	1,430	5,000	6,500	1,750	6,000	
Specified Net Capacity		MBH	6,000	7,000	1,430	5,000	6,500	1,750	6,000	
Shell Side - Steam										
Design Specifications	Fluid		Steam	Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow		LBS/Hr	6,000	7,000	1,430	5,000	6,500	1,750	6,000
	Entering Fluid Temp.		°F	302.10	302.10	302.10	302.10	302.10	302.10	302.10
	Entering Fluid Pressure		PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Entering Fluid Enthalpy		Btu/(Lbm-F)	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30
	Entering Fluid Deg Superheat		°F	83.60	83.60	83.60	83.60	83.60	83.60	83.60
	Leaving Fluid Temp.		°F	218.50	218.50	218.50	218.50	218.50	218.50	218.50
	Leaving Fluid Pressure		PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Leaving Fluid Enthalpy		Btu/(Lbm-F)	186.70	186.70	186.70	186.70	186.70	186.70	186.70
	Tube Side - Water									
	Fluid		Water	Water	Water	Water	Water	Water	Water	
	Fluid Flow		GPM	600	700	220	500	650	175	600
	Fluid Pressure Drop		Ft of H2O	1.1	1.5	4.0	1.2	1.2	1.1	1.1
	Entering Fluid Temp.		°F	140.00	140.00	80.00	140.00	140.00	140.00	140.00
	Leaving Fluid Temp.		°F	160.00	160.00	93.00	160.00	160.00	160.00	160.00
	Fluid Passes		No.	2	2	4	2	2	2	2
	Approach		°F	-	-	-	-	-	-	-
	Design Working Pressure		PSIG	175	125	250	125	125	125	175
	Min. Flow		GPM	-	-	-	-	-	-	-
Max. Flow		GPM	-	-	-	-	-	-	-	

2.7.3.2 Tower C, Steam-to-Domestic Hot Water Heaters

Table 32. Tower C Steam-to-Domestic Hot Water Heater Specifications

Hot Water Heaters									
Hot Water Heater ID		H-1	H-2	H-3	H-4	H-5	H-6	H-7	
Location		-	-	-	-	-	P-62 Room	P-62 Room	
Service		-	-	-	-	-	Kitchen	Kitchen	
Make		Patterson-Kelley	Patterson-Kelley	Patterson-Kelley	Patterson-Kelley	Patterson-Kelley	Patterson-Kelley	Patterson-Kelley	
Model Number		-	-	-	-	-	-	-	
Serial Number		270622	270623	270624	270625	270626	270914	270915	
Nominal Capacity	MBH	1,000	1,000	1,000	1,000	1,000	2,000	2,000	
Specified Net Capacity	MBH	1,000	1,000	1,000	1,000	1,000	2,000	2,000	
Year Manufactured		1984	1984	1984	1984	1984	1984	1984	
Design Specifications	Shell Side - Steam								
	Fluid		Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow	LBS/Hr	1,029	1,029	1,029	1,029	1,029	2,077	2,077
	Entering Fluid Temp.	°F	303.50	303.50	303.50	303.50	303.50	305.80	305.80
	Entering Fluid Pressure	PSIG	5.0	5.0	5.0	5.0	5.0	10.0	10.0
	Entering Fluid Enthalpy	Btu/(Lbm-F)	1,193.00	1,193.00	1,193.00	1,193.00	1,193.00	1,193.30	1,193.30
	Entering Fluid Deg Superheat	°F	76.40	76.40	76.40	76.40	76.40	66.40	66.40
	Leaving Fluid Temp.	°F	227.10	227.10	227.10	227.10	227.10	239.40	239.40
	Leaving Fluid Pressure	PSIG	5.0	5.0	5.0	5.0	5.0	10.0	10.0
	Leaving Fluid Enthalpy	Btu/(Lbm-F)	195.40	195.40	195.40	195.40	195.40	207.80	207.80
	Fluid Passes	No.	-	-	-	-	-	-	-
	Design Working Pressure	PSIG	150	150	150	150	150	150	150
	Tube Side - Water								
	Fluid		Water	Water	Water	Water	Water	Water	Water
	Fluid Flow	GPM	25	25	25	25	25	40	40
	Fluid Pressure Drop	Ft of H2O	150.0	150.0	150.0	-	-	150.0	150.0
	Entering Fluid Temp.	°F	40.0	40.0	40.0	40.0	40.0	40.0	40.0
	Leaving Fluid Temp.	°F	120.00	120.00	120.00	120.00	120.00	140.00	140.00
	Fluid Passes	No.	-	-	-	-	-	-	-
Approach	°F	-	-	-	-	-	-	-	
Design Working Pressure	PSIG	150	150	150	150	150	150	150	
Min. Flow	GPM	-	-	-	-	-	-	-	
Max. Flow	GPM	-	-	-	-	-	-	-	

2.7.3.3 Tower C, Steam: Air Handling Units

Table 33. Tower C Steam Air Handling Unit Specifications

Air Handling Unit						
AHU ID		AC-1C	AC-2C	AHU-301C	AHU-303C	
Location		3rd Flr Fan Room	3rd Flr Fan Room	3rd Flr Fan Room	3rd Flr Fan Room	
Service		Lobby Levels	Retail Areas	Kitchen	Kitchen	
Make		Trane	Trane	-	-	
Supply Air Flow	CFM	63,000	117,500	-	-	
Design	Supply Fan					
	Type	Centrifugal	Centrifugal	-	-	
	Maximum Fan Speed	RPM	844	834	-	-
	Air Flow	CFM	63,000	58,750	-	-
	Total Fan Static	In of H2O	6.0	6.0	-	-
	Fan Brake Horsepower	BHP	77	72	-	-
	Fan Efficiency	%	78%	77%	-	-
	Quantity	No.	1	2	1	1
	Supply Fan Driver - Electric Motor					
	Make		-	-	-	-
	Motor Control Type		VFD	VFD	Starter	Starter
	Motor Nameplate Power	HP	100	100	-	-
	Voltage	V	460	460	-	-
	Speed	RPM	1,750	1,750	-	-
	Nominal Efficiency	%	-	-	-	-
	Quantity	No.	1	2	1	1
	Pre-Heat Coil					
	Capacity	MBH	1,615	1,520	-	-
	Fluid		Steam	Steam	Steam	Steam
	Steam Flow	Lbs/Hr	1,700	1,600	-	-
	Steam Inlet Pressure	PSIG	-	-	-	-
	Steam Temperature	°F	-	-	-	-
	Fluid Rows	No.	1	1	-	-
	Number of Coils	No.	-	-	-	-
	Air Flow	CFM	63,000	117,500	-	-
	Air Pressure Drop	In of H2O	0.2	0.2	-	-
	Entering Air Temp. - DB	°F	5.00	5.00	-	-
	Leaving Air Temp. - DB	°F	55.00	55.00	-	-
Approach	°F	-	-	-	-	
Design Working Pressure	PSIG	-	-	-	-	

2.7.3.4 Tower C, Heating Hot Water: Heating and Ventilating Units

Table 34. Tower C Heating Hot Water Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit				
H&V ID		HV-1C	HV-2C	
Location		Service Level	Service Level	
Service		Garage Vent	MER Ventilation	
Make		Trane	Trane	
Supply Air Flow	CFM	25,000	5,410	
Design	Supply Fan			
	Type		-	
	Maximum Fan Speed	RPM	1,083	1,060
	Air Flow	CFM	25,000	5,410
	Total Fan Static	In of H2O	2.0	2.5
	Fan Brake Horsepower	BHP	15	4
	Fan Efficiency	%	53%	59%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make		-	-
	Motor Control Type		Starter	Starter
	Motor Nameplate Power	HP	20	5
	Voltage	V	460	460
	Quantity	No.	1	1
	Pre-Heat Coil			
	Capacity	MBH	1,660	360
	Fluid		Water	Water
	Fluid Flow	GPM	166	36
	Fluid Pressure Drop	Ft of H2O	10.1	1.0
	Entering Fluid Temp.	°F	160.00	160.00
	Leaving Fluid Temp.	°F	140.00	140.00
	Fluid Rows	No.	1	1
	Number of Coils	No.	-	-
	Air Flow	CFM	25,000	5,410
	Air Pressure Drop	In of H2O	0.3	0.3
	Entering Air Temp. - DB	°F	5.00	5.00
	Leaving Air Temp. - DB	°F	65.00	65.00
	Approach	°F	-	-
Design Working Pressure	PSIG	-	-	

2.7.3.5 Tower C, Heating Hot Water: Unit Heaters and Cabinet Unit Heaters

Table 35. Tower C Heating Hot Water Unit Heaters and Cabinet Unit Heaters Specifications

Unit Heaters			
Unit Heater ID		UH-1C to 7C	
Location		Penthouse MER	
Service		Penthouse MER	
Make		Trane	
Model Number		90S	
Serial Number		-	
Quantity of Units		7	
Unit Heater Capacity		MBH	40
Design	Air Flow Data		
	Supply Air Flow	CFM	1,214
	Discharge Air Temp	°F	102.30
	Motor Data		
	Make	-	
	Motor Control Type	Starter	
	Motor Nameplate Power	HP	1/8
	Voltage	V	115
	Speed	RPM	1550
	Nominal Efficiency	%	-
	Quantity	No.	1
	Heating Coil		
	Capacity	MBH	40
	Fluid	Water	
	Fluid Flow	GPM	4
	Fluid Pressure Drop	Ft of H2O	0.8
	Entering Fluid Temp.	°F	160.00
	Leaving Fluid Temp.	°F	140.00
	Number of Coils	No.	-
	Air Flow	CFM	1,214
	Air Pressure Drop	In of H2O	-
	Entering Air Temp. - DB	°F	-
Leaving Air Temp. - DB	°F	102.30	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.4 Tower D

The following tables present detailed specifications for various systems and equipment in Tower D, including both steam and heating hot water-based components. These systems are integral to the heating and ventilation infrastructure of the tower. The tables include specifications for the steam-to-heating hot water heat exchangers, steam-to-domestic hot water heaters, air handling units, heating and ventilating units, unit heaters, and cabinet unit heaters, as well as the corresponding heating hot water systems. Each table outlines the technical details and configurations of these components, providing a comprehensive overview of the HVAC systems in Tower D.

2.7.4.1 Tower D, Steam: Steam-to-Heating Hot Water Heat Exchangers

Table 36. Tower D Steam-to-Heating Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger									
Heat Exchanger ID		HX-1D	HX-2D	HX-3D	HX-4D	HX-5D	HX-6D	HX-7D	
Location		Penthouse MER	2nd Floor MER	2nd Floor MER	2nd Floor MER	Penthouse MER	4th Floor MER	4th Floor MER	
Service		HHW High Rise	HHW Mid Rise	HHW Trench System	HHW Trench System	HHW AC-35D	HHW Low Rise	HHW Low Rise	
Make		Yula	Yula	Yula	Yula	Yula	B&G	B&G	
Model Number		CV-2E-78B	CV-2C-108B	CV-2E-66B	CV-2E-66B	CV-2F-66B	SU-124-2	SU-124-2	
Serial Number		11934	11937	11936	11935	11938	-	-	
Nominal Capacity	MBH	4,350	2,000	3,500	3,500	8,010	3,800	3,800	
Specified Net Capacity	MBH	4,350	2,000	3,500	3,500	8,010	3,800	3,800	
Design Specifications	Shell Side - Steam								
	Fluid		Steam	Steam	Steam	Steam	Steam	Steam	
	Fluid Mass Flow	LBS/Hr	4,570	2,070	3,625	3,625	8,260	3,934	3,934
	Entering Fluid Temp.	°F	302.10	302.10	302.10	302.10	302.10	302.10	302.10
	Entering Fluid Pressure	PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Entering Fluid Enthalpy	Btu/(Lbm-F)	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30	1193.30
	Entering Fluid Deg Superheat	°F	83.60	83.60	83.60	83.60	83.60	83.60	83.60
	Leaving Fluid Temp.	°F	218.50	218.50	218.50	218.50	218.50	218.50	218.50
	Leaving Fluid Pressure	PSIG	2.0	2.0	2.0	2.0	2.0	2.0	2.0
	Leaving Fluid Enthalpy	Btu/(Lbm-F)	186.70	186.70	186.70	186.70	186.70	186.70	186.70
	Fluid Passes	No.	2	2	2	2	2	1	1
	Design Working Pressure	PSIG	150	150	150	150	150	150	150
	Tube Side								
	Fluid		Water	Water	Water	Water	Water	Water	Water
	Fluid Flow	GPM	435	200	350	350	450	380	380
	Fluid Pressure Drop	Ft of H2O	1.6	1.0	1.0	1.0	1.6	1.7	1.7
	Entering Fluid Temp.	°F	140.00	140.00	140.00	140.00	65.00	140.00	140.00
	Leaving Fluid Temp.	°F	160.00	160.00	160.00	160.00	100.60	160.00	160.00
	Fluid Passes	No.	2	2	2	2	2	2	2
Approach	°F	-	-	-	-	-	-	-	
Design Working Pressure	PSIG	175	175	175	175	175	-	-	
Min. Flow	GPM	-	-	-	-	-	-	-	
Max. Flow	GPM	-	-	-	-	-	-	-	

2.7.4.2 Tower D, Steam: Air Handling Units

Table 37. Tower D Steam Air Handling Unit Specifications

Air Handling Unit				
AHU ID		AC-1D	AC-2DN	AC-2DS
Location		+0'-6" Lvl MER	3rd Flr MER North	3rd Flr MER South
Service		Lobby	Courtyard North & East Area	Courtyard South & West Area
Make		Trane	Trane	Trane
Supply Air Flow	CFM	33,300	51,600	65,600
Design Specifications	Supply Fan			
	Type	Centrifugal	-	-
	Model	40-AFSW-21-10-11	44 AFD	49 AFD
	Design Fan Speed	RPM	1,136	-
	Maximum Fan Speed	RPM	1,283	819
	Air Flow	CFM	33,300	51,600
	Fan Static	In of H2O	3.9	3.3
	Fan Brake Horsepower	BHP	-	46
	Fan Efficiency	%	-	58%
	Quantity	No.	1	1
	Supply Fan Driver - Electric Motor			
	Make		-	Marathon
	Motor Control Type		VFD	VFD
	Motor Nameplate Power	HP	40	60
	Voltage	V	460	460
	Speed	RPM	1,725	1,785
	Nominal Efficiency	%	94.1%	95.0%
	Quantity	No.	1	1
	Pre-Heat Coil			
	Capacity	MBH	-	2,762
	Fluid		Steam	Steam
	Steam Flow	Lbs/Hr	-	2,907
	Fluid Rows	No.	-	1
	Number of Coils	No.	-	-
	Air Flow	CFM	-	51,600
	Air Pressure Drop	In of H2O	-	0.2
	Entering Air Temp. - DB	°F	-	5.0
Leaving Air Temp. - DB	°F	-	55.0	
Approach	°F	-	-	
Design Working Pressure	PSIG	-	-	

2.7.4.3 Tower D, Steam: Heating and Ventilating Units

Table 38. Tower D Steam Heating and Ventilating Unit Information Specifications

Heating and Ventilation Air Handling Unit			
H&V ID		SF-1D	SF-2D
Location		3rd Flr MER North	3rd Flr MER South
Service		Retail H&V	Retail H&V
Make		Trane	Trane
Supply Air Flow		CFM	50,000
			75,000
Design	Supply Fan		
	Type	Centrifugal	Centrifugal
	Model	36 AF	44 AF
	Design Fan Speed	RPM	-
	Maximum Fan Speed	RPM	1,321
	Air Flow	CFM	50,000
	Total Fan Static	In of H2O	4.5
	Fan Brake Horsepower	BHP	70
	Fan Efficiency	%	51%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make	-	Marathon
	Motor Control Type	VFD	VFD
	Motor Nameplate Power	HP	75
	Voltage	V	460
	Speed	RPM	-
	Nominal Efficiency	%	-
	Quantity	No.	1
	Pre-Heat Coil		
	Capacity	MBH	1,634
	Fluid	Steam	Steam
	Steam Flow	Lbs/Hr	1,720
	Air Flow	CFM	30,350
	Air Pressure Drop	In of H2O	0.1
	Entering Air Temp. - DB	°F	5.0
	Leaving Air Temp. - DB	°F	55.00
	Approach	°F	-
	Design Working Pressure	PSIG	-

2.7.4.4 Tower D, Steam: Unit Heaters and Cabinet Unit Heaters

Table 39. Tower D Steam Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters			
Unit Heater ID		UH-44D	
Location		+0'-6" MER	
Service		+0'-6" MER Smoke Exh Make Up	
Make		Trane	
Model Number		UHS-090-S	
Serial Number		-	
Quantity of Units		1	
Unit Heater Capacity		MBH 93	
Design	Air Flow Data		
	Supply Air Flow	CFM 1,214	
	Discharge Air Temp	°F 127.00	
	Motor Data		
	Make		-
	Motor Control Type		Starter
	Motor Nameplate Power	HP 1/8	
	Voltage	V 115	
	Speed	RPM 1550	
	Nominal Efficiency	% -	
	Quantity	No. 1	
	Heating Coil		
	Capacity	MBH 93	
	Fluid		Steam
	Steam Flow	Lbs/Hr 98	
	Steam Inlet Pressure	PSIG 5.0	
	Steam Temperture	°F -	
	Fluid Rows	No. -	
	Number of Coils	No. -	
	Air Flow	CFM 1,214	
Air Pressure Drop	In of H2O -		
Entering Air Temp. - DB	°F -		
Leaving Air Temp. - DB	°F 127.00		
Approach	°F -		
Design Working Pressure	PSIG -		

2.7.4.5 Tower D, Heating Hot Water: Air Handling Unit

Table 40. Tower D Heating Hot Water Air Handling Unit Specifications

Air Handling Unit			
AHU ID		AC-35D	
Location		Penthouse MER	
Service		OA to CUs	
Make		Trane	
Model Number		-	
Serial Number		-	
Supply Air Flow	CFM	-	
OA Airflow	CFM	-	
Total External Static	In of H2O	-	
Design Specifications	Supply Fan		
	Type	Centrifugal	
	Model	542 VCR AF	
	Design Fan Speed	RPM	-
	Maximum Fan Speed	RPM	-
	Air Flow	CFM	-
	Fan Static	In of H2O	-
	Fan Brake Horsepower	BHP	-
	Fan Efficiency	%	-
	Quantity	No.	2
	Supply Fan Driver - Electric Motor		
	Make	Teco	
	Motor Control Type	VFD	
	Motor Nameplate Power	HP	200
	Voltage	V	460
	Speed	RPM	1,780
	Nominal Efficiency	%	95.0%
	Quantity	No.	2
	Reclaim Coil		
	Capacity	MBH	-
	Fluid	Water	
	Fluid Flow	GPM	-
	Fluid Pressure Drop	Ft of H2O	-
	Entering Fluid Temp.	°F	-
	Leaving Fluid Temp.	°F	-
	Fluid Rows	No.	-
	Number of Coils	No.	-
	Air Flow	CFM	-
Air Pressure Drop	In of H2O	-	
Entering Air Temp. - DB	°F	-	
Leaving Air Temp. - DB	°F	-	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.4.6 Tower D, Heating Hot Water: Unit Heaters and Cabinet Unit Heaters

Table 41. Tower D Heating Hot Water Unit Heater and Cabinet Unit Heater Specifications

Shell and Tube Heat Exchanger			
Heat Exchanger ID		HX-1W	HX-W2
Location		Winter Garden	Winter Garden
Service		Secondary HW System	Secondary HW System
Make		B&G	B&G
Model Number		SU-125-2	SU-125-2
Serial Number		-	-
Nominal Capacity	MBH	1,080	1,080
Specified Net Capacity	MBH	1,078	1,078
Shell Side - Steam			
Fluid		Steam	Steam
Fluid Mass Flow	LBs/Hr	1,115	1,115
Entering Fluid Temp.	°F	302.10	302.10
Entering Fluid Pressure	PSIG	2.00	2.00
Entering Fluid Enthalpy	Btu/(Lbm-F)	1193.30	1193.30
Entering Fluid Deg Superheat	F	83.60	83.60
Leaving Fluid Temp.	°F	218.50	218.50
Leaving Fluid Pressure	PSIG	2.0	2.0
Leaving Fluid Enthalpy	Btu/(Lbm-F)	186.70	186.70
Fluid Passes	No.	-	-
Design Working Pressure	PSIG	-	-
Tube Side			
Fluid		Water	Water
Fluid Flow	GPM	98	98
Fluid Pressure Drop	Ft of H2O	1.0	1.0
Entering Fluid Temp.	°F	178.00	178.00
Leaving Fluid Temp.	°F	200.00	200.00
Fluid Passes	No.	2	2
Approach	°F	-	-
Design Working Pressure	PSIG	125	125
Min. Flow	GPM	-	-
Max. Flow	GPM	-	-

2.7.5 Winter Garden

The following tables present detailed specifications for various systems and equipment in Winter Garden, including both steam and heating hot water-based components. These systems are integral to the heating and ventilation infrastructure of the tower. The tables include specifications for the steam-to-heating hot water heat exchangers, steam-to-domestic hot water heaters, air handling units, heating and ventilating units, unit heaters, and cabinet unit heaters, as well as the corresponding heating hot water systems. Each table outlines the technical details and configurations of these components, providing a comprehensive overview of the HVAC systems in Winter Garden.

2.7.5.1 Winter Garden, Steam: Steam-to-Heating Hot Water Heat Exchangers

Table 42. Winter Garden Steam-to-Heating Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger			
Heat Exchanger ID		HX-1W	HX-W2
Location		Winter Garden	Winter Garden
Service		Secondary HW System	Secondary HW System
Make		B&G	B&G
Model Number		SU-125-2	SU-125-2
Serial Number		-	-
Nominal Capacity	MBH	1,080	1,080
Specified Net Capacity	MBH	1,078	1,078
Shell Side - Steam			
Fluid		Steam	Steam
Fluid Mass Flow	LBS/Hr	1,115	1,115
Entering Fluid Temp.	°F	302.10	302.10
Entering Fluid Pressure	PSIG	2.00	2.00
Entering Fluid Enthalpy	Btu/(Lbm-F)	1193.30	1193.30
Entering Fluid Deg Superheat	F	83.60	83.60
Leaving Fluid Temp.	°F	218.50	218.50
Leaving Fluid Pressure	PSIG	2.0	2.0
Leaving Fluid Enthalpy	Btu/(Lbm-F)	186.70	186.70
Fluid Passes	No.	-	-
Design Working Pressure	PSIG	-	-
Tube Side			
Fluid		Water	Water
Fluid Flow	GPM	98	98
Fluid Pressure Drop	Ft of H2O	1.0	1.0
Entering Fluid Temp.	°F	178.00	178.00
Leaving Fluid Temp.	°F	200.00	200.00
Fluid Passes	No.	2	2
Approach	°F	-	-
Design Working Pressure	PSIG	125	125
Min. Flow	GPM	-	-
Max. Flow	GPM	-	-

2.7.5.2 Winter Garden, Steam: Air Handling Units

Table 43. Winter Garden Steam Air Handling Unit Specifications

Air Handling Unit									
AHU ID		AC-1W	AC-2W	AC-5W	AC-7W	AC-8W	AC-9W	AC-10W	AC-11W
Location		Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Penthouse MER	Service Lvl MER	Penthouse MER
Service		Winter Garden	Winter Garden	Winter Garden	Winter Garden	Winter Garden	Retail Area	Entrance Lobby East Side	Retail Area
Make		Trane	Trane	Trane	Trane	Trane	Trane	Trane	Trane
Model Number		-	-	-	-	-	-	-	-
Serial Number		-	-	-	-	-	-	-	-
Supply Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Supply Fan									
Type		-	-	-	-	-	-	-	-
Maximum Fan Speed	RPM	912	921	1,078	921	912	1,129	1,114	1,375
Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Total Fan Static	In of H2O	4.0	4.1	4.1	4.1	4.0	5.1	3.5	4.8
Fan Brake Horsepower	BHP	23	24	24	24	23	26	6	9
Fan Efficiency	%	81%	81%	80%	81%	81%	77%	63%	57%
Quantity	No.	1	1	1	1	1	1	1	1
Supply Fan Driver - Electric Motor									
Make		-	-	-	-	-	-	-	-
Motor Control Type		VFD	VFD	VFD	VFD	VFD	VFD	VFD	VFD
Motor Nameplate Power	HP	30	30	30	30	30	30	8	10
Voltage	V	460	460	460	460	460	460	460	460
Speed	RPM	1,750	1,750	1,750	1,750	1,750	1,750	1,750	1,750
Nominal Efficiency	%	-	-	-	-	-	-	-	-
Quantity	No.	1	1	1	1	1	1	1	1
Pre-Heat Coil									
Capacity	MBH	2,240	2,240	2,431	2,240	2,240	1,872	364	524
Fluid		Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam
Steam Flow	Lbs/Hr	2,358	2,358	2,559	2,358	2,358	1,971	384	552
Fluid Rows	No.	1	1	1	1	1	1	1	1
Number of Coils	No.	-	-	-	-	-	-	-	-
Air Flow	CFM	29,900	29,900	29,900	29,900	29,900	25,000	6,800	7,000
Air Pressure Drop	In of H2O	0.1	0.1	0.1	0.1	0.1	0.1	0.1	0.2
Entering Air Temp. - DB	°F	5.00	5.00	5.00	5.00	5.00	5.00	70.00	5.00
Leaving Air Temp. - DB	°F	75.00	75.00	81.00	75.00	75.00	75.00	120.00	75.00
Approach	°F	-	-	-	-	-	-	-	-
Design Working Pressure	PSIG	-	-	-	-	-	-	-	-

Design

2.7.5.3 Winter Garden, Steam: Heating and Ventilating Units

Table 44. Winter Garden Steam Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit			
H&V ID		HV-1W	HV-2W
Location		PRV Room	PRV Room
Service		Pipe Tunnel	PRV Room & Storage
Make		Trane	Trane
Model Number		-	-
Serial Number		-	-
Supply Air Flow		CFM	2,800
		2,800	2,800
Design	Supply Fan		
	Type		-
	Model		-
	Design Fan Speed	RPM	-
	Maximum Fan Speed	RPM	1,136
	Air Flow	CFM	2,800
	Total Fan Static	In of H2O	2.0
	Fan Brake Horsepower	BHP	1
	Fan Efficiency	%	63%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make		-
	Motor Control Type		Starter
	Motor Nameplate Power	HP	2
	Voltage	V	460
	Speed	RPM	1,750
	Nominal Efficiency	%	-
	Quantity	No.	1
	Pre-Heat Coil		
	Capacity	MBH	178
	Fluid		Steam
	Steam Flow	Lbs/Hr	187
	Air Flow	CFM	2,800
	Air Pressure Drop	In of H2O	-
	Entering Air Temp. - DB	°F	5.00
	Leaving Air Temp. - DB	°F	65.00
	Approach	°F	-
Design Working Pressure	PSIG	-	

2.7.5.4 Winter Garden, Heating Hot Water: Air Handling Units

Table 45. Winter Garden Heating Hot Water Air Handling Unit Specifications

Air Handling Unit			
AHU ID		AC-12W & 13W	
Location		Entrance Vestibule Ceilings	
Service		Entrance Vestibule West Side	
Make		Trane	
Model Number		-	
Serial Number		-	
Supply Air Flow	CFM	2,500	
Design	Supply Fan		
	Type	-	
	Maximum Fan Speed	RPM	1,031
	Air Flow	CFM	2,500
	Total Fan Static	In of H2O	1.8
	Fan Brake Horsepower	BHP	1
	Fan Efficiency	%	69%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make	-	
	Motor Control Type	Starter	
	Motor Nameplate Power	HP	2
	Voltage	V	460
	Speed	RPM	1,750
	Nominal Efficiency	%	-
	Quantity	No.	1
	Dual-Temp Coil (1 Coil on this AHU) - Heating Design Conditions		
	Capacity	MBH	136
	Fluid	Water	
	Fluid Flow	GPM	-
Air Flow	CFM	2,500	
Air Pressure Drop	In of H2O	-	
Entering Air Temp. - DB	°F	70.00	
Leaving Air Temp. - DB	°F	120.00	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.6 Central Plant

The following tables present detailed specifications for various systems and equipment in the Central Plant, including both steam and heating hot water-based components. These systems are integral to the heating and ventilation infrastructure of the tower. The tables include specifications for the steam-to-heating hot water heat exchangers, steam-to-domestic hot water heaters, air handling units, heating and ventilating units, unit heaters, and cabinet unit heaters, as well as the corresponding heating hot water systems. Each table outlines the technical details and configurations of these components, providing a comprehensive overview of the HVAC systems in the Central Plant.

2.7.6.1 Central Plant, Steam: Steam-to-Heating Hot Water Heat Exchanger

Table 46. Central Plant Steam-to-Heating Hot Water Heat Exchanger Specifications

Shell and Tube Heat Exchanger		
Heat Exchanger ID		HX-1P
Location		MER No.1 +19 Level
Service		Glycol Heating
Make		Yula
Model Number		CB-2C-90B
Serial Number		-
Nominal Capacity	MBH	3,250
Specified Net Capacity	MBH	3,250
Shell Side - Steam		
Fluid		Steam
Fluid Mass Flow	LBS/Hr	3,000
Entering Fluid Temp.	°F	305.80
Entering Fluid Pressure	PSIG	10.00
Entering Fluid Enthalpy	Btu/(Lbm-F)	1193.30
Entering Fluid Deg Superheat	F	66.40
Leaving Fluid Temp.	°F	239.40
Leaving Fluid Pressure	PSIG	10.0
Leaving Fluid Enthalpy	Btu/(Lbm-F)	207.80
Fluid Passes	No.	1
Design Working Pressure	PSIG	-
Tube Side		
Fluid		Water
Fluid Flow	GPM	130
Fluid Pressure Drop	Ft of H2O	5.0
Entering Fluid Temp.	°F	90.00
Leaving Fluid Temp.	°F	140.00
Design Working Pressure	PSIG	125
Min. Flow	GPM	-
Max. Flow	GPM	-

2.7.6.2 Central Plant, Steam: Air Handling Unit

Table 47. Central Plant Steam Air Handling Unit Specifications

Air Handling Unit			
AHU ID	AC-5P		
Location	1st Sub-Cellar		
Service	1st Sub-Cellar Offices		
Make	Trane		
Supply Air Flow	CFM	1,915	
Design	Supply Fan		
	Design Fan Speed	RPM	-
	Maximum Fan Speed	RPM	1,372
	Air Flow	CFM	1,915
	Total Fan Static	In of H2O	2.3
	Fan Brake Horsepower	BHP	1
	Fan Efficiency	%	48%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Make	-	
	Motor Control Type	-	
	Motor Nameplate Power	HP	2
	Voltage	V	460
	Speed	RPM	1,750
	Nominal Efficiency	%	-
	Quantity	No.	1
	Pre-Heat Coil		
	Capacity	MBH	61
	Fluid	Steam	
	Steam Flow	Lbs/Hr	64
	Steam Inlet Pressure	PSIG	-
	Steam Temperture	°F	-
	Air Flow	CFM	1,915
	Air Pressure Drop	In of H2O	0.2
	Entering Air Temp. - DB	°F	50.00
	Leaving Air Temp. - DB	°F	80.00
	Approach	°F	-
Design Working Pressure	PSIG	-	

2.7.6.3 Central Plant, Steam: Heating and Ventilating Unit

Table 48. Central Plant Steam Heating and Ventilating Unit Specifications

Heating and Ventilation Air Handling Unit											
H&V ID		HV-1P	HV-2P	HV-3P	HV-4P	HV-5P	HV-6P	HV-7P	HV-8P	HV-10P	HV-11P
Location		-7'-9" MER #3	-7'-9" MER #3	-7'-9" MER #3	+19 MER #2	+19 MER #2	+19 MER #1	-7'-9" MER #3	-7'-9" MER #3	+19 MER #2	Cellar Level
Service		Garage Ventilation 2nd, 3rd, & 4th Sub-Cellar	Garage Ventilation 2nd, 3rd, & 4th Sub-Cellar	Garage Ventilation Cellar & 1st Sub-Cellar	Central Plant Ventilation	5th Sub-Cellar Electric Rooms	Core Supply	Steam MER	Truck Dock Supply	5th Sub-Cellar Electric Room	Gas & Water Meter Room
Make		Joy	Joy	Joy	Joy	Joy	Joy	Joy	Joy	Joy	Trane
Supply Air Flow	CFM	99,700	99,700	46,800	48,000	59,000	68,355	38,000	43,400	40,500	2,500
Supply Fan											
Type		-	-	-	-	-	-	-	-	-	-
Model		-	-	-	-	-	-	-	-	-	-
Air Flow	CFM	99,700	99,700	46,800	48,000	59,000	68,355	38,000	43,400	40,500	2,500
Total Fan Static	In of H2O	3.0	3.0	3.3	3.3	3.0	3.5	3.0	2.5	3.0	0.8
Fan Brake Horsepower	BHP	75	75	38	38	40	65	27	27	29	1
Fan Efficiency	%	63%	63%	63%	65%	70%	58%	66%	63%	66%	31%
Quantity	No.	1	1	1	1	1	1	1	1	1	1
Supply Fan Driver - Electric Motor											
Make		-	-	-	-	-	-	-	-	-	-
Motor Control Type	RPM	VFD	VFD	VFD	VFD	-	-	VFD	VFD	-	-
Motor Nameplate Power	HP	75	75	40	40	50	75	40	30	40	2
Voltage	V	460	460	460	460	460	460	460	460	460	460
Speed	RPM	1,170	1,170	-	-	1,170	1,770	1,770	1,770	1,770	1,770
Nominal Efficiency	%	-	-	-	-	-	-	-	-	-	-
Quantity	No.	1	1	1	1	1	1	1	1	1	1
Pre-Heat Coil											
Capacity	MBH	5,707	5,707	2,678	2,886	-	3,945	2,153	2,471	-	-
Fluid		Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam	Steam
Steam Flow	Lbs/Hr	-	-	-	3,038	-	4,153	2,266	2,601	-	-
Fluid Rows	No.	1	1	1	1	-	1	1	1	-	-
Number of Coils	No.	-	-	-	-	-	-	-	-	-	-
Air Flow	CFM	99,700	99,700	46,800	48,000	59,000	68,355	38,000	43,400	40,500	2,500
Air Pressure Drop	In of H2O	0.3	0.3	0.3	0.3	-	0.3	0.3	0.3	-	-
Entering Air Temp. - DB	°F	10.00	10.00	10.00	10.00	-	10.00	10.00	10.00	-	-
Leaving Air Temp. - DB	°F	63.00	63.00	63.00	62.00	-	64.00	63.00	63.00	-	-
Approach	°F	-	-	-	-	-	-	-	-	-	-
Design Working Pressure	PSIG	-	-	-	-	-	-	-	-	-	-

2.7.6.4 Central Plant, Heating Hot Water: Air Handling Unit

Table 49. Central Plant Heating Hot Water Air Handling Unit Specifications

Heating and Ventilation Air Handling Unit			
H&V ID		HV-9P	
Location		River Water Pump Room	
Service		River Water Pump Room	
Make		Joy	
Supply Air Flow	CFM	43,600	
Design	Supply Fan		
	Air Flow	CFM	43,600
	Total Fan Static	In of H2O	1.8
	Fan Brake Horsepower	BHP	24
	Fan Efficiency	%	50%
	Quantity	No.	1
	Supply Fan Driver - Electric Motor		
	Motor Nameplate Power	HP	30
	Voltage	V	460
	Speed	RPM	1,770
	Nominal Efficiency	%	-
	Quantity	No.	1
	Pre-Heat Coil		
	Capacity	MBH	1,649
	Fluid		40% EG
	Fluid Flow	GPM	106
	Fluid Pressure Drop	Ft of H2O	2.9
	Entering Fluid Temp.	°F	130.00
	Leaving Fluid Temp.	°F	93.00
	Fluid Rows	No.	4
	Number of Coils	No.	-
	Air Flow	CFM	43,600
	Air Pressure Drop	In of H2O	0.8
Entering Air Temp. - DB	°F	10.00	
Leaving Air Temp. - DB	°F	49.00	
Approach	°F	-	
Design Working Pressure	PSIG	-	

2.7.6.5 Central Plant, Steam: Unit Heaters and Cabinet Unit Heaters

Table 50. Central Plant Steam Unit Heater and Cabinet Unit Heater Specifications

Unit Heaters							
Unit Heater ID		UH-3P to 5P	EH-1P	EH-2P & 3P	EH-4P	EH-5P	
Location		+19 MER	Entrance at Car Ramps	Car Ramp	Car Ramp	Truck Ramp	
Service		Space Heater	Entrance Heater	Car Ramp Entrance	Car Ramp & Service Entrance	Truck Ramp Entrance	
Make		Trane	Trane	DynaForce	DynaForce	DynaForce	
Model Number		UHSA 91-S8	N46	96 L/1	144 L/1	168-H	
Serial Number		-	-	-	-	-	
Quantity of Units		3	1	2	1	1	
Unit Heater Capacity		MBH	89	48	447	670	1216
Design	Air Flow Data						
	Supply Air Flow	CFM	1,740	600	7,580	11,375	31,500
	Discharge Air Temp	°F	108.00	138.00	-	-	-
	Motor Data						
	Motor Nameplate Power	HP	1/6	1/8	1.5	1.5	5
	Voltage	V	120	120	460	460	460
	Speed	RPM	1100	1200	900	900	1200
	Nominal Efficiency	%	-	-	-	-	-
	Quantity	No.	1	1	2	3	3
	Heating Coil						
	Capacity	MBH	89	48	447	670	1,216
	Fluid		Steam	Steam	Steam	Steam	Steam
	Steam Flow	Lbs/Hr	94	50	470	705	1,280
	Steam Inlet Pressure	PSIG	5.0	5.0	5.0	5.0	5.0
	Steam Temperature	°F	-	-	-	-	-
	Air Flow	CFM	1,740	600	7,580	11,375	31,500
	Leaving Air Temp. - DB	°F	108.00	138.00	-	-	-
Approach	°F	-	-	-	-	-	
Design Working Pressure	PSIG	-	-	-	-	-	

3 Existing System Sequences

3.1 Cooling System Controls

The chilled water system is designed to efficiently manage cooling for building comfort loads year-round while also optimizing energy usage during off-peak hours by charging thermal storage tanks. The following components work together to maintain desired temperatures and reduce energy consumption:

3.1.1 Chillers

The chillers are primarily used to supply the building comfort cooling loads 24/7/365, as well as to feed the thermal storage tanks during off-peak hours. When staging the plant, the newer VFD chiller is prioritized. The chillers maintain a chilled water supply temperature set-point of 41°F.

3.1.2 Chilled Water Pumps

3.1.2.1 *Primary Chilled Water Pumps*

The 10 primary chilled water pumps are equipped with VFDs. The plant operates one primary chilled water pump at a reduced speed per chiller.

3.1.2.2 *Secondary Chilled Water Pumps*

Of the 10 secondary chilled water pumps, 9 are equipped with VFDs. These pumps are staged so that at least one pump from each group is running to supply a tower when the connecting tower requires chilled water. The pumps modulate based on a remote differential pressure (ΔP) set-point. The set-points resets based on the tower heat exchanger valve position.

3.1.3 Thermal Storage Pumps and Storage Tanks

3.1.3.1 *Primary Thermal Energy Storage Pumps*

The primary thermal energy storage pumps are staged manually based on dispatching the thermal energy storage system.

3.1.3.2 Secondary Thermal Energy Storage Pumps

The secondary thermal energy storage pumps are also staged manually based on dispatching the thermal energy storage system.

3.1.3.3 Thermal Energy Storage Tanks

The thermal energy storage tanks are arranged in a pump-in, pump-out configuration and are used during summer to reduce total plant chilled water and power demand

3.1.4 Heat Rejection System Controls

3.1.4.1 River Water Heat Exchangers

The river water heat exchangers are staged based on load and heat exchanger approach to maintain condenser water supply temperature (CWST) to the chiller or used for free cooling if the river water temperature permits.

3.1.4.2 River Water Pumps

The river water pumps are staged based on condenser water demand and operate using VFDs.

3.1.4.3 Condenser Water Pumps

The condenser water pumps are staged one per chiller. Two pumps are equipped with VFDs and used for free cooling.

3.2 Heating System Controls

The complex's Heating system controls are described in Section 3.2.

3.2.1 Tower A, Lower House: Heating Hot Water Generation

The lower house of Tower A is provided with heating hot water with the source energy being the steam service. The following shell and tube heat exchangers have a modulating control valve that control the steam flow at to meet the heating hot water supply temperature setpoint. The setpoint rests from 100°F to 200°F with outdoor air temperature varying from 50°F to 0°F.

- HX-3A through HX-7A, serves levels 3 through 21

- HX-4A, which serves the service level, street level, and 2nd floor
- HX-5A, this provides hot water for the garage ventilation units (HV-1A and HV-2A)

3.2.2 Tower A, Upper House: Heating Hot Water Generation

The upper house of Tower A is provided with heating hot water for its space heating; the source energy for the heating hot water being the site's steam service. Heat is exchanged through a shell and tube heat exchangers. The shell and tube heat exchanger have a modulating control valve which controls the steam flow to meet the heating hot water supply temperature setpoint. The setpoint rests from 100°F to 200°F with outdoor air temperature. The heat exchanger tags and their services are listed below.

- HX-1A, which serves levels 22 through 40
- HX-2A, which provides auxiliary heating for AC-41A by heating the chilled water entering the reclaim coil for freeze protection. AC-41A provides ventilation air for the upper house office floors. The steam valve on the heat exchanger opens to 100%, and P26B activates if the average post-reclaim temperature falls below 48°F.

The Penthouse Mechanical Equipment Room unit heaters have UH-5A through UH-12A which use standalone controls to maintain the locally set space temperature set-point.

3.2.3 Tower A, Lower House: Steam Use

The following air handling systems use a steam heating coil for tempering of air during the heating season. The equipment name, its service and the control strategy are listed below:

- AC-1A, which serves the lobby
- AC-2A, which serves the retail areas
 - The heating mode controls for AC-1A and AC-2A is such that in heating mode, the steam valve controls to maintain the supply air temperature set-point. The set-point is adjustable, with the unoccupied minimum and maximum range being 60°F to 74°F and the occupied minimum and maximum range being 65°F to 75°F. The supply air temperature set-point resets based on the space temperature set-point.
 - The unit heaters which serve the loading dock have standalone controls which modulate the steam coil control valve to meet the space temperature setpoint. The unit heater tags are UH-1A, UH-2A, UH-3A, and UH-4A.

3.2.4 Tower A, Lower House: Heating Hot Water Use

The following unit heaters and cabinet unit heaters have standalone controls which are used to maintain the space temperature for which they serve. The standalone controls modulate the hot water control valve. The tag and service for the heaters include:

- UH-14A, UH-15A, UH-16A, serving Garage Unit Heaters
- UH-18A through UH-25A, serving Gatehouse Penthouse
- CUH-1A through CUH-4A, serving Street Level Entrance
- CUH-5A through CUH-6A, serving Street Level Service Area

The following heating and ventilation units (HV) have a modulating heating hot water valve to maintain a discharge air temperature set-point of 70°F. The units activate when the outdoor air temperature falls below 50°F.

- HV-1A, which provides Garage Ventilation
- HV-2A, which serves the Service Level

3.2.5 Tower B, Lower House: Heating Hot Water Generation

The lower house of Tower B is provided with heating hot water for its space heating; the source energy for the heating hot water being the site's steam service. Heat is exchanged through a shell and tube heat exchangers. The shell and tube heat exchangers have a modulating control valve which controls the steam flow at to meet the heating hot water supply temperature setpoint. The setpoint rests from 110°F to 160°F with outdoor air temperature.

- HX-3B and HX-4B, which serve levels 3 through 25
- HX-5B, which serves the street level and the lobby
- HX-6B, which serves the bridge that connects Building A to Building B
- HX-7B and HX-8B, which provide hot water for the garage ventilation unit (HV-1B)

3.2.6 Tower B, Upper House: Heating Hot Water Generation

The steam control valve for the shell and tube heat exchanger (HX-2B) modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 110°F and 160°F depending on the outdoor air temperature, which ranges from 0°F to 60°F. This unit serves floors 26 through 44.

To prevent freezing, heat exchanger HX-1B heats the chilled water entering the reclaim coil on AC-46B. The steam valve on the heat exchanger opens to 100%, and P26B activates if the average post-reclaim temperature falls below 48°F. AC-46B provides ventilation to the upper house office floors.

3.2.7 Tower B, Lower House: Steam Use

In heating mode, the preheat steam control valve within air handling systems AC-1B and AC-2B modulates to maintain the supply air temperature set-point. The set-point adjusts between 55°F and 80°F depending on the return air temperature, which ranges from 78°F to 68°F. The south lobby reheat steam coil control valve in AC-1B modulates to maintain the lobby set-point of 68°F. AC-1B serves the lobby and street level spaces. AC-2B service the Liberty Street bridge.

In heating mode, the preheat steam control valve AC-45B, a 100% outdoor air unit, modulates to maintain a heating set-point of 65°F. This unit provides outside air for levels 3 through 15.

In winter mode, the preheat steam control valve AC-47B modulates to maintain the supply air temperature set-point. The set-point adjusts between 55°F and 80°F depending on the return air temperature, which ranges from 78°F to 66°F. This unit serves the tenant storage spaces in the cellar level.

In heating mode, the preheat steam control valve AC-48B modulates to maintain the supply air temperature set-point. The set-point adjusts between 55°F and 80°F depending on the return air temperature, which ranges from 78°F to 66°F. This unit serves the basement level core spaces.

The steam coil modulating valve for the unit HV-3B, modulates to maintain the space temperature set-point of 55°F. This unit provides outside air for the retail spaces.

The following unit heaters have steam coils and use standalone controls to maintain their local space temperature setpoints. The unit heater names and service are listed below:

- UH-13B through UH-20B ; Gatehouse Penthouse
- UH-27B and UH-28B ; The Link Mechanical Equipment Room
- UH-29B ; The East Generator Room of the Gatehouse

3.2.8 Tower B, Lower House: Heating Hot Water Use

The street-level exit unit heaters (CUH-1B through CUH-7B) use standalone controls to maintain the locally set space temperature set-point.

The heating and ventilation units (HV-1B and HV-2B) have a modulating hot water coil control valve which maintains a discharge air setpoint of 65°F. The unit HV-1B serves the garage. The unit HV-2B serves the core spaces in the cellar.

3.2.9 Tower B, Upper House: Heating Hot Water Use

The Penthouse Mechanical Equipment Room unit heaters in UH-7B through UH-12B use standalone controls to maintain the locally set space temperature set-point.

3.2.10 Tower C, Lower House: Heating Hot Water Generation

The steam control valve in the shell and tube heat exchangers (HX-1C, HX-2C, and HX-6C) modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 120°F and 200°F depending on the outdoor air temperature. The heat exchangers serve the items listed below:

- HX-1C; 3rd floor through 27th floor
- HX-2C; Street level, lobby and 2nd floor
- HX-6C; provides hot water for heating and ventilation units (HV-1C and HV-2C). These HV units serve the garage and mechanical room respectively. The heat exchanger HX-7C serves as a backup for HX-6C.

3.2.11 Tower C, Upper House: Heating Hot Water Generation

In Tower C's Upper House, the heat exchanger in HX-4C provides auxiliary heating for AC-52C by heating the chilled water entering the reclaim coil to prevent freeze damage. The heat exchanger pump activates when required, and the steam valve modulates to maintain a temperature of 55°F if the average post-reclaim temperature drops below 45°F. Additionally, the steam control valve in HX-5C modulates to maintain the heating hot water supply temperature set-point, which adjusts between 120°F and 200°F depending on the outdoor air temperature, which ranges from 60°F to 10°F. The system activates when the outdoor air temperature falls below 50°F and deactivates when the temperature rises above 60°F.

The heat exchanger in HX-4C provides auxiliary heating for AC-52C by heating the chilled water entering the reclaim coil for freeze protection. The heat exchanger pump activates when necessary, and the steam valve modulates to maintain a temperature of 55°F if the average postreclaim temperature drops below 45°F.

The steam control valve in HX-5C modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 120°F and 200°F based on outdoor air temperature.

3.2.12 Tower C, Lower House: Steam Use

In heating mode, the preheat steam control valve in AC-1C modulates to maintain the supply air temperature set-point. The set-point adjusts between 50°F and 65°F based on the lobby temperature set-point of 70°F.

In heating mode, the heating hot water valve in AC-2C modulates to maintain the supply air temperature set-point of 55°F.

In heating mode, the heating hot water valve in AC-303C and AC-301C controls to maintain the supply air temperature set-point. The set-point is adjustable, with the unoccupied minimum and maximum range between 60°F to 74°F and the occupied minimum and maximum between 65°F and 75°F. The supply air temperature set-point adjusts based on the space temperature set-point.

3.2.13 Tower C, Lower House: Heating Hot Water Use

The heating hot water valve in HV-1C modulates to maintain discharge air temperature set-point of 70°F. The unit activates when the outdoor air temperature drops below setpoint. This unit serves the garage.

The heating hot water valve in HV-2C modulates to maintain the space temperature set-point of 70°F. The unit activates when the outdoor air temperature falls below 60°F. This unit serves a mechanical equipment room.

3.2.14 Tower D, Lower House: Heating Hot Water Generation

The steam control valve in shell and tube heat exchangers (HX-2D, HX-3D, HX-4D, HX-6D, HX-7D) modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 120°F to 200°F depending on the outdoor air temperature, which ranges from 60°F to 10°F. The system activates below 55°F and deactivates above 60°F outdoor air temperature. The heat exchangers serve the items listed below.

- HX-2D; Heating Hot Water for Mid-Rise Office Floors
- HX-3D and HX-4D; Heating Hot Water for the trench fan-coil units serving the lobby
- HX-6D and HX-7D; Heating Hot Water for Low-Rise office floors

3.2.15 Tower D, Upper House: Heating Hot Water Generation

The steam control valve in HX-1D modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 120°F to 200°F depending on the outdoor air temperature, which ranges from 60°F to 10°F. The system activates below 50°F and deactivates above 60°F outdoor air temperature.

The heat exchanger in HX-5D heats the chilled water entering the reclaim coil on AC-35C for freeze protection. The heat exchanger pump starts when necessary, and the steam valve modulates to maintain a temperature of 55°F if the average post-reclaim temperature drops below 45°F.

3.2.16 Tower D, Lower House: Steam Use

In heating mode, the preheat steam control valve in AC-1D modulates to maintain the supply air temperature set-point. The set-point adjusts between 50°F to 65°F depending on the lobby temperature set-point of 70°F.

In heating mode, the heating hot water valve in AC-2DN modulates to maintain the supply air temperature set-point, which is fixed at 55°F.

In heating mode, the heating hot water valve in AC-2DS controls to maintain the air temperature set-point. The set-point is adjustable, with the unoccupied minimum and maximum range being 60°F to 74°F and the occupied minimum and maximum range being 65°F to 75°F. The supply air temperature set-point resets based on the space temperature set-point.

In heating mode, the steam preheat valves in SF-1D and SF-1D modulate to maintain the supply air temperature set-point of 50°F.

The unit heater (UH-44D) located in the +0'-6" Mechanical Equipment Room, uses standalone controls to maintain the locally set space temperature set-point.

3.2.17 Tower D, Lower House: Heating Hot Water Use

The unit heaters (UH-42D and UH-43D) located in the 3rd floor penthouse, use standalone controls to maintain the locally set space temperature set-point.

The cabinet unit heaters (CUH-1D and CUH-2D) located in stairwells 6, 9, and 10, use standalone controls to maintain the locally set space temperature set-point.

3.2.18 Tower D, Upper House: Heating Hot Water Use

The unit heaters (UH-36D through UH-40D) located in the Penthouse Mechanical Equipment Room, use standalone controls to maintain the locally set space temperature set-point.

3.2.19 Winter Garden: Heating Hot Water Generation

The steam control valve in HX-1W and HX-2W modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 120°F and 200°F based on the outdoor air temperature. These heat exchangers provide Heating Hot Water for the Winter Garden.

3.2.20 Winter Garden: Steam Use

In heating mode, the preheat steam control valve in the following air handling systems (AC-1W, AC-2W, AC-5W, AC-6W, AC-7W, AC-8W, AC-9W, AC-10W, and AC-11W) modulates to maintain the supply air temperature set-point. The set-point adjusts between 50°F to 75°F based on outdoor air temperature, ranging from 60°F and 10°F. The units AC-1W through AC-8W serve the atrium space. The units AC-9W and AC-11W serve retail spaces. AC-10W serves the east side lobby entrance.

In heating mode, the preheat steam control valve in the heating and ventilation units (HV-1W and HV-2W) modulate to maintain the supply air temperature set-point. The set-point adjusts between 50°F to 75°F based on outdoor air temperature. The unit HV-1W serves the Pipe Tunnel. The unit HV-2W serves the PRV room and storage space.

3.2.21 Central Plant, Heating Hot Water Generation

The steam control valve in the shell and tube heat exchangers (HX-1P) modulates to maintain the heating hot water supply temperature set-point. The set-point adjusts between 100°F and 140°F based on outdoor air temperature.

3.2.22 Central Plant: Steam Use

In heating mode, the preheat steam control valve in the heating and ventilation units (HV-1P, HV-2P, HV-3P, HV-4P, HV-5P, HV-6P, HV-7P, HV-8P, HV-9P, HV-10P, HV-11P,) modulate to maintain the supply air temperature set-point. The set-point adjusts between 50°F to 65°F based on outdoor air temperature. The unit HV-1P through HV-3P serve the garage. HV-4P, HV-5P, HV-7P, HV-9P, HV-10P and HV-11P serve mechanical or electrical rooms. The unit HV-6P serves the core. The unit HV-8P serves the Truck Dock.

4 Utility Data Analysis

Utility bill information is provided in Figure 2 and Figure 3 and Table 51. Figure 2 illustrates the monthly electrical consumption from 2015 to 2022, while Figure 3 shows the average monthly electrical demand for the same period. Additionally, Table 51 displays the monthly steam consumption and associated costs for 2021.

Figure 2. Monthly Electric Energy Consumption (2015–2022)

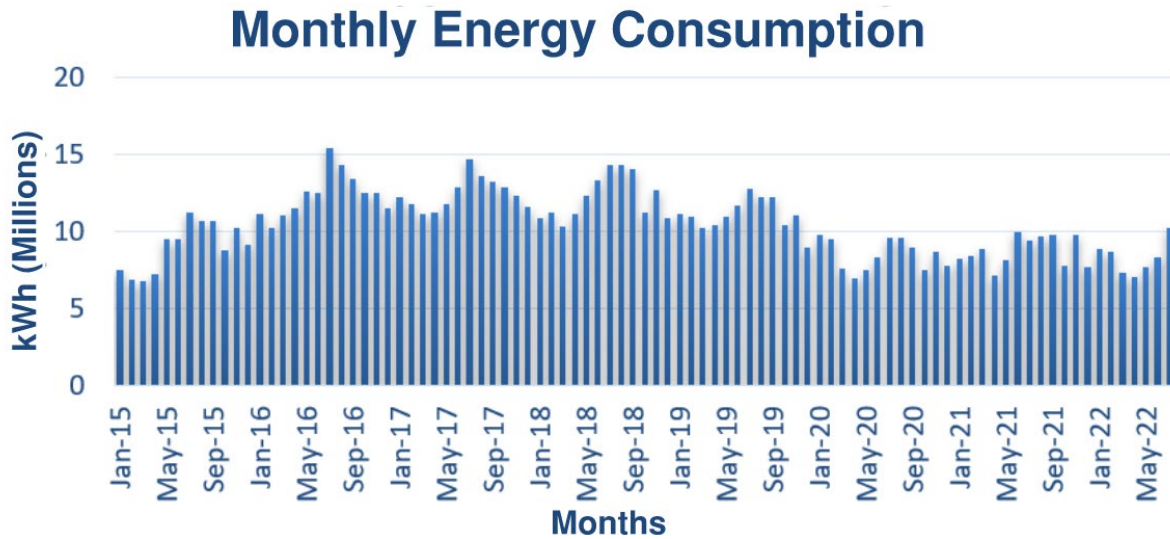


Figure 3. Monthly Electric Demand (2015–2022)

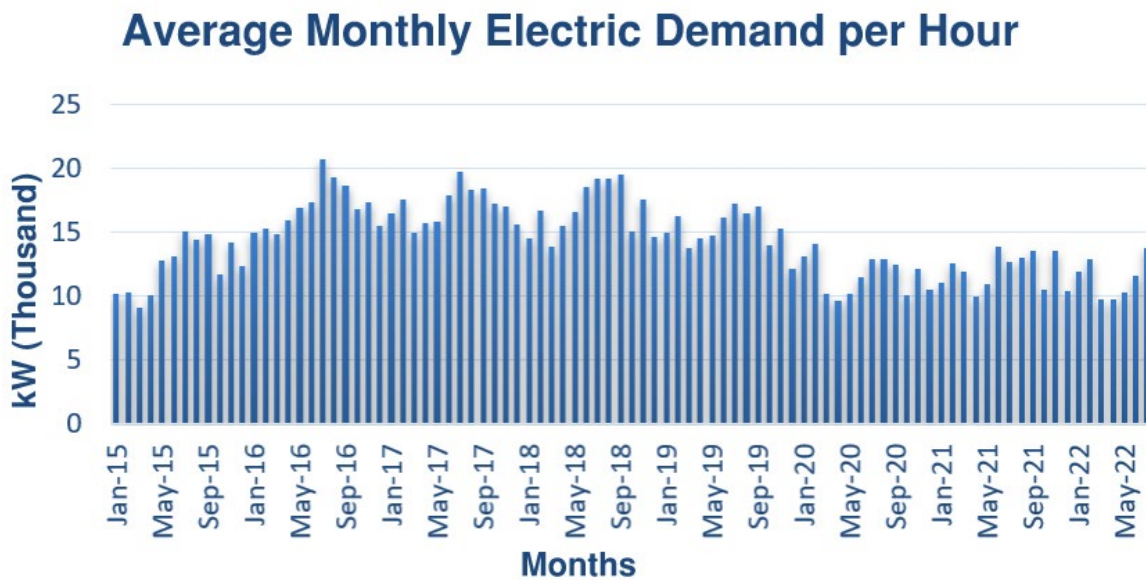


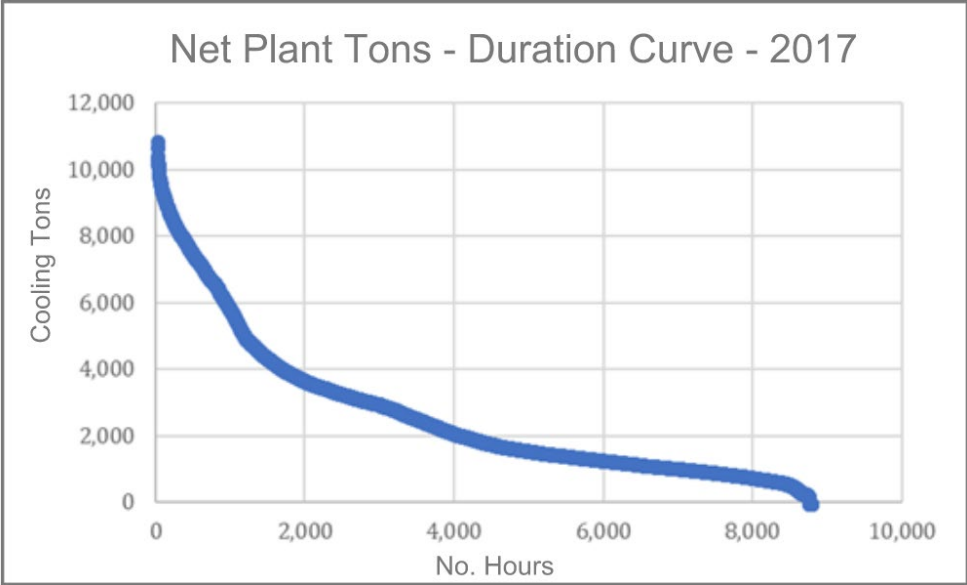
Table 51. Monthly Steam Consumption and Cost (2021)

Date Range	Cost	klbs	\$/klb
Dec 13 - Jan 14	\$ 920,819.39	30,332.29	\$ 30.36
Jan 14 - Feb 16	\$ 1,087,311.91	36162.8000	\$ 30.07
Feb 16 - Mar 21	\$ 796,961.97	24089.2400	\$ 33.08
Mar 21- Apr 15	\$ 516,298.22	16537.0100	\$ 31.22
April 15 - May 13	\$ 120,834.84	7878.9900	\$ 15.34
May 13 - Jun 14	\$ 62,616.31	4555.2000	\$ 13.75
Jun 12 - Jul 14	\$ 58,728.26	3627.8100	\$ 16.19
Jul 14 - Aug 12	\$ 49,149.59	3228.3000	\$ 15.22
Aug 12 - Sep 11	\$ 56,407.72	3517.0300	\$ 16.04
Sep 13 - Oct 13	\$ 59,562.41	3485.8200	\$ 17.09
Oct 13 - Nov 10	\$ 263,778.51	7648.1100	\$ 34.49
Oct 13 - Nov 10	\$ 658,886.81	18279.7000	\$ 36.04
Total	\$4,651,355.95	\$ 159,342.30	\$ 29.19

5 Cooling Load Analysis

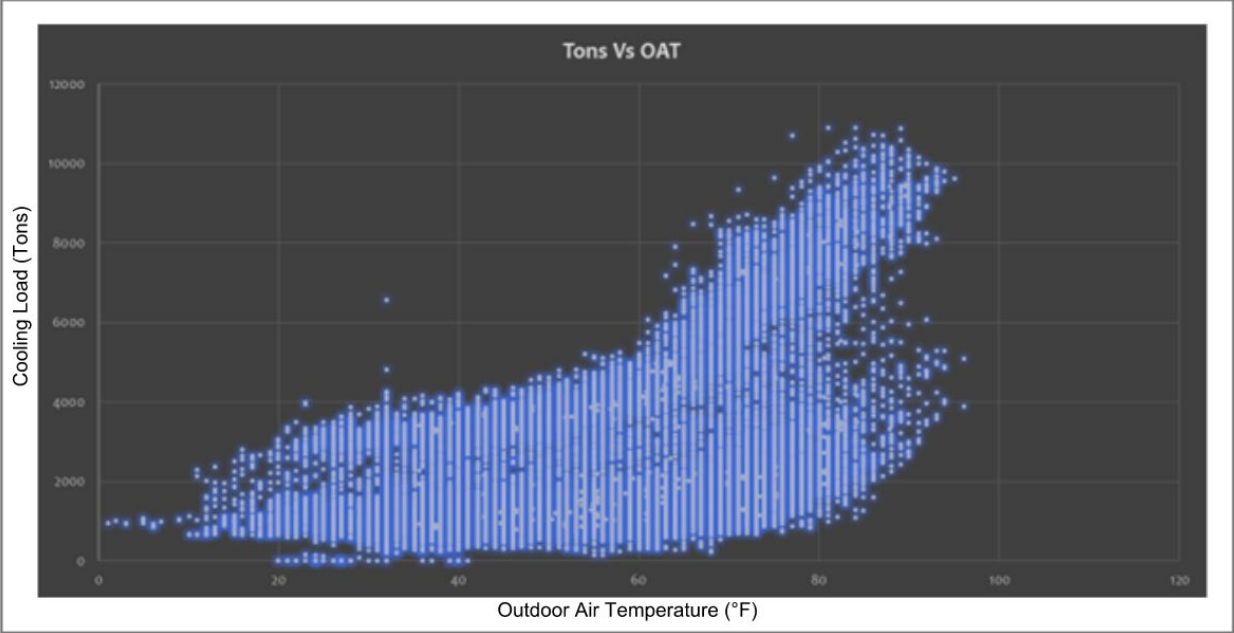
The team calculated the existing load for the central chiller plant using data obtained from site. Figure 4 presents the cooling load duration curve for the 2017 year, based on measured building chilled water flow rates and differential temperature (ΔT).

Figure 4. Net Plant Tons, Duration Curve (2017)



The graph shows that the peak cooling load for the facility is approximately 11,363 tons. This load aligns with the building’s square footage and usage.

Figure 5. Cooling Tons versus Outdoor Air Temperature



6 Heating Load Analysis

The heat load was generated using an 8,760-energy model, which calculates steam usage for each piece of equipment in the six subdivided areas of the complex, each equipped with a submeter.

Table 52 through Table 58 compare the modeled heating load to the past five years of submeter steam data for various areas within the building complex, including Towers A, B, C, D, the Winter Garden, Central Plant, and Coned Meter, serving as a validation exercise.

Table 52. Tower A Modeled Heating Load versus Past Five-Year Submeter Steam Data

Tower A						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	7,163	4,917	4,771	4,512	3,958	4,143
Feb	6,109	6,123	5,005	5,648	6,221	3,341
Mar	3,924	5,382	3,809	3,674	3,748	3,789
Apr	3,704	3,107	2,729	2,465	2,082	1,469
May	1,971	1,740	1,801	1,346	1,352	1,114
Jun	1,414	944	648	960	716	898
July	1,072	740	417	920	618	924
Aug	989	813	616	997	723	927
Sep	1,042	603	902	864	839	915
Oct	1,069	719	860	717	1,040	1,408
Nov	2,195	2,220	4,588	1,286	1,236	2,914
Dec	5,052	4,310	3,357	3,225	0	4,525
Total	35,704	31,618	29,503	26,614	22,533	26,366

Table 53. Tower B Modeled Heating Load versus Past Five-Year Submeter Steam Data

Tower B						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	10,582	7,485	7,278	7,119	6,642	7,231
Feb	9,772	10,171	6,834	9,301	8,943	5,431
Mar	8,024	8,489	5,160	6,191	5,924	6,563
Apr	7,475	5,485	2,630	4,177	4,008	1,509
May	3,409	2,614	1,624	1,976	2,508	1,141
Jun	1,851	1,868	1,323	1,636	1,591	811
July	1,746	1,821	1,228	1,580	1,484	800
Aug	1,718	1,719	1,166	1,636	1,329	805
Sep	1,618	1,510	1,325	1,418	1,498	832
Oct	1,624	1,778	1,463	1,495	1,568	1,332
Nov	3,305	3,528	2,145	2,190	1,993	4,092
Dec	7,474	6,625	4,097	5,599	0	8,665
Total	58,598	53,093	36,273	44,319	37,489	39,212

Table 54. Tower C Modeled Heating Load versus Past Five-Year Submeter Steam Data

Tower C						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	6,182	5,077	5,021	4,865	4,910	5,956
Feb	5,190	5,791	4,773	5,764	6,513	4,639
Mar	4,077	5,120	4,194	3,726	4,728	5,358
Apr	3,742	3,905	2,595	3,067	3,436	2,316
May	2,768	2,789	1,851	2,021	2,624	2,015
Jun	1,722	2,034	1,650	1,588	1,503	1,585
July	1,586	1,822	1,521	1,521	1,420	1,584
Aug	1,587	1,606	1,435	1,491	1,310	1,591
Sep	1,724	1,530	1,557	1,530	1,492	1,610
Oct	1,699	1,634	1,975	1,604	1,681	2,143
Nov	3,042	3,548	2,785	2,179	2,044	4,089
Dec	5,240	4,591	3,577	4,386	0	6,591
Total	38,559	39,447	32,934	33,742	31,662	39,478

Table 55. Tower D Modeled Heating Load versus Past Five-Year Submeter Steam Data

Tower D						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	13,527	6,708	6,136	5,881	5,178	7,139
Feb	8,382	7,326	5,896	7,273	7,324	5,496
Mar	4,841	5,626	5,326	5,145	5,084	6,498
Apr	4,668	4,105	4,165	5,055	3,358	2,721
May	2,653	1,925	2,838	2,245	1,662	2,186
Jun	1,468	1,669	1,406	1,651	1,482	1,282
July	1,426	1,753	1,471	1,518	1,435	1,187
Aug	1,592	1,584	1,571	1,655	1,337	1,200
Sep	1,715	1,566	1,675	1,443	1,530	1,344
Oct	1,698	1,633	1,723	1,397	1,664	2,297
Nov	3,266	2,969	3,044	1,965	2,117	5,019
Dec	6,688	5,156	3,993	4,543	0	7,877
Total	51,924	42,020	39,245	39,772	32,171	44,246

Table 56. Winter Garden Modeled Heating Load versus Past Five-Year Submeter Steam Data

Winter Garden						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	1,903	924	778	947	613	1,239
Feb	1,552	1,314	719	1,175	1,059	890
Mar	932	1,160	596	740	739	1,051
Apr	1,042	1,036	508	616	562	170
May	748	10	6	207	12	79
Jun	1	0	0	0	0	5
July	2	0	0	0	0	0
Aug	0	0	0	0	0	0
Sep	0	0	0	0	0	8
Oct	0	0	0	1	0	119
Nov	1,155	809	457	505	599	685
Dec	1,385	902	683	711	0	1,431
Total	8,720	6,155	3,747	4,901	3,583	5,678

Table 57. Central Plant Modeled Heating Load versus Past Five-Year Submeter Steam Data

Central Plant						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	1,458	1,600	1,934	2,143	868	2,711
Feb	1,859	2,074	1,928	2,202	1,686	1,940
Mar	1,786	1,606	1,496	1,866	1,076	2,297
Apr	1,188	516	802	1,157	342	350
May	499	72	314	205	152	155
Jun	370	14	285	5	125	9
July	367	16	29	1	113	0
Aug	225	15	32	1	110	0
Sep	0	14	27	1	136	14
Oct	15	17	21	104	151	245
Nov	248	277	22	164	130	1,474
Dec	1,777	1,329	511	520	0	3,146
Total	9,792	7,549	7,401	8,368	4,889	12,339

Table 58. Coned Meter Modeled Heating Load versus Past Five-Year Submeter Steam Data

Main Coned Meter						
	2018	2019	2020	2021	2022	Model TMY3
	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs	Total klbs
Jan	28,000*	26,935	26,317	30,332	22,609	35,163
Feb	34,179	33,712	33,000*	36,163	31,444	26,890
Mar	27,000*	28,525	20,453	24,089	22,951	31,683
Apr	22,241	17,403	11,073	11,000*	11,000*	10,877
May	10,826	7,841	6,516	7,879	7,037	8,510
Jun	4,856	4,906	3,203	4,555	3,958	5,519
July	4,500*	4,389	4,500*	3,628	3,589	5,301
Aug	4,472	3,741	3,300*	3,228	3,341	5,362
Sep	4,487	3,842	3,517	3,517	3,637	5,714
Oct	4,428	4,310	3,867	3,486	4,663	9,456
Nov	11,783	10,504	8,277	7,648	6,930	22,932
Dec	27,131	23,998	19,053	18,280	19,000*	39,717
Total	183,902	170,105	143,075	153,805	140,157	207,124

*Denotes an estimated value in absence of an actual bill for this month.

7 Carbon Analysis

Figure 6 through Figure 8 show the historical carbon production from the site.

Figure 6. Monthly Carbon Production, Electric

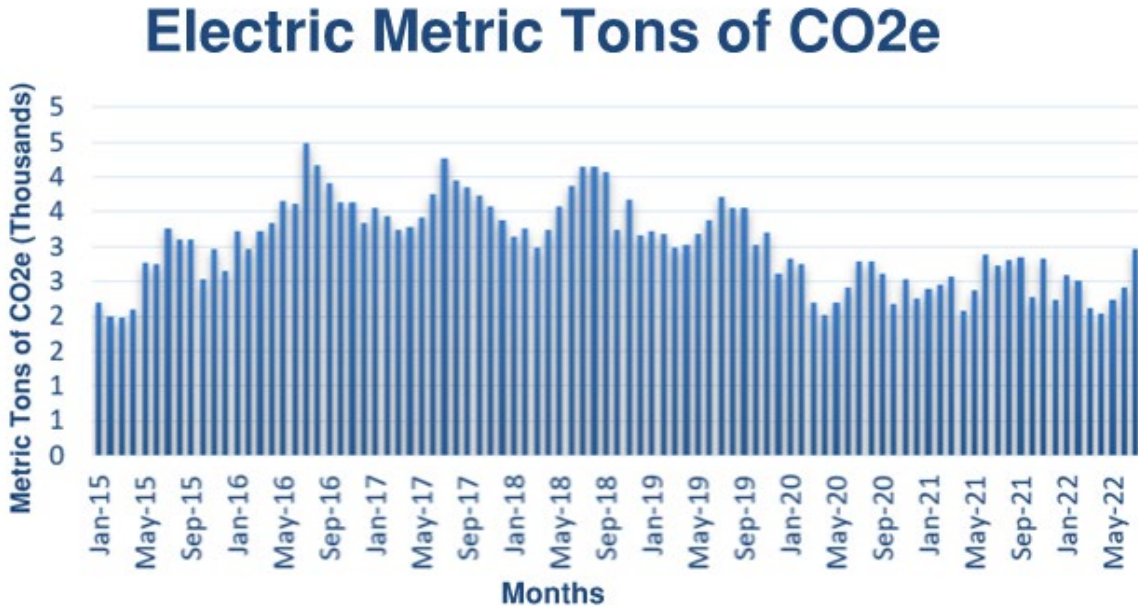


Figure 7. Monthly Carbon Production, Steam

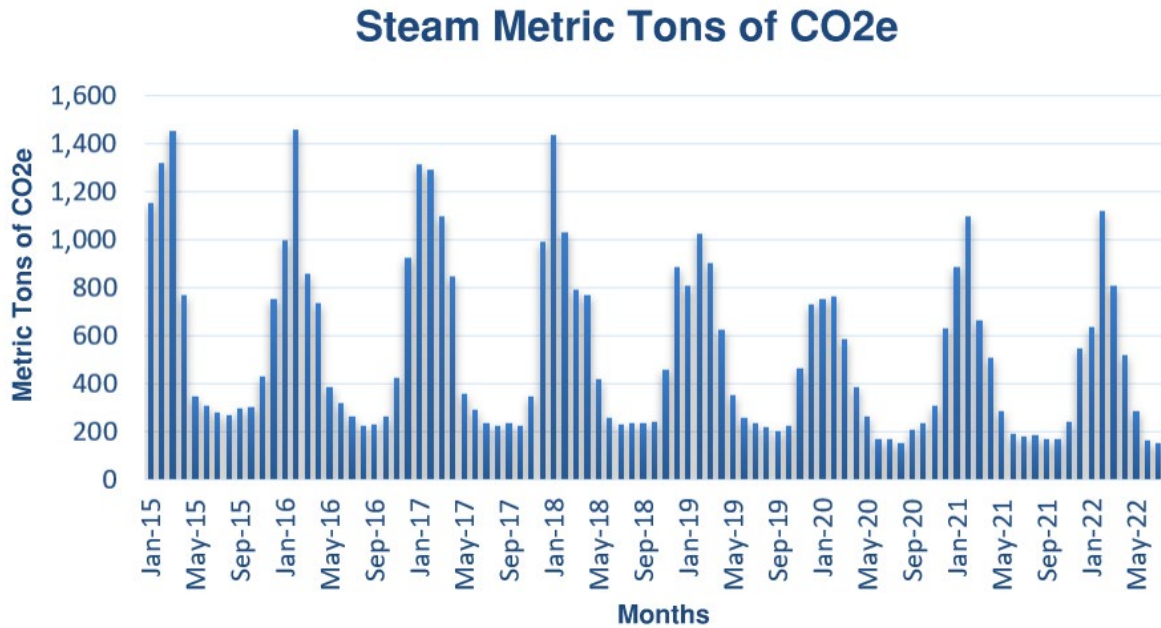
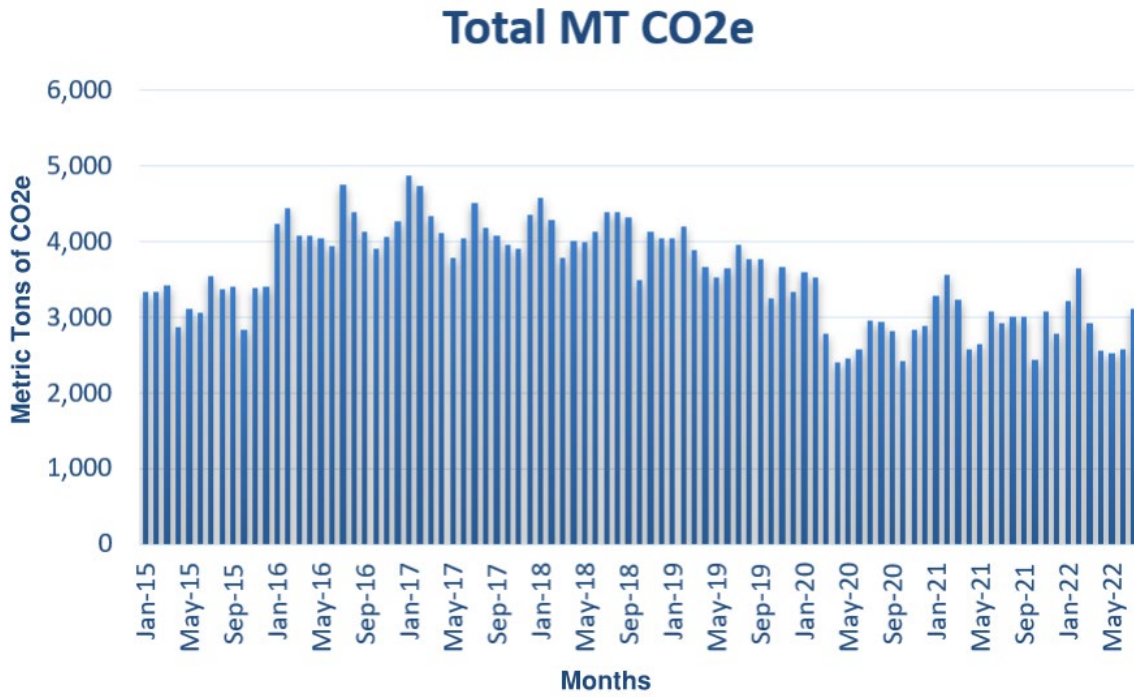


Figure 8. Monthly Carbon Production, Steam and Electric Combined



8 Heat Pump Technology Analysis

The contract requires an evaluation of four technologies, and the team assessed 15 commercially available heat pump models.

Table 59. Net Plant Tons, Duration Curve

Water Source Heat Pump Chillers / Heat Recovery Chillers																	
Chiller ID		JCI/York					Carrier					Trane					
Make		JCI/York	JCI/York	JCI/York	JCI/York	JCI/York	Carrier	Carrier	Carrier	Carrier	Carrier	Trane	Trane	Trane	Trane	Trane	
Model Number		CYK	YVWA	YMC2	OM Titan	Frick	30MP	30HXC	30HXA	30XW	19DV	RTWD	RTHD	HDW	CTV	CVHH/CDHH	
Max Capacity	Tons	2,500	300	1,000	5,500	1,200	71	265	265	400	1,000	250	430	450	2,000	2,000	
Min Capacity	Tons	300	125	165	3,000	74	50	75	75	325	500	80	125	150	125	900	
Score		1	2	4	3	5	6	4	4	2	4	5	2	2	4	4	
Refrigerant		R-134a	R-134a	R-134a	R-134a	Ammonia	R-410A	R-134a	R-134a	R-134a	R-1233zd	R-134a or R-513A	R-134a or R-513A	R-134a or R-513A	R-514A	R-1233zd	
Capacity Control		Independent IGVs and HGBP	VFD	VFD / IG / HGBP	VFD / IG / HGBP	Slide Valve	Compressor Staging with HGBP	Compressor Staging and Two unloaders per compressor	Compressor Staging and Two unloaders per compressor	Compressor Staging on the dual compressor machines, Slide Valve, and HGBP	VFD / IG / HGBP	VFD, Compressor Staging, Slide Valve, and HGBP	VFD, Compressor Staging, Slide Valve, and HGBP	Compressor Staging on the dual compressor machines, Slide Valve, and HGBP	VFD / IG / HGBP	VFD / IG / HGBP	
Capacity Control Process Variable		Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp and Leaving Cond Temp	Leaving Evap Temp	Leaving Evap Temp	
Design	Compressor																
	Type	Centrifugal	Screw	Centrifugal	Centrifugal	Screw	Scroll	Screw	Screw	Screw	Centrifugal	Screw	Screw	Screw	Centrifugal	Centrifugal	
	Model	YK	YVWA	YMC2	OM	Frick	30MP	30HXC	30HXA	30XW	19DV	CHHP	CHHC	HDMA	CVH/CDH		
	Stages	Dual Series Single Stage	Dual Parallel Screw	Single Single Stage	Single Two Stage or Single Three Stage	Single Compressor Single Stage Screw	Dual Parallel Single Stage Scroll	Dual or Triple Parallel Screw	Dual or Triple Parallel Screw	Single or Dual Parallel Screw	Dual Series Single Stage	Dual Parallel Screw	Dual Parallel Screw	Two Stage Centrifugal	Single or Dual parallel Two or Three Stage Centrifugal	Single or Dual parallel Two or Three Stage Centrifugal	
	Driver - Electric Motor																
	Motor Control Type	Starter	VFD	VFD	Starter or VFD	Starter	Starter	Starter	Starter	Starter	VFD	Starter or VFD	Starter or VFD	VFD	Starter or VFD	Starter of VFD	
	Full Load Efficiency	kW/Ton															
	Refrigerant Evaporator																
	Minimum Leaving Fluid Temp. Water	°F	20.00	38.00	42.00	36.00		40.00	40.00	40.00	40.00	38.00	?	?	?	34.00	?
	Minimum Leaving Fluid Temp. Glycol	°F	20.00	8.00	42.00	3.00		15.00	14.00	14.00	15.00	42.00	?	?	?	20.00	?
Refrigerant Condenser																	
Max Leaving Fluid Temp.	°F	155.00	160.00	120.00	170.00	140.00	140.00	120.00	135.00	140.00	120.00	?	?	?	105.00	?	

Based on our assessment, we identified the following viable options for specific locations:

- **The plant:**
 - York CYK
 - Potentially 19XR, YK, or CTV in series counterflow (This option will require further engagement with the manufacturer.)
- **The buildings:**
 - York YVWA
 - Carrier 30XW

9 Proposed System Modifications

9.1 M.1, Central Plant: Heat Pump

This section outlines the proposed modification and system descriptions, controls strategy, analysis of the proposed modifications, and first cost analysis for the energy conservation measure for the Central Plant heat pump. The team proposes the following system modifications across the Central Plant, various towers, and Winter Garden.

- Central Plant heat pump: The proposal includes replacing Chiller 2 and Chiller 6 with two York CYK heat pumps, each with a capacity of 1,250 tons.
- Tower A modifications: Due to the distance and complexity involved in extending piping to Tower A, no heating piping is included in this proposal for Tower A.
- Tower B modifications: The team proposes the following modifications for Tower B units and systems.
 - For these units, replace the steam preheat coil with a heating hot water coil: AC-1B, AC-2B, AC-47B, and AC-48B.
 - For the following heat exchangers, tap into the heating hot water distribution from the Central Plant using the existing heating hot water return piping to preheat the water before it enters the heat exchanger: HV-3B, HX-3B/4B, HX-5B, HX-6B, and HX-7B.
 - For the following units, install duplex heating hot water-to-domestic hot water heat exchangers to preheat the domestic hot water before it enters the steam-to-hot-water heat exchangers: H-1, H-2, H-3, H-4, H-5, H-6, and H-7.
- Tower C modifications: The team proposes the following modifications for Tower C units and systems.
 - For the following units, replace the steam preheat coil with a heating hot water coil: AC-1C, AC-2C, AC-301C, and AC-303C.
 - For the following heat exchangers, tap into the heating hot water distribution from the Central Plant using the existing heating hot water return piping to preheat the water before it enters the heat exchanger: HV-3B, HX-1C, HX-2C, HX-6C, and HX-7C.
 - For the following units, install duplex heating hot water-to-domestic hot water heat exchangers to preheat the domestic hot water before it enters the steam-to-hot-water heat exchangers: H-1, H-2, H-3, H-4, H-5, H-6, and H-7.
- Tower D modifications: The team proposes the following modifications for Tower D units and systems.
 - For the following units, replace the steam preheat coil with a heating hot water coil: AC-1D, AC-2DN, AC-2DS, SF-1D, and SF-2D.
 - For the following heat exchangers, tap into the heating hot water distribution from the Central Plant using the existing heating hot water return piping to preheat the water before it enters the heat exchanger: HX-2D, HX-3D/4D, and HX-6D/7D.

- For the following units, install duplex heating hot water-to-domestic hot water heat exchangers to preheat the domestic hot water before it enters the steam-to-hot-water heat exchangers: H-1, H-2, H-3, H-4, H-5, H-6, and H-7.
- Winter Garden modifications: The team proposes the following modifications for the Winter Garden units and systems.
- For the following units, replace the steam preheat coil with a heating hot water coil: AC-1W, AC-2W, AC-7W, AC-8W, AC-9W, AC-10W, AC-11W, HV-1W, and HV-2W.
- For the following heat exchangers, tap into the heating hot water distribution from the Central Plant using the existing heating hot water return piping to preheat the water before it enters the heat exchanger: HX-1W and HX-2W.
- Central Plant heating modifications: The team proposes the following heating modifications for the Central Plant units and systems.
- For the following units, replace the steam preheat coil with a heating hot water coil: HV-1P, HV-2P, HV-3P, HV-4P, HV-5P, HV-6P, HV-7P, HV-8P, HV-10P, HV-11P, and AC-5P.
- For the following heat exchangers, tap into the heating hot water distribution from the Central Plant using the existing heating hot water return piping to preheat the water before it enters the heat exchanger: HX-1P.

9.1.1 Proposed Controls Strategy

The team proposes the following controls strategy and adjustments to optimize the system's performance:

- **Central Plant heat pump:** Replace Chiller-2 and Chiller-6 with a York CYK heat pump. The heat pump will operate to maintain heating hot water distribution set-point, which will adjust based on outdoor air temperature. All building equipment must coordinate with this set-point.
- **Air handling units and heating and ventilation units:** Provide new hot water unit heaters in the mixed air plenum of AC and HV units. The unit heaters will take load off of the steam coil and will satisfy set-point under most conditions. The steam coil will only need to be used under extreme conditions
- **Heating hot water:** The heat pump generated hot water will preheat the local heating hot water systems. If the heat pump generated water cannot satisfy the system setpoint then the existing steam-to-hot-water heat exchanger will add the remaining degrees of heat to meet setpoint.
- **Domestic hot water:** Use the heat pump generated heating hot water to heat the domestic cold water. New domestic water heaters will be provided in order to use heating hot water as the source energy. A local steam-to-hot water heat exchanger will be provided as a means of backup for the domestic water heat source.

9.1.2 Analysis of Proposed Modification: Base Case and Energy Conservation Measure Analysis

The team proposes the following modeling process for this energy conservation measure (ECM):

- **Base case**
 - Model the base case using an 8760 Excel model built solely for each heating load.
 - Implement the schedules and controls strategy for each unit.
- **Energy conservation measure analysis**
 - Incorporate a switch into the base model to toggle the heat pump on and off, with an adjustable heating hot water supply temperature. This hourly supply temperature dictates how much heating can transfer from steam to the heat pump. The model calculates the heating hot water flow profile based on the calculated load and differential temperature.
 - Use this flow to calculate the energy consumption of the new heating hot water pump (HHWP).
 - Use the model to calculate the pump energy power draw using factory equipment data. The existing plant chiller's 8,760-kilowatt per ton (kW/ton) profile calculates the reduction in power consumption for the existing plant.

9.1.3 First Cost Analysis

Smith Engineering produced the first cost estimate for this option. The estimate for the proposed modifications is \$35,657,579, which is detailed in Table 60 through Table 65.

Table 60. M.1 Energy Model Heat Pump Cost Summary

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023					
					Client:	Brookfield					
For	M.1 - Summary				Project:	BFPL Community Heat Pump Stg A					
Basis of Estimate		<input type="checkbox"/> No Design		<input checked="" type="checkbox"/> Conceptual Design		<input type="checkbox"/> Final Design		<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost		
Division 01000 - General											
1	Tower B				\$ 510,000			\$ 670,000	\$ 1,180,000		
2	Tower C				\$ 345,000			\$ 563,125	\$ 908,125		
3	Tower D				\$ 425,000			\$ 611,875	\$ 1,036,875		
4	Winter Garden				\$ 420,000			\$ 674,875	\$ 1,094,875		
5	Central Plant				\$ 460,000			\$ 845,875	\$ 1,305,875		
Subtotal					\$ 2,160,000			\$ 3,365,750	\$ 5,525,750		
Division 23000 - Mechanical											
1	Tower B				\$ 1,420,000			\$ 1,112,500	\$ 2,532,500		
2	Tower C				\$ 1,060,000			\$ 800,000	\$ 1,860,000		
3	Tower D				\$ 790,000			\$ 627,500	\$ 1,417,500		
4	Winter Garden				\$ 950,000			\$ 662,500	\$ 1,612,500		
5	Central Plant				\$ 4,368,000			\$ 1,112,500	\$ 5,480,500		
Subtotal					\$ 8,588,000			\$ 4,315,000	\$ 12,903,000		
Division 25000 - Controls											
1	Tower B				\$ 175,000			\$ 300,000	\$ 475,000		
2	Tower C				\$ 141,000			\$ 255,000	\$ 396,000		
3	Tower D				\$ 100,000			\$ 255,000	\$ 355,000		
4	Winter Garden				\$ 110,000			\$ 300,000	\$ 410,000		
5	Central Plant				\$ 100,000			\$ 360,000	\$ 460,000		
Subtotal					\$ 626,000			\$ 1,470,000	\$ 2,096,000		
Division 26000 - Electrical											
1	Tower B				\$ 170,000			\$ 247,500	\$ 417,500		
2	Tower C				\$ 93,000			\$ 206,250	\$ 299,250		
3	Tower D				\$ 77,000			\$ 220,000	\$ 297,000		
4	Winter Garden				\$ 110,000			\$ 225,500	\$ 335,500		
5	Central Plant				\$ 270,000			\$ 412,500	\$ 682,500		
Subtotal					\$ 720,000			\$ 1,311,750	\$ 2,031,750		
Subtotal					\$ 12,094,000	Multiplier		\$10,462,500	\$ 22,556,500		
Contingency					10%			\$ 1,209,400	10%	\$ 1,046,250	\$ 2,255,650
Subtotal					\$ 13,303,400			\$11,508,750	\$ 24,812,150		
Construction Management Overhead					10%			\$ 1,330,340	10%	\$ 1,150,875	\$ 2,481,215
Profit					10%			\$ 1,330,340	10%	\$ 1,150,875	\$ 2,481,215
Subtotal Construction					\$ 15,964,080			\$13,810,500	\$ 29,774,580		
Tax					8.875%			\$ 1,416,812	0%	\$ -	\$ 1,416,812
Design Engineering										\$ 2,381,966	\$ 2,381,966
Construction Administration										\$ 1,190,983	\$ 1,190,983
Commissioning										\$ 893,237	\$ 893,237
Total Estimated Cost					\$					35,657,579	

Table 61. M.1 Energy Model Tower B Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023				
For M.1 - Tower B					Client:	Brookfield				
					Project:	BFPL Community Heat Pump Stg A				
Basis of Estimate					<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost		
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost	
Division 01000 - General										
1	Permits and Filing	1		\$ 35,000	\$ 35,000	250	\$ 250	\$ 62,500	\$ 97,500	
2	Superintended	1		\$ -	\$ -	900	\$ 225	\$ 202,500	\$ 202,500	
3	Demolition and Protection	1		\$ 175,000	\$ 175,000	750	\$ 195	\$ 146,250	\$ 321,250	
4	Rigging	1		\$ 50,000	\$ 50,000	500	\$ 225	\$ 112,500	\$ 162,500	
5	Gen Conditions	1		\$ 250,000	\$ 250,000	750	\$ 195	\$ 146,250	\$ 396,250	
Subtotal						\$ 510,000	3150	\$ 670,000	\$ 1,180,000	
Division 23000 - Mechanical										
1	Convert AC-1B to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
2	Convert AC-2B to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
3	Convert AC-45B to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
4	Convert AC-47B to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
5	Convert AC-48B to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
6	HV-3B Glycol Loop with HX	1		\$ 125,000	\$ 125,000	400	\$ 250	\$ 100,000	\$ 225,000	
7	Tempered Loop HX-1B	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000	
8	Tempered Loop HX-5B	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000	
9	Tempered Loop HX-3B/4B	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000	
10	HX Preheat HX-6B	1		\$ 95,000	\$ 95,000	400	\$ 250	\$ 100,000	\$ 195,000	
11	HX Preheat HX-7B/8B	1		\$ 95,000	\$ 95,000	400	\$ 250	\$ 100,000	\$ 195,000	
12	DHW HXs	1		\$ 110,000	\$ 110,000	400	\$ 250	\$ 100,000	\$ 210,000	
13	Piping Distribution	1		\$ 425,000	\$ 425,000	950	\$ 250	\$ 237,500	\$ 662,500	
Subtotal						\$ 1,420,000	4450	\$ 1,112,500	\$ 2,532,500	
Division 25000 - Controls										
1	Schneider Software Prog and Graphics	1		\$ 25,000	\$ 25,000	400	\$ 300	\$ 120,000	\$ 145,000	
2	Hardware and Labor	1		\$ 150,000	\$ 150,000	600	\$ 300	\$ 180,000	\$ 330,000	
3				\$ -	\$ -			\$ -	\$ -	
Subtotal						\$ 175,000	1000	\$ 300,000	\$ 475,000	
Division 26000 - Electrical										
1	Power Conduit and Wiring	1		\$ 95,000	\$ 95,000	400	\$ 275	\$ 110,000	\$ 205,000	
2	Controls Conduit and Wiring	1		\$ 75,000	\$ 75,000	500	\$ 275	\$ 137,500	\$ 212,500	
3				\$ -	\$ -			\$ -	\$ -	
Subtotal						\$ 170,000	900	\$ 247,500	\$ 417,500	
Subtotal					Multiplier	\$ 2,275,000	Multiplier	\$ 2,330,000	\$ 4,605,000	
Contingency					10%	\$ 227,500	10%	\$ 233,000	\$ 460,500	
Subtotal						\$ 2,502,500		\$ 2,563,000	\$ 5,065,500	
Construction Management Overhead					10%	\$ 250,250	10%	\$ 256,300	\$ 506,550	
Profit					10%	\$ 250,250	10%	\$ 256,300	\$ 506,550	
Subtotal Construction						\$ 3,003,000		\$ 3,075,600	\$ 6,078,600	
Tax					8.875%	\$ 266,516	0%	\$ -	\$ 266,516	
Design Engineering								\$ 486,288	\$ 486,288	
Construction Administration								\$ 243,144	\$ 243,144	
Commissioning								\$ 182,358	\$ 182,358	
Total Estimated Cost						\$		7,256,906		

Table 62. M.1 Energy Model Tower C Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023				
					Client:	Brookfield				
For M.1 - Tower C					Project:	BFPL Community Heat Pump Stg A				
Basis of Estimate					<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost		
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost	
Division 01000 - General										
1	Permits and Filing	1		\$ 25,000	\$ 25,000	250	\$ 250	\$ 62,500	\$ 87,500	
2	Superintended	1		\$ -	\$ -	750	\$ 225	\$ 168,750	\$ 168,750	
3	Demolition and Protection	1		\$ 95,000	\$ 95,000	500	\$ 195	\$ 97,500	\$ 192,500	
4	Rigging	1		\$ 50,000	\$ 50,000	500	\$ 225	\$ 112,500	\$ 162,500	
5	Gen Conditions	1		\$ 175,000	\$ 175,000	625	\$ 195	\$ 121,875	\$ 296,875	
Subtotal								\$ 563,125	\$ 908,125	
Division 23000 - Mechanical										
1	Convert AC-1C to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
2	Convert AC-2C to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
3	Convert AC-301C to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
4	Convert AC-303C to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000	
5	Tempered Loop HX-1C/7C	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000	
6	Tempered Loop HX-2C/7C	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000	
7	HX Preheat HX-6C	1		\$ 95,000	\$ 95,000	400	\$ 250	\$ 100,000	\$ 195,000	
8	DHW HXs	1		\$ 110,000	\$ 110,000	400	\$ 250	\$ 100,000	\$ 210,000	
9	Piping Distribution	1		\$ 425,000	\$ 425,000	1000	\$ 250	\$ 250,000	\$ 675,000	
10										
11										
12										
13										
Subtotal						3200		\$ 800,000	\$ 1,860,000	
Division 25000 - Controls										
1	Siemens Software Prog and Graphics	1		\$ 25,000	\$ 25,000	400	\$ 300	\$ 120,000	\$ 145,000	
2	Hardware and Labor	1		\$ 116,000	\$ 116,000	450	\$ 300	\$ 135,000	\$ 251,000	
3				\$ -	\$ -			\$ -	\$ -	
Subtotal						850		\$ 255,000	\$ 396,000	
Division 26000 - Electrical										
1	Power Conduit and Wiring	1		\$ 35,000	\$ 35,000	250	\$ 275	\$ 68,750	\$ 103,750	
2	Controls Conduit and Wiring	1		\$ 58,000	\$ 58,000	500	\$ 275	\$ 137,500	\$ 195,500	
3				\$ -	\$ -			\$ -	\$ -	
Subtotal						750		\$ 206,250	\$ 299,250	
Subtotal					Multiplier		Multiplier	\$ 1,824,375	\$ 3,463,375	
Contingency					10%		10%	\$ 182,438	\$ 346,338	
Subtotal								\$ 2,006,813	\$ 3,809,713	
Construction Management Overhead					10%	\$ 180,290	10%	\$ 200,681	\$ 380,971	
Profit					10%	\$ 180,290	10%	\$ 200,681	\$ 380,971	
Subtotal Construction						\$ 2,163,480		\$ 2,408,175	\$ 4,571,655	
Tax					8.875%	\$ 192,009	0%	\$ -	\$ 192,009	
Design Engineering								\$ 365,732	\$ 365,732	
Construction Administration								\$ 182,866	\$ 182,866	
Commissioning								\$ 137,150	\$ 137,150	
Total Estimated Cost					\$			5,449,412		

Table 63. M.1 Energy Model Tower D Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
					Client:	Brookfield			
For M.1 - Tower D					Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 25,000	\$ 25,000	250	\$ 250	\$ 62,500	\$ 87,500
2	Superintended	1		\$ -	\$ -	750	\$ 225	\$ 168,750	\$ 168,750
3	Demolition and Protection	1		\$ 175,000	\$ 175,000	750	\$ 195	\$ 146,250	\$ 321,250
4	Rigging	1		\$ 50,000	\$ 50,000	500	\$ 225	\$ 112,500	\$ 162,500
5	Gen Conditions	1		\$ 175,000	\$ 175,000	625	\$ 195	\$ 121,875	\$ 296,875
Subtotal						2875		\$ 611,875	\$ 1,036,875
Division 23000 - Mechanical									
1	Convert AC-1D to Hot Water	1		\$ 75,000	\$ 75,000	160	\$ 250	\$ 40,000	\$ 115,000
2	Convert SF-3-1D to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
3	Convert SC-SF-2D to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
4	Tempered Loop HX-2D	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
5	Tempered Loop HX-4D	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
6	DHW HXs	1		\$ 110,000	\$ 110,000	400	\$ 250	\$ 100,000	\$ 210,000
7	Piping Distribution	1		\$ 325,000	\$ 325,000	950	\$ 250	\$ 237,500	\$ 562,500
8									
9									
10									
11									
12									
13									
Subtotal						2510		\$ 627,500	\$ 1,417,500
Division 25000 - Controls									
1	TEC Software Prog and Graphics	1		\$ 25,000	\$ 25,000	400	\$ 300	\$ 120,000	\$ 145,000
2	Hardware and Labor	1		\$ 75,000	\$ 75,000	450	\$ 300	\$ 135,000	\$ 210,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal						850		\$ 255,000	\$ 355,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 35,000	\$ 35,000	300	\$ 275	\$ 82,500	\$ 117,500
2	Controls Conduit and Wiring	1		\$ 42,000	\$ 42,000	500	\$ 275	\$ 137,500	\$ 179,500
3				\$ -	\$ -			\$ -	\$ -
Subtotal						800		\$ 220,000	\$ 297,000
Subtotal					Multiplier			\$ 1,392,000	\$ 3,106,375
Contingency					10%			\$ 139,200	\$ 310,638
Subtotal								\$ 1,531,200	\$ 3,417,013
Construction Management Overhead					10%			\$ 153,120	\$ 341,701
Profit					10%			\$ 153,120	\$ 341,701
Subtotal Construction								\$ 1,837,440	\$ 4,100,415
Tax					8.875%			\$ 163,073	\$ 163,073
Design Engineering								\$ 328,033	\$ 328,033
Construction Administration								\$ 164,017	\$ 164,017
Commissioning								\$ 123,012	\$ 123,012
Total Estimated Cost								\$ 4,878,550	\$ 4,878,550

Table 64. M.1 Energy Model Winter Garden Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
					Client:	Brookfield			
For	M.1 - Winter Garden				Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 25,000	\$ 25,000	250	\$ 250	\$ 62,500	\$ 87,500
2	Superintended	1		\$ -	\$ -	900	\$ 225	\$ 202,500	\$ 202,500
3	Demolition and Protection	1		\$ 175,000	\$ 175,000	900	\$ 195	\$ 175,500	\$ 350,500
4	Rigging	1		\$ 45,000	\$ 45,000	500	\$ 225	\$ 112,500	\$ 157,500
5	Gen Conditions	1		\$ 175,000	\$ 175,000	625	\$ 195	\$ 121,875	\$ 296,875
Subtotal						3175		\$ 674,875	\$ 1,094,875
Division 23000 - Mechanical									
1	Convert AC-1W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
2	Convert AC-2W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
3	Convert AC-5W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
4	Convert AC-7W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
5	Convert AC-8W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
6	Convert AC-9W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
7	Convert AC-10W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
8	Convert AC-11W to Hot Water	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
9	Tempered Loop HX-1W/2W	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
10	Piping Distribution	1		\$ 285,000	\$ 285,000	750	\$ 250	\$ 187,500	\$ 472,500
11									
12									
13									
Subtotal						2650		\$ 662,500	\$ 1,612,500
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 25,000	\$ 25,000	400	\$ 300	\$ 120,000	\$ 145,000
2	Hardware and Labor	1		\$ 85,000	\$ 85,000	600	\$ 300	\$ 180,000	\$ 265,000
3				\$ -				\$ -	\$ -
Subtotal						1000		\$ 300,000	\$ 410,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 35,000	\$ 35,000	320	\$ 275	\$ 88,000	\$ 123,000
2	Controls Conduit and Wiring	1		\$ 75,000	\$ 75,000	500	\$ 275	\$ 137,500	\$ 212,500
3				\$ -				\$ -	\$ -
Subtotal						820		\$ 225,500	\$ 335,500
Subtotal					Multiplier		Multiplier	\$ 1,862,875	\$ 3,452,875
Contingency					10%		10%	\$ 186,288	\$ 345,288
Subtotal								\$ 2,049,163	\$ 3,798,163
Construction Management Overhead					10%		10%	\$ 204,916	\$ 379,816
Profit					10%		10%	\$ 204,916	\$ 379,816
Subtotal Construction								\$ 2,458,995	\$ 4,557,795
Tax					8.875%		0%	\$ -	\$ 186,269
Design Engineering								\$ 364,624	\$ 364,624
Construction Administration								\$ 182,312	\$ 182,312
Commissioning								\$ 136,734	\$ 136,734
Total Estimated Cost								\$ 5,427,733	

Table 65. M.1 Energy Model Central Plant Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
					Client:	Brookfield			
For	M.1 - Central Plant				Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 50,000	\$ 50,000	250	\$ 250	\$ 62,500	\$ 112,500
2	Superintended	1		\$ -	\$ -	1400	\$ 225	\$ 315,000	\$ 315,000
3	Demolition and Protection	1		\$ 75,000	\$ 75,000	1200	\$ 195	\$ 234,000	\$ 309,000
4	Rigging	1		\$ 250,000	\$ 250,000	500	\$ 225	\$ 112,500	\$ 362,500
5	Gen Conditions	1		\$ 85,000	\$ 85,000	625	\$ 195	\$ 121,875	\$ 206,875
Subtotal								\$ 845,875	\$ 1,305,875
Division 23000 - Mechanical									
1	Heat Pump	2		\$ 850,000	\$ 1,700,000	500	\$ 250	\$ 125,000	\$ 1,825,000
2	Convert HV-1P/2P/3P to Glycol	1		\$ 225,000	\$ 225,000	200	\$ 250	\$ 50,000	\$ 275,000
3	Convert HV-4P/5P to Glycol	1		\$ 145,000	\$ 145,000	200	\$ 250	\$ 50,000	\$ 195,000
4	Convert HV-6P to Glycol	1		\$ 145,000	\$ 145,000	200	\$ 250	\$ 50,000	\$ 195,000
5	Convert HV-7P to Glycol	1		\$ 145,000	\$ 145,000	200	\$ 250	\$ 50,000	\$ 195,000
6	Convert HV-8P to Glycol	1		\$ 145,000	\$ 145,000	200	\$ 250	\$ 50,000	\$ 195,000
7	HHW Pumps	3		\$ 76,000	\$ 228,000	400	\$ 250	\$ 100,000	\$ 328,000
8	Plant Piping	1		\$ 110,000	\$ 110,000	400	\$ 250	\$ 100,000	\$ 210,000
9	Piping Distribution	1		\$ 325,000	\$ 325,000	650	\$ 250	\$ 162,500	\$ 487,500
10	Tunnel Piping	1		\$ 1,200,000	\$ 1,200,000	1500	\$ 250	\$ 375,000	\$ 1,575,000
11									
12									
13									
Subtotal								\$ 1,112,500	\$ 5,480,500
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 25,000	\$ 25,000	600	\$ 300	\$ 180,000	\$ 205,000
2	Hardware and Labor	1		\$ 75,000	\$ 75,000	600	\$ 300	\$ 180,000	\$ 255,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal								\$ 360,000	\$ 460,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 195,000	\$ 195,000	1000	\$ 275	\$ 275,000	\$ 470,000
2	Controls Conduit and Wiring	1		\$ 75,000	\$ 75,000	500	\$ 275	\$ 137,500	\$ 212,500
3				\$ -	\$ -			\$ -	\$ -
Subtotal								\$ 412,500	\$ 682,500
Subtotal					Multiplier		Multiplier	\$ 2,730,875	\$ 7,928,875
Contingency					10%		10%	\$ 273,088	\$ 792,888
Subtotal								\$ 3,003,963	\$ 8,721,763
Construction Management Overhead					10%		10%	\$ 300,396	\$ 872,176
Profit					10%		10%	\$ 300,396	\$ 872,176
Subtotal Construction								\$ 3,604,755	\$ 10,466,115
Tax					8.875%		0%	\$ -	\$ 608,946
Design Engineering								\$ 837,289	\$ 837,289
Construction Administration								\$ 418,645	\$ 418,645
Commissioning								\$ 313,983	\$ 313,983
Total Estimated Cost								\$ 12,644,978	

9.1.4 Operational and Economic Analysis

Table 66 through Table 72 summarize the energy and financial results for this option.

Table 66. M.1 Energy Model Tower A Heating Summary

Tower A Heating		Measure:	M.1 - Central Plant Heat Pump - Tower A												Date:	Client:										
Calculation Method:		8,760	Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input type="checkbox"/> Actual Dynamic <input checked="" type="checkbox"/> Manufacturer Dynamic						11/14/2022	Brookfield													
Design	Tag	Heat Pump Summary		Tower A LH HHW Gen			Tower A UH HHW Gen			Tower A LH Steam Utilization			Tower A UH Steam Utilization			Tower A LH HHW Utilization			Tower A UH HHW Utilization			Utility Rate Data				
																			Utility Blend Rate Winter Steam \$ 36.000 \$/klb Summer Steam \$ 18.000 \$/klb \$ - \$/klb Swr \$ - \$/hcf Wtr \$ - \$/hcf							
Existing Operation		LH HP	UH HP																							
		HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost
	Jan	0	0	0	0.00	364	1,713,087	1793.81	364	886,663	928.44	325	980,524	1026.73	0	0	0.00									
	Feb	0	0	0	0.00	336	1,382,642	1447.79	336	712,250	745.81	257	768,416	804.62	0	0	0.00									
	Mar	0	0	0	0.00	378	1,567,456	1641.32	378	832,618	871.85	287	863,537	904.23	0	0	0.00									
	Apr	0	0	0	0.00	350	535,729	560.97	350	322,580	337.78	75	183,567	192.22	0	0	0.00									
	May	0	0	0	0.00	378	361,045	378.06	378	295,133	309.04	31	58,630	61.39	0	0	0.00									
	Jun	0	0	0	0.00	364	260,244	272.51	364	260,244	272.51	0	0	0.00	0	0	0.00									
	July	0	0	0	0.00	364	259,176	271.39	364	259,176	271.39	0	0	0.00	0	0	0.00									
	Aug	0	0	0	0.00	378	270,037	282.76	378	270,037	282.76	0	0	0.00	0	0	0.00									
	Sep	0	0	0	0.00	364	268,701	281.36	364	262,525	274.89	2	4,679	4.90	0	0	0.00									
	Oct	0	0	0	0.00	364	498,042	521.51	364	322,360	337.55	66	155,501	162.83	0	0	0.00									
	Nov	0	0	0	0.00	364	1,203,098	1259.79	364	588,911	616.66	238	646,353	676.81	0	0	0.00									
Dec	0	0	0	0.00	364	1,859,547	1947.17	364	1,030,450	1079.00	337	1,054,141	1103.81	0	0	0.00										
Total		0	0	0	0.00	4,368	10,178,803	10,658	4,368	6,042,946	6,328	1,618	4,715,349	4,938	0	0	0	0	0	0	0	0	0	20,937,098	21,924	\$ 716,281
Total Cost	\$					\$			\$			\$			\$											\$
M.1 - Central Plant Heat Pump - Tower A																						Utility Rate Data				
Modified Operation	Tag	Heat Pump Summary		Tower A LH HHW Gen			Tower A UH HHW Gen			Tower A LH Steam Utilization			Tower A UH Steam Utilization			Tower A LH HHW Utilization			Tower A UH HHW Utilization			Utility Rate Data				
																			Utility Blend Rate Winter Steam \$ 36.000 \$/klb Summer Steam \$ 18.000 \$/klb 0 \$ - \$/klb Swr \$ - \$/hcf Wtr \$ - \$/hcf							
Modified Operation		LH HP	UH HP																							
		HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost
	Jan	0	887	0	0.00	364	1,713,087	1793.81	364	886,663	928.44	325	980,524	1026.73	0	0	0.00									
	Feb	0	855	0	0.00	336	1,382,642	1447.79	336	712,250	745.81	257	768,416	804.62	0	0	0.00									
	Mar	0	846	0	0.00	378	1,567,456	1641.32	378	832,618	871.85	287	863,537	904.23	0	0	0.00									
	Apr	0	372	0	0.00	350	535,729	560.97	350	322,580	337.78	75	183,567	192.22	0	0	0.00									
	May	0	249	0	0.00	378	361,045	378.06	378	295,133	309.04	31	58,630	61.39	0	0	0.00									
	Jun	0	14	0	0.00	364	260,244	272.51	364	260,244	272.51	0	0	0.00	0	0	0.00									
	July	0	13	0	0.00	364	259,176	271.39	364	259,176	271.39	0	0	0.00	0	0	0.00									
	Aug	0	13	0	0.00	378	270,037	282.76	378	270,037	282.76	0	0	0.00	0	0	0.00									
	Sep	0	161	0	0.00	364	268,701	281.36	364	262,525	274.89	2	4,679	4.90	0	0	0.00									
	Oct	0	304	0	0.00	364	498,042	521.51	364	322,360	337.55	66	155,501	162.83	0	0	0.00									
	Nov	0	846	0	0.00	364	1,203,098	1259.79	364	588,911	616.66	238	646,353	676.81	0	0	0.00									
Dec	0	884	0	0.00	364	1,859,547	1947.17	364	1,030,450	1079.00	337	1,054,141	1103.81	0	0	0.00										
Total		0	5,444	0	0.00	4,368	10,178,803	888.20	4,368	6,042,946	6,328	1,618	4,715,349	4,938	0	0	0	0	0	0	0	0	0	20,937,098	21,924	\$ 716,281
Total Cost	\$					\$			\$			\$			\$											\$

Table 70. M.1 Energy Model Winter Garden Heating Summary

WG Heating		Measure:	M.1 - Central Plant Heat Pump - Winter Garden													Date:	Client: Brookfield Community Heat Pump										
Calculation Method:		8,760	Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input type="checkbox"/> Actual Dynamic <input checked="" type="checkbox"/> Manufacturer Dynamic							Utility Rate Data														
Design	Tag	Heat Pump Summary				WG HHW Gen			WG HHW Gen			WG Steam Utilization (1 of 2)			WG Steam Utilization (2 of 2)			Utility Rate Data									
		LH HP	UH HP																Utility	Blend Rate							
																			Winter Steam	\$ 36.000	\$/klb						
																			Summer Steam	\$ 18.000	\$/klb						
																			Swr	\$ -	\$/hcf						
																			Wtr	\$ -	\$/hcf						
Existing Operation	HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost		
	Jan	0	0	0	0.00	720	206,511	216	0	0	0	720	977,095	1,023	0	0	0.00								1,183,607	1,239	\$ 44,618
	Feb	0	0	0	0.00	566	147,765	155	0	0	0	566	702,214	735	0	0	0.00								849,979	890	\$ 32,041
	Mar	0	0	0	0.00	632	174,967	183	0	0	0	632	829,174	868	0	0	0.00								1,004,141	1,051	\$ 37,852
	Apr	0	0	0	0.00	255	26,640	28	0	0	0	255	135,852	142	0	0	0.00								162,492	170	\$ 3,063
	May	0	0	0	0.00	166	11,813	12	0	0	0	166	63,442	66	0	0	0.00								75,256	79	\$ 1,418
	Jun	0	0	0	0.00	18	653	1	0	0	0	18	4,039	4	0	0	0.00								4,692	5	\$ 88
	Jul	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00								0	0	\$ -
	Aug	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00								0	0	\$ -
	Sep	0	0	0	0.00	26	1,083	1	0	0	0	26	6,466	7	0	0	0.00								7,549	8	\$ 142
	Oct	0	0	0	0.00	182	18,653	20	0	0	0	182	95,330	100	0	0	0.00								113,983	119	\$ 4,297
	Nov	0	0	0	0.00	568	112,299	118	0	0	0	568	542,006	568	0	0	0.00								654,305	685	\$ 24,665
	Dec	0	0	0	0.00	715	239,678	251	0	0	0	715	1,126,724	1,180	0	0	0.00								1,366,402	1,431	\$ 51,508
Total	0	0	0	0.00	3,848	940,062	984	0	0	0	3,848	4,482,343	4,694	0	0	0	0	0	0	0	0	0	0	0	5,422,405	5,678	\$ 199,693
Total Cost	\$				\$		32,621	\$			\$		165,014	\$													

M.1 - Central Plant Heat Pump - Winter Garden		M.1 - Central Plant Heat Pump - Winter Garden													Utility Rate Data										
Tag	Heat Pump Summary				WG HHW Gen			WG HHW Gen			WG Steam Utilization (1 of 2)			WG Steam Utilization (2 of 2)			Utility Rate Data								
	LH HP	UH HP																	Utility	Blend Rate					
																			Winter Steam	\$ 36.000	\$/klb				
																			Summer Steam	\$ 18.000	\$/klb				
																			0	\$ -	\$/klb				
																			Swr	\$ -	\$/hcf				
																			Wtr	\$ -	\$/hcf				
HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost	
Jan	1,183,607	187	0	0.00	720	0	0	0	0	0	720	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Feb	849,979	155	0	0.00	566	0	0	0	0	0	566	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Mar	1,004,141	167	0	0.00	632	0	0	0	0	0	632	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Apr	162,492	93	0	0.00	255	0	0	0	0	0	255	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
May	75,256	65	0	0.00	166	0	0	0	0	0	166	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Jun	4,692	24	0	0.00	18	0	0	0	0	0	18	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Jul	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Aug	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Sep	7,549	28	0	0.00	26	0	0	0	0	0	26	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Oct	113,983	73	0	0.00	182	0	0	0	0	0	182	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Nov	654,305	147	0	0.00	568	0	0	0	0	0	568	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Dec	1,366,402	187	0	0.00	715	0	0	0	0	0	715	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Total	5,422,405	187	0	0.00	3,848	0	0.00	0	0	0	3,848	0	0	0	0	0	0	0	0	0	0	0	0	\$ -	
Total Cost	\$				\$		-	\$			\$		-	\$											

Summary	Tag	Heat Pump Summary				WG HHW Gen			WG HHW Gen			WG Steam Utilization (1 of 2)			WG Steam Utilization (2 of 2)			Total Saving							
	Change Hrs					0			0		0			0					5,422,405	MBH					
	Energy Reduction					940,062			0		4,482,343			0					5,678	klbs					
	Reduction in Cost	\$				\$		32,621	\$		-	\$		165,014	\$				\$						199,693

Table 71. M.1 Energy Model Central Plant Heating Summary

CP Heating		Measure:	M.1 - Central Plant Heat Pump - Central Plant													Date:	Client: Brookfield Community Heat Pump											
Calculation Method:		8,760	Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input type="checkbox"/> Actual Dynamic <input checked="" type="checkbox"/> Manufacturer Dynamic							Utility Rate Data															
Design	Tag	Heat Pump Summary				CP HHW Gen			CP Steam Utilization (1 of 2)			CP Steam Utilization (2 of 2)			Utility Rate Data													
		LH HP	UH HP													Utility	Blend Rate											
															Winter Steam	\$ 36.000	\$/klb											
															Summer Steam	\$ 18.000	\$/klb											
																\$ -	\$/klb											
															Swr	\$ -	\$/hcf											
															Wtr	\$ -	\$/hcf											
Existing Operation		HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost		
	Jan	0	0	0	0.00	720	182,629	191	0	0	0	720	2,133,624	2,234	720	272,585	285.43									2,588,838	2,711	\$ 97,590
	Feb	0	0	0	0.00	566	130,677	137	0	0	0	566	1,526,671	1,599	566	195,043	204.23									1,852,390	1,940	\$ 69,828
	Mar	0	0	0	0.00	632	154,733	162	0	0	0	632	1,807,713	1,893	632	230,948	241.83									2,193,394	2,297	\$ 82,683
	Apr	0	0	0	0.00	255	23,559	25	0	0	0	255	275,241	288	255	35,164	36.82									333,964	350	\$ 6,295
	May	0	0	0	0.00	166	10,447	11	0	0	0	166	122,051	128	166	15,593	16.33									148,091	155	\$ 2,791
	Jun	0	0	0	0.00	18	577	1	0	0	0	18	6,743	7	18	862	0.90									8,182	9	\$ 154
	July	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00									0	0	\$ -
	Aug	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00									0	0	\$ -
	Sep	0	0	0	0.00	26	957	1	0	0	0	26	11,185	12	26	1,429	1.50									13,571	14	\$ 256
	Oct	0	0	0	0.00	182	16,496	17	0	0	0	182	192,718	202	182	24,621	25.78									233,835	245	\$ 8,815
	Nov	0	0	0	0.00	568	99,312	104	0	0	0	568	1,160,240	1,215	568	148,229	155.21									1,407,780	1,474	\$ 53,068
	Dec	0	0	0	0.00	715	211,960	222	0	0	0	715	2,476,291	2,593	715	316,363	331.27									3,004,614	3,146	\$ 113,263
Total	0	0	0	0.00	3,848	831,347	871	0	0	0	3,848	9,712,476	10,170	3,848	1,240,835	1,299	0	0	0	0	0	0	0	0	0	11,784,659	12,340	\$ 434,743
Total Cost	\$				\$		30,669	\$			\$		358,299	\$		44,670,076												

		M.1 - Central Plant Heat Pump - Central Plant													Utility Rate Data													
Modified Operation	Tag	Heat Pump Summary				CP HHW Gen			CP Steam Utilization (1 of 2)			CP Steam Utilization (2 of 2)			Utility Rate Data													
		LH HP	UH HP														Utility	Blend Rate										
																Winter Steam	\$ 36.000	\$/klb										
																Summer Steam	\$ 18.000	\$/klb										
																0	\$ -	\$/klb										
																Swr	\$ -	\$/hcf										
																Wtr	\$ -	\$/hcf										
Modified Operation		HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost		
	Jan	2,588,838	445	0	0.00	720	0	0	0	0	0	720	0	0	720	0	0.00									0	0	\$ -
	Feb	1,852,390	367	0	0.00	566	0	0	0	0	0	566	0	0	566	0	0.00									0	0	\$ -
	Mar	2,193,394	395	0	0.00	632	0	0	0	0	0	632	0	0	632	0	0.00									0	0	\$ -
	Apr	333,964	217	0	0.00	255	0	0	0	0	0	255	0	0	255	0	0.00									0	0	\$ -
	May	148,091	149	0	0.00	166	0	0	0	0	0	166	0	0	166	0	0.00									0	0	\$ -
	Jun	8,182	49	0	0.00	18	0	0	0	0	0	18	0	0	18	0	0.00									0	0	\$ -
	July	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00									0	0	\$ -
	Aug	0	0	0	0.00	0	0	0	0	0	0	0	0	0	0	0	0.00									0	0	\$ -
	Sep	13,571	60	0	0.00	26	0	0	0	0	0	26	0	0	26	0	0.00									0	0	\$ -
	Oct	233,835	169	0	0.00	182	0	0	0	0	0	182	0	0	182	0	0.00									0	0	\$ -
	Nov	1,407,780	347	0	0.00	568	0	0	0	0	0	568	0	0	568	0	0.00									0	0	\$ -
	Dec	3,004,614	445	0	0.00	715	0	0	0	0	0	715	0	0	715	0	0.00									0	0	\$ -
Total	11,784,659	445	0	0.00	3,848	0	0.00	0	0	0	3,848	0	0	3,848	0	0									0	0	\$ -	
Total Cost	\$				\$		-	\$			\$		-	\$		-										0	0	\$ -

Summary	Tag	Heat Pump Summary				CP HHW Gen			CP Steam Utilization (1 of 2)			CP Steam Utilization (2 of 2)			Total Saving													
	Change Hrs					0			0			0																
	Energy Reduction					831,347			0			9,712,476			1,240,835													
Reduction in Cost	\$				\$		30,669	\$			\$		358,299	\$		44,670,076	\$							\$		11,784,659	12,340	MBH klbs
																												434,743

Table 72. M.1 Energy Model Central Plant Chilled Water Summary

Chiller Plant Operational Data		Measure:	M.1 - Central Plant Heat Pump													Date: 11/14/2022	Client: Brookfield									
Calculation Method:		Baseline - Design, Trended			Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input checked="" type="checkbox"/> Actual Dynamic			Utility Rate Data															
Design	Tag	CHW Plant													Utility		Blend Rate									
Existing Operation		Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	1,447	107,333	0.09																	2,696	\$ 30,597	107,333	\$ 9,875	\$ 40,472
	Feb	360	1,394	680,413	0.49																	5,148	\$ 18,539	680,413	\$ 62,598	\$ 81,137
	Mar	727	1,655	1,538,161	0.94																	5,283	\$ 20,771	1,538,161	\$ 141,511	\$ 162,282
	Apr	720	1,588	1,506,940	0.96																	6,129	\$ 25,966	1,506,940	\$ 138,638	\$ 164,604
	May	744	2,000	1,918,768	0.98																	7,386	\$ 68,073	1,918,768	\$ 176,527	\$ 244,600
	Jun	720	2,803	2,414,972	0.91																	7,562	\$ 178,622	2,414,972	\$ 222,177	\$ 400,800
	July	744	3,121	2,520,502	0.86																	7,500	\$ 160,134	2,520,502	\$ 231,886	\$ 392,020
	Aug	744	3,081	2,537,487	0.88																	7,498	\$ 177,095	2,537,487	\$ 233,449	\$ 410,544
	Sep	720	2,478	2,126,600	0.93																	7,471	\$ 154,435	2,126,600	\$ 195,647	\$ 350,083
	Oct	744	1,717	1,564,803	1.01																	6,033	\$ 62,980	1,564,803	\$ 143,962	\$ 206,942
	Nov	720	1,279	1,355,734	1.10																	5,070	\$ 23,236	1,355,734	\$ 124,728	\$ 147,964
	Dec	728	999	923,479	0.99																	3,687	\$ 12,704	923,479	\$ 84,960	\$ 97,664
	Total	7,731	23,563	19,195,193	0.84	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	Peak 7,562	\$ 933,152	19,195,193	\$ 1,765,958	\$ 2,699,110	
	Total Cost	\$		1,765,958	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-						
M.1 - Central Plant Heat Pump																				Utility Rate Data						
Design	Tag	CHW Plant				CH-2 (Heat Pump)			CH-6 (Heat Pump)			HHW Pumps						Utility		Blend Rate						
Modified Operation		Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	97	37,121	0.15	744	1,010,579	1.5	357	548,893	1.5	744	30,809	0.03								5,136	\$ 58,291	1,627,403	\$ 149,721	\$ 208,012
	Feb	360	736	390,392	0.66	672	827,556	1.5	257	402,380	1.5	672	23,216	0.03								5,476	\$ 52,403	1,643,544	\$ 151,206	\$ 203,609
	Mar	727	1,609	786,774	1.64	744	916,770	1.5	300	469,785	1.5	744	26,971	0.03								5,614	\$ 52,428	2,200,300	\$ 202,428	\$ 254,855
	Apr	720	1,588	1,087,714	1.12	720	490,931	1.5	41	47,960	1.5	720	4,909	0.01								5,595	\$ 33,966	1,631,514	\$ 150,099	\$ 184,066
	May	744	2,000	1,419,329	0.88	744	423,659	1.5	13	15,559	1.5	744	3,113	0.01								6,942	\$ 66,850	1,861,660	\$ 171,273	\$ 238,123
	Jun	720	2,803	2,018,458	0.81	720	319,868	1.5	0	0	1.5	720	1,585	0.01								7,183	\$ 169,655	2,339,911	\$ 215,272	\$ 384,927
	July	744	3,121	2,230,763	0.80	744	317,937	1.5	0	0	1.5	744	1,493	0.01								7,191	\$ 151,165	2,550,193	\$ 234,618	\$ 385,782
	Aug	744	3,081	2,141,307	0.80	744	319,158	1.5	0	0	1.5	744	1,502	0.01								7,188	\$ 169,617	2,461,968	\$ 226,501	\$ 396,118
	Sep	720	2,478	1,828,305	0.87	720	326,472	1.5	0	0	1.5	720	1,663	0.01								6,371	\$ 150,479	2,156,441	\$ 198,393	\$ 348,871
	Oct	744	1,717	1,184,696	0.98	744	441,872	1.5	38	44,246	1.5	744	3,954	0.01								6,050	\$ 61,765	1,674,768	\$ 154,079	\$ 215,843
	Nov	720	1,279	635,472	1.83	720	784,344	1.5	210	301,940	1.5	720	16,976	0.02								4,655	\$ 44,628	1,738,733	\$ 159,963	\$ 204,592
	Dec	728	999	336,945	0.60	744	1,057,492	1.5	380	617,617	1.5	744	36,545	0.03								5,108	\$ 57,531	2,048,600	\$ 188,471	\$ 246,002
	Total	7,731	21,508	14,097,278	0.93	8,760	7,236,638	1.5	1,596	2,448,380	1.50	8,760	152,738	0.02	0	0	#DIV/0!	0	0	#DIV/0!	Peak 7,191	\$ 1,068,778	23,935,034	\$ 2,202,023	\$ 3,270,801	
	Total Cost	\$		1,296,950	\$	665,771	\$	225,251	\$	14,052	\$			\$												
Summary	Tag	CHW Plant				CH-2 (Heat Pump)			CH-6 (Heat Pump)			HHW Pumps			0			0								
	Change Hrs	0				-8,760			-1,596			-8,760			0			371								
	Energy Savings	5,097,915				-7,236,638			-2,448,380			-152,738			0			-4,739,842								
Reduction in Cost	\$ 469,008				\$ (665,771)			\$ (225,251)			\$ (14,052)			\$ -			\$ (571,691)									

9.1.5 Results

Table 73 summarizes the energy and financial results for this option.

Table 73. M.1 Energy Model Financial Summary

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.1	Central Heat Pump	113,650	-4,739,842	4,800	-1,370	3,430	\$ (35,657,579)	\$ -	\$ (35,657,579)	\$ 2,971,526	12.0	\$ 919,313	9.2	\$ 39,892,695
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

9.2 M.2, Tower A: Decentralized Heat Pump

This section outlines the proposed modification and system descriptions, controls strategy, analysis of the proposed modifications, and first cost analysis for the energy conservation measure for the Tower A decentralized heat pump.

9.2.1 Proposed Modification Description

The team proposes the following modifications to the current system:

- Install a heat pump at the building level.
- Modify the heating hot water distribution piping to accommodate the new system.
- Replace select non-100% outside air (OA) unit steam coils with heating hot water coils.
- Replace select 100% outside air unit steam coils with hot water coils.
- Replace steam-to-hot-water domestic hot water heaters with water-to-water domestic hot water heaters.
- Implement associated controls.

9.2.2 Proposed System Description

The team proposes the following system configuration and integration for Tower A:

- Install a heat pump in the building penthouse using the building's chilled water and condenser water as the heat source. The 215-ton heat pump will generate 140°F heating hot water supply temperature.
- Tap into the existing heating hot water distribution using existing heating hot water return piping to preheat the water before it enters the heat exchangers for the following units:
 - HX-1A
 - HX-2A
- Install duplex heating hot water-to-domestic hot water heat exchangers to preheat the domestic hot water before it enters the steam-to-hot water heat exchangers for the following units:
 - H-1
 - H-2
 - H-3
 - H-4
 - H-5
 - H-6
 - H-7

9.2.3 Proposed Controls Strategy

The team proposes the following controls strategy and adjustments to optimize the system's performance:

- **Tower A heat pump:** The proposed heat pump will be installed in the Tower A penthouse. The heat pump will operate to maintain the heating hot water distribution set-point, which will adjust based on the outdoor air temperature. All building equipment must coordinate with this set-point.
- **Heating hot water:** The heating hot water generated by the heat pump will blend with the existing heating hot water return to maintain the heating hot water set-point. If the plant-generated hot water cannot maintain the local heating hot water set-point, the existing steam-to-hot water heat exchanger will trim to maintain the local set-point.
- **Domestic hot water:** The heat pump-generated heating hot water will preheat the domestic hot water before it enters the existing steam-to-hot water heat exchangers. If the heat pump-generated hot water cannot maintain the local domestic hot water set-point, the existing steam-to-domestic hot water heat exchanger will trim to maintain the local set-point.

9.2.4 Analysis of Proposed Modification: Base Case and Energy Conservation Measure Analysis

The team proposes the following modeling process for this energy conservation measure:

- **Base case:**
 - Model the base case using an 8760 Excel model built solely for each heating load.
 - Implement the schedules and controls strategy for each unit.
- **Energy conservation measure analysis:**
 - Incorporate a switch into the base model to toggle the heat pump on and off, with an adjustable heating hot water supply temperature. This hourly supply temperature dictates how much heating can transfer from steam to the heat pump. The model calculates a heating hot water flow profile based on the calculated load and differential temperature.
 - Use this flow to calculate the new energy consumption of the heating hot water pump.
 - Use the heating load to calculate the heat pump power draw using factory equipment data. The existing plant chiller 8,760-kW/ton profile is used to calculate the reduction in power consumption for the existing plant.

9.2.5 First Cost Analysis

Smith Engineering produced an estimate for the first cost of this option, which totals \$3,741,971.

Table 74 details these costs.

Table 74. M.2 Energy Model Tower A Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
For					Client:	Brookfield			
M.2 - Tower A					Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 35,000	\$ 35,000	125	\$ 250	\$ 31,250	\$ 66,250
2	Superintended	1		\$ -	\$ -	250	\$ 225	\$ 56,250	\$ 56,250
3	Demolition and Protection	1		\$ 25,000	\$ 25,000	350	\$ 195	\$ 68,250	\$ 93,250
4	Rigging	1		\$ 150,000	\$ 150,000	500	\$ 225	\$ 112,500	\$ 262,500
5	Gen Conditions	1		\$ 35,000	\$ 35,000	250	\$ 195	\$ 48,750	\$ 83,750
Subtotal					\$ 245,000	1475		\$ 317,000	\$ 562,000
Division 23000 - Mechanical									
1	Heat Pump	1		\$ 322,500	\$ 322,500	325	\$ 250	\$ 81,250	\$ 403,750
2	Heat Exchanger	1		\$ 75,000	\$ 75,000	200	\$ 250	\$ 50,000	\$ 125,000
3	Plant Piping and Insulation	1		\$ 80,625	\$ 80,625	200	\$ 250	\$ 50,000	\$ 130,625
4	Piping into 100% OA CC	1		\$ 65,000	\$ 65,000	250	\$ 250	\$ 62,500	\$ 127,500
5	Tempered Loop HX-2A	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
6	DHW HXs	1		\$ 85,000	\$ 85,000	350	\$ 250	\$ 87,500	\$ 172,500
7									
8									
9									
10									
Subtotal					\$ 693,125	1625		\$ 406,250	\$ 1,099,375
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 20,000	\$ 20,000	400	\$ 300	\$ 120,000	\$ 140,000
2	Hardware and Labor	1		\$ 35,000	\$ 35,000	600	\$ 300	\$ 180,000	\$ 215,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 55,000	1000		\$ 300,000	\$ 355,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 80,625	\$ 80,625	400	\$ 275	\$ 110,000	\$ 190,625
2	Controls Conduit and Wiring	1		\$ 35,000	\$ 35,000	500	\$ 275	\$ 137,500	\$ 172,500
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 115,625	900		\$ 247,500	\$ 363,125
Subtotal			Multiplier		\$ 1,108,750		Multiplier	\$ 1,270,750	\$ 2,379,500
Contingency			10%		\$ 110,875		10%	\$ 127,075	\$ 237,950
Subtotal					\$ 1,219,625			\$ 1,397,825	\$ 2,617,450
Construction Management Overhead			10%		\$ 121,963		10%	\$ 139,783	\$ 261,745
Profit			10%		\$ 121,963		10%	\$ 139,783	\$ 261,745
Subtotal Construction					\$ 1,463,550			\$ 1,677,390	\$ 3,140,940
Tax			8.875%		\$ 129,890		0%	\$ -	\$ 129,890
Design Engineering								\$ 251,275	\$ 251,275
Construction Administration								\$ 125,638	\$ 125,638
Commissioning								\$ 94,228	\$ 94,228
Total Estimated Cost					\$			3,741,971	

Table 76. M.2 Energy Model Tower A Cooling Summary

Chiller Plant Operational Data		Measure:	M.2 - Tower A Heat Pump												Date:	Client:														
Calculation Method:		Baseline - Design, Trended			Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input checked="" type="checkbox"/> Actual Dynamic			Utility Rate Data						11/14/2022	Brookfield												
Design	Tag	CHW Plant													Utility		Blend Rate													
Existing Operation		Hours	Ton-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost					
	Jan	60	1,447	107,333	0.09																	2,696	\$ 30,597	107,333	\$ 9,875	\$ 40,472				
	Feb	360	1,394	680,413	0.49																	5,148	\$ 18,539	680,413	\$ 62,598	\$ 81,137				
	Mar	727	1,655	1,538,161	0.94																	5,283	\$ 20,771	1,538,161	\$ 141,511	\$ 162,282				
	Apr	720	1,588	1,506,940	0.96																	6,129	\$ 25,966	1,506,940	\$ 138,638	\$ 164,604				
	May	744	2,000	1,918,768	0.98																	7,386	\$ 68,073	1,918,768	\$ 176,527	\$ 244,600				
	Jun	720	2,803	2,414,972	0.91																	7,562	\$ 178,622	2,414,972	\$ 222,177	\$ 400,800				
	July	744	3,121	2,520,502	0.86																	7,500	\$ 160,134	2,520,502	\$ 231,886	\$ 392,020				
	Aug	744	3,081	2,537,487	0.88																	7,498	\$ 177,095	2,537,487	\$ 233,449	\$ 410,544				
	Sep	720	2,478	2,126,600	0.93																	7,471	\$ 154,435	2,126,600	\$ 195,647	\$ 350,083				
	Oct	744	1,717	1,564,803	1.01																	6,033	\$ 62,980	1,564,803	\$ 143,962	\$ 206,942				
	Nov	720	1,279	1,355,734	1.10																	5,070	\$ 23,236	1,355,734	\$ 124,728	\$ 147,964				
	Dec	728	999	923,479	0.99																	3,687	\$ 12,704	923,479	\$ 84,960	\$ 97,664				
Total		7,731	23,563	19,195,193	0.84	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0					
Total Cost		\$		1,765,958	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	Peak 7,562	\$ 933,152	19,195,193	\$ 1,765,958	\$ 2,699,110					
M.2 - Tower A Heat Pump														Utility Rate Data																
Design	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps						Utility		Blend Rate													
Modified Operation		Hours	Ton-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost		
	Jan	60	1,401	105,257	0.09	744	69,210	1.5	744	2,628	0.02																			
	Feb	360	1,355	673,943	0.50	672	58,774	1.5	672	2,183	0.0																			
	Mar	727	1,612	1,493,679	0.94	744	65,908	1.5	744	2,467	0.0																			
	Apr	720	1,570	1,492,467	0.97	720	28,002	1.5	720	353	0.0																			
	May	744	1,983	1,906,251	0.98	744	25,619	1.5	744	186	0.0																			
	Jun	720	2,788	2,415,337	0.91	720	22,591	1.5	720	99	0.0																			
	July	744	3,106	2,514,355	0.86	744	22,498	1.5	744	99	0.0																			
	Aug	744	3,066	2,530,251	0.88	744	23,441	1.5	744	102	0.0																			
	Sep	720	2,462	2,118,641	0.93	720	22,789	1.5	720	105	0.0																			
	Oct	744	1,698	1,559,742	1.02	744	27,983	1.5	744	319	0.0																			
	Nov	720	1,246	1,247,320	1.03	720	49,921	1.5	720	1,450	0.0																			
	Dec	728	950	903,061	1.04	744	76,864	1.5	744	3,246	0.0																			
Total		7,731	23,237	18,960,303	0.85	8,760	493,599	1.5	8,760	13,236	0.01	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	
Total Cost		\$		1,744,348	\$	45,411	\$	1,218	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	Peak 8,395	\$ 946,393	19,467,138	\$ 1,790,977	\$ 2,737,370					
Summary	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps			0			0			0												
	Change Hrs	0				-8,760			-8,760			0			0			0												
	Energy Savings	234,889				-493,599			-13,236			0			0			0												
Reduction in Cost		\$	21,610			\$	(45,411)			\$	(1,218)			\$	-			\$	-			\$	-			\$	(38,259)			

9.2.7 Results

Table 77 summarizes the energy and financial results for this option.

Table 77. M.2 Energy Model Financial Summary

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.2	Tower A Heat Pump	5,954	-271,945	251	-79	173	\$ (3,741,971)	\$ -	\$ (3,741,971)	\$ 118,944	31.5	\$ 46,334	22.6	\$ (532,686)
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

9.3 M.3, Tower B: Decentralized Heat Pump

This section outlines the proposed modification and system descriptions, controls strategy, analysis of the proposed modifications, and first cost analysis for the energy conservation measure for the Tower B decentralized heat pump.

9.3.1 Proposed Modification Description

The team proposes the following modifications to the current system:

- Install a heat pump at the building level
- Modify the heating hot water distribution piping
- Replace select non-100% outside air unit steam coils with heating hot water coils
- Replace select 100% outside air unit steam coils with hot water coils
- Replace steam-to-hot water domestic hot water heaters with water-to-water domestic hot water heaters
- Implement associated controls

9.3.2 Proposed System Description

The team proposes the following system configuration and integration for Tower B:

- Install a heat pump in the building penthouse, using the building's chilled water and condenser water as a heat source. The 250-ton heat pump will generate 130°F heating hot water supply temperature.
- Install duplex heating hot water-to-domestic hot water heat exchangers to preheat the domestic hot water before it enters the steam-to-hot water heat exchangers for the following units:
 - H-1
 - H-2
 - H-3
 - H-4
 - H-5
 - H-6
 - H-7

9.3.3 Proposed Controls Strategy

The team proposes the following controls strategy and adjustments to optimize the system's performance:

- **Tower B heat pump:** Install a new heat pump in the Tower B plant. The heat pump will operate to maintain the heating hot water distribution set-point, which will adjust based on the outdoor air temperature. All building equipment must coordinate with this set-point.
- **Domestic hot water:** Use the heat pump-generated heating hot water to preheat the domestic water before it enters the existing steam-to-hot water heat exchangers. If the

heat pump-generated hot water cannot maintain the local domestic hot water set-point, the existing steam-to-domestic hot water heat exchanger will trim to maintain the local set-point.

9.3.4 Analysis of Proposed Modification: Base Case and Energy Conservation Measures

The team proposes the following modeling process for this energy conservation measure:

- **Base case:**
 - Model the base case using an 8760 Excel model built solely for each heating load.
 - Implement the schedules and controls strategy for each unit.
- **Energy conservation measure analysis:**
 - Incorporate a switch into the base model to toggle the heat pump on and off, with an adjustable heating hot water supply temperature. This hourly supply temperature value dictates how much heating can transfer from steam to the heat pump. The model calculates a heating hot water flow profile based on the calculated load and differential temperature.
 - Use this flow to calculate the energy consumption of the new heating hot water pump.
 - Use this heating load to calculate the heat pump power draw using factory equipment data. The existing plant chiller 8,760-kW/ton profile is used to calculate the reduction in power consumption for the existing plant.

9.3.5 First Cost Analysis

Smith Engineering produced an estimate for the first cost of this option, which totals \$3,210,609.

Table 78 details these costs.

Table 78. M.3 Energy Model Tower B Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
For					Client:	Brookfield			
M.3 - Tower B					Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 35,000	\$ 35,000	125	\$ 250	\$ 31,250	\$ 66,250
2	Superintended	1		\$ -	\$ -	250	\$ 225	\$ 56,250	\$ 56,250
3	Demolition and Protection	1		\$ 25,000	\$ 25,000	350	\$ 195	\$ 68,250	\$ 93,250
4	Rigging	1		\$ 150,000	\$ 150,000	500	\$ 225	\$ 112,500	\$ 262,500
5	Gen Conditions	1		\$ 35,000	\$ 35,000	250	\$ 195	\$ 48,750	\$ 83,750
Subtotal					\$ 245,000	1475		\$ 317,000	\$ 562,000
Division 23000 - Mechanical									
1	DHW HXs	1		\$ 110,000	\$ 110,000	400	\$ 250	\$ 100,000	\$ 210,000
2	Heat Pump	1		\$ 375,000	\$ 375,000	325	\$ 250	\$ 81,250	\$ 456,250
3	Plant Piping and Insulation	1		\$ 27,500	\$ 27,500	200	\$ 250	\$ 50,000	\$ 77,500
4									
5									
6									
7									
8									
9									
Subtotal					\$ 512,500	925		\$ 231,250	\$ 743,750
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 25,000	\$ 25,000	400	\$ 300	\$ 120,000	\$ 145,000
2	Hardware and Labor	1		\$ 35,000	\$ 35,000	600	\$ 300	\$ 180,000	\$ 215,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 60,000	1000		\$ 300,000	\$ 360,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 93,750	\$ 93,750	400	\$ 275	\$ 110,000	\$ 203,750
2	Controls Conduit and Wiring	1		\$ 35,000	\$ 35,000	500	\$ 275	\$ 137,500	\$ 172,500
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 128,750	900		\$ 247,500	\$ 376,250
Subtotal					\$ 946,250		Multiplier	\$ 1,095,750	\$ 2,042,000
Contingency							10%	\$ 94,625	\$ 204,200
Subtotal					\$ 1,040,875			\$ 1,205,325	\$ 2,246,200
Construction Management Overhead							10%	\$ 104,088	\$ 224,620
Profit							10%	\$ 104,088	\$ 224,620
Subtotal Construction					\$ 1,249,050			\$ 1,446,390	\$ 2,695,440
Tax							8.875%	\$ 110,853	\$ 110,853
Design Engineering								\$ 215,635	\$ 215,635
Construction Administration								\$ 107,818	\$ 107,818
Commissioning								\$ 80,863	\$ 80,863
Total Estimated Cost					\$			3,210,609	

Table 80. M.3 Energy Model Tower B Cooling Summary

Chiller Plant Operational Data		Measure:	M.3 - Tower B Heat Pump													Date:	11/14/2022 Client: Brookfield									
Calculation Method:		Baseline - Design, Trended			Basis:			<input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency:			<input checked="" type="checkbox"/> Actual Dynamic			Utility Rate Data									
Design	Tag	CHW Plant																Utility		Blend Rate						
Existing Operation	Tag	Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	1,447	107,333	0.09																	2,696	\$ 30,597	107,333	\$ 9,875	\$ 40,472
	Feb	360	1,394	680,413	0.49																	5,148	\$ 18,539	680,413	\$ 62,598	\$ 81,137
	Mar	727	1,655	1,538,161	0.94																	5,283	\$ 20,771	1,538,161	\$ 141,511	\$ 162,282
	Apr	720	1,588	1,506,940	0.96																	6,129	\$ 25,966	1,506,940	\$ 138,638	\$ 164,604
	May	744	2,000	1,918,768	0.98																	7,386	\$ 68,073	1,918,768	\$ 176,527	\$ 244,600
	Jun	720	2,803	2,414,972	0.91																	7,562	\$ 178,622	2,414,972	\$ 222,177	\$ 400,800
	July	744	3,121	2,520,502	0.86																	7,500	\$ 160,134	2,520,502	\$ 231,886	\$ 392,020
	Aug	744	3,081	2,537,487	0.88																	7,498	\$ 177,095	2,537,487	\$ 233,449	\$ 410,544
	Sep	720	2,478	2,126,600	0.93																	7,471	\$ 154,435	2,126,600	\$ 195,647	\$ 350,083
	Oct	744	1,717	1,564,803	1.01																	6,033	\$ 62,980	1,564,803	\$ 143,962	\$ 206,942
	Nov	720	1,279	1,355,734	1.10																	5,070	\$ 23,236	1,355,734	\$ 124,728	\$ 147,964
	Dec	728	999	923,479	0.99																	3,687	\$ 12,704	923,479	\$ 84,960	\$ 97,664
Total	7,731	23,563	19,195,193	0.84	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	
Total Cost	\$	1,765,958	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	7,562	\$ 933,152	19,195,193	\$ 1,765,958	\$ 2,699,110	
M.3 - Tower B Heat Pump																				Utility Rate Data						
Design	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps									Utility		Blend Rate						
Modified Operation	Tag	Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	1,375	100,661	0.09	744	93,498	1.3	744	12,553	0.17											2,851	\$ 32,363	206,713	\$ 19,018	\$ 51,381
	Feb	360	1,337	664,365	0.50	672	74,575	1.3	672	9,850	0.1											5,148	\$ 19,842	748,790	\$ 68,889	\$ 88,730
	Mar	727	1,592	1,493,611	0.96	744	82,235	1.3	744	10,806	0.1											5,337	\$ 21,486	1,586,653	\$ 145,972	\$ 167,458
	Apr	720	1,538	1,473,655	0.98	720	65,991	1.3	720	1,209	0.0											6,129	\$ 26,365	1,540,854	\$ 141,759	\$ 168,124
	May	744	1,952	1,874,640	0.97	744	62,534	1.3	744	560	0.0											7,386	\$ 69,286	1,937,734	\$ 178,272	\$ 247,557
	Jun	720	2,759	2,379,775	0.90	720	58,127	1.3	720	368	0.0											7,628	\$ 179,923	2,438,270	\$ 224,321	\$ 404,243
	July	744	3,075	2,550,731	0.88	744	59,901	1.3	744	380	0.0											8,415	\$ 161,679	2,611,012	\$ 240,213	\$ 401,892
	Aug	744	3,035	2,510,284	0.89	744	60,054	1.3	744	380	0.0											7,604	\$ 179,597	2,570,719	\$ 236,506	\$ 416,103
	Sep	720	2,433	2,131,069	0.95	720	58,262	1.3	720	368	0.0											7,539	\$ 157,073	2,189,698	\$ 201,452	\$ 358,525
	Oct	744	1,667	1,542,332	1.04	744	65,261	1.3	744	808	0.0											6,033	\$ 63,622	1,608,401	\$ 147,973	\$ 211,595
	Nov	720	1,214	1,136,550	0.96	720	84,865	1.3	720	7,032	0.1											5,070	\$ 16,338	1,228,447	\$ 113,017	\$ 129,355
	Dec	728	936	898,040	1.13	744	83,031	1.3	744	14,645	0.2											3,800	\$ 14,984	995,716	\$ 91,606	\$ 106,590
Total	7,731	22,912	18,755,712	0.85	8,760	848,336	1.3	8,760	58,958	0.07	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	0	0	0	0	
Total Cost	\$	1,725,526	\$	78,047	\$	5,424	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	\$	-	8,415	\$ 942,558	19,663,006	\$ 1,808,997	\$ 2,751,554	
Summary	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps			0			0			0								
	Change Hrs	0				-8,760			-8,760			0			0			0								
	Energy Savings	439,480				-848,336			-58,958			0			0			0								
Reduction in Cost	\$	40,432				\$ (78,047)			\$ (5,424)			\$ -			\$ -			\$ -			\$ (52,444)					

9.3.7 Results

Table 81 summarizes the energy and financial results for this option.

Table 81. M.3 Energy Model Financial Summary

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.3	Tower B Heat Pump	11,324	-467,814	478	-135	343	\$ (3,210,609)	\$ -	\$ (3,210,609)	\$ 267,538	12.0	\$ 91,940	8.9	\$ 3,769,543
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

9.4 M.4, Tower C: Decentralized Heat Pump

This section outlines the proposed modification and system descriptions, controls strategy, analysis of the proposed modifications, and first cost analysis for the energy conservation measure for the Tower C decentralized heat pump.

9.4.1 Proposed Modification Description

The team proposes the following modifications to the current system:

- Install a heat pump at the building level
- Modify the heating hot water distribution piping
- Replace select non-100% outdoor air unit steam coils with heating hot water coils
- Replace select 100% outdoor air unit steam coils with hot water coils
- Replace steam-to-hot water domestic hot water heaters with water-to-water domestic hot water heaters
- Implement associated controls

9.4.2 Proposed System Description

The team proposes the following system configuration and integration for Tower C:

- Install a heat pump in the building penthouse using the building's chilled water as the heat source. The 210-ton heat pump will generate 130°F heating hot water supply temperature.
- Tap into the AC-52C chilled water coil and the secondary chilled water return to provide the chilled water supply to the heat pump.
- Tap the heating hot water distribution from the heat pump into the existing heating hot water return piping to preheat the heating hot water before it enters the heat exchangers for the following units:
 - HX-5C
 - HX-4C

9.4.3 Proposed Controls Strategy

The team proposes the following controls strategy and adjustments to optimize the system's performance:

- **Tower C heat pump:** Install a new heat pump in building C penthouse. The heat pump will operate to maintain the heating hot water distribution set-point, which will adjust based on the outdoor air temperature. All building equipment must coordinate with this set-point.
- **Air handling units:** The heating hot water will feed AC-52C to provide heating to the 100% outdoor air. If the mixed air temperature (MAT) exceeds the acceptable level for an individual floor, the compartment unit's chilled water will trim the mixed air temperature per floor.

- **Heating hot water:** The heat pump-generated heating hot water will blend with the heating hot water return to maintain the heating hot water set-point. If the heat pump-generated hot water is insufficient to maintain the local heating hot water set-point, the existing steam-to-hot water heat exchanger will trim to maintain the local set-point.

9.4.4 Analysis of Proposed Modification: Base Case and Energy Conservation Measure Analysis

The team proposes the following modeling process for this energy conservation measure:

- **Base case:**
 - Model the base case using an 8760 Excel model built solely for each heating load.
 - Implement the schedules and controls strategy for each unit.
- **Energy conservation measure analysis:**
 - Incorporate a switch into the base model to toggle the heat pump on and off, with an adjustable heating hot water supply temperature. This hourly supply temperature dictates how much heating can transfer from steam to the heat pump. The model calculates a heating hot water flow profile based on the calculated load and differential temperature.
 - Use this flow to calculate the energy consumption of the new heating hot water pump.
 - Use the heating load to calculate the heat pump power draw using factory equipment data. The existing plant chiller 8,760-kW/ton profile is used to calculate the reduction in power consumption for the existing plant.

9.4.5 First Cost Analysis

Smith Engineering produced an estimate for the first cost of this option, which totals \$3,210,609.

Table 82 details these costs.

Table 82. M.4 Energy Model Tower C Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
For					Client:	Brookfield			
M.4 - Tower C					Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate			<input type="checkbox"/> No Design	<input checked="" type="checkbox"/> Conceptual Design	<input type="checkbox"/> Final Design	<input type="checkbox"/> Actual Cost			
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 35,000	\$ 35,000	125	\$ 250	\$ 31,250	\$ 66,250
2	Superintended	1		\$ -	\$ -	250	\$ 225	\$ 56,250	\$ 56,250
3	Demolition and Protection	1		\$ 25,000	\$ 25,000	350	\$ 195	\$ 68,250	\$ 93,250
4	Rigging	1		\$ 150,000	\$ 150,000	500	\$ 225	\$ 112,500	\$ 262,500
5	Gen Conditions	1		\$ 35,000	\$ 35,000	250	\$ 195	\$ 48,750	\$ 83,750
Subtotal					\$ 245,000	1475		\$ 317,000	\$ 562,000
Division 23000 - Mechanical									
1	Heat Pump	1		\$ 375,000	\$ 375,000	325	\$ 250	\$ 81,250	\$ 456,250
2	Plant Piping and Insulation	1		\$ 93,750	\$ 93,750	200	\$ 250	\$ 50,000	\$ 143,750
3	Tempered Loop HX-5C	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
4	Piping into 100% OA CC	1		\$ 65,000	\$ 65,000	250	\$ 250	\$ 62,500	\$ 127,500
5									
6									
7									
8									
9									
10									
Subtotal					\$ 598,750	1075		\$ 268,750	\$ 867,500
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 20,000	\$ 20,000	400	\$ 300	\$ 120,000	\$ 140,000
2	Hardware and Labor	1		\$ 35,000	\$ 35,000	600	\$ 300	\$ 180,000	\$ 215,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 55,000	1000		\$ 300,000	\$ 355,000
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 93,750	\$ 93,750	400	\$ 275	\$ 110,000	\$ 203,750
2	Controls Conduit and Wiring	1		\$ 35,000	\$ 35,000	500	\$ 275	\$ 137,500	\$ 172,500
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 128,750	900		\$ 247,500	\$ 376,250
Subtotal					\$ 1,027,500		Multiplier	\$ 1,133,250	\$ 2,160,750
Contingency							10%	\$ 113,325	\$ 216,075
Subtotal					\$ 1,130,250			\$ 1,246,575	\$ 2,376,825
Construction Management Overhead							10%	\$ 124,658	\$ 237,683
Profit							10%	\$ 124,658	\$ 237,683
Subtotal Construction					\$ 1,356,300			\$ 1,495,890	\$ 2,852,190
Tax							8.875%	\$ 120,372	\$ 120,372
Design Engineering								\$ 228,175	\$ 228,175
Construction Administration								\$ 114,088	\$ 114,088
Commissioning								\$ 85,566	\$ 85,566
Total Estimated Cost					\$			\$ 3,400,390	\$ 3,400,390

9.4.6 Operational and Economic Analysis

Table 83 summarizes the energy and financial results for this option.

Table 83. M.4 Energy Model Tower C Heating Summary

Chiller Plant Operational Data										Measure:	M.4 - Tower C Heat Pump										Date: 11/14/2022		Client: Brookfield					
Calculation Method:				Baseline - Design, Trended						Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input checked="" type="checkbox"/> Actual Dynamic				Utility Rate Data											
Design	Tag	CHW Plant																Utility		Blend Rate								
Existing Operation	Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	1,447	107,333	0.09																			2,696	\$ 30,597	107,333	\$ 9,875	\$ 40,472
	Feb	360	1,394	680,413	0.49																			5,148	\$ 18,539	680,413	\$ 62,598	\$ 81,137
	Mar	727	1,655	1,538,161	0.94																			5,283	\$ 20,771	1,538,161	\$ 141,511	\$ 162,282
	Apr	720	1,588	1,506,940	0.96																			6,129	\$ 25,966	1,506,940	\$ 138,638	\$ 164,604
	May	744	2,000	1,918,768	0.98																			7,386	\$ 68,073	1,918,768	\$ 176,527	\$ 244,600
	Jun	720	2,803	2,414,972	0.91																			7,562	\$ 178,622	2,414,972	\$ 222,177	\$ 400,800
	July	744	3,121	2,520,502	0.86																			7,500	\$ 160,134	2,520,502	\$ 231,886	\$ 392,020
	Aug	744	3,081	2,537,487	0.88																			7,498	\$ 177,095	2,537,487	\$ 233,449	\$ 410,544
	Sep	720	2,478	2,126,600	0.93																			7,471	\$ 154,435	2,126,600	\$ 195,647	\$ 350,083
	Oct	744	1,717	1,564,803	1.01																			6,033	\$ 62,980	1,564,803	\$ 143,962	\$ 206,942
	Nov	720	1,279	1,355,734	1.10																			5,070	\$ 23,236	1,355,734	\$ 124,728	\$ 147,964
	Dec	728	999	923,479	0.99																			3,687	\$ 12,704	923,479	\$ 84,960	\$ 97,664
Total	7,731	23,563	19,195,193	0.84	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0
Total Cost	\$			1,765,958	\$			\$			\$			\$			\$			\$			7,562	\$ 933,152	19,195,193	\$ 1,765,958	\$ 2,699,110	
M.4 - Tower C Heat Pump																				Utility Rate Data								
Design	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps									Utility		Blend Rate								
Modified Operation	Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost	
	Jan	60	1,436	106,544	0.09	744	15,143	1.3	744	4,860	0.04													2,696	\$ 30,597	126,547	\$ 11,642	\$ 42,240
	Feb	360	1,392	679,899	0.49	672	2,911	1.3	672	3,822	0.1													5,154	\$ 18,539	686,632	\$ 63,170	\$ 81,709
	Mar	727	1,649	1,535,557	0.95	744	8,700	1.3	744	4,186	0.1													5,283	\$ 20,771	1,548,442	\$ 142,457	\$ 163,228
	Apr	720	1,587	1,505,712	0.96	720	1,845	1.3	720	1,868	0.2													6,129	\$ 25,966	1,509,425	\$ 138,867	\$ 164,833
	May	744	1,999	1,915,682	0.97	744	1,643	1.3	744	1,166	0.1													7,386	\$ 68,073	1,918,491	\$ 176,501	\$ 244,574
	Jun	720	2,803	2,414,910	0.91	720	430	1.3	720	148	0.0													7,562	\$ 178,622	2,415,487	\$ 222,225	\$ 400,847
	July	744	3,121	2,520,497	0.86	744	8	1.3	744	1	0.0													7,500	\$ 160,134	2,520,506	\$ 231,887	\$ 392,021
	Aug	744	3,081	2,537,271	0.88	744	52	1.3	744	4	0.0													7,498	\$ 177,095	2,537,328	\$ 233,434	\$ 410,529
	Sep	720	2,477	2,126,269	0.93	720	543	1.3	720	202	0.0													7,471	\$ 154,435	2,127,014	\$ 195,685	\$ 350,121
	Oct	744	1,716	1,564,566	1.01	744	861	1.3	744	1,205	0.1													6,033	\$ 62,980	1,566,632	\$ 144,130	\$ 207,110
	Nov	720	1,277	1,354,440	1.10	720	1,955	1.3	720	3,738	0.1													5,070	\$ 23,592	1,360,133	\$ 125,132	\$ 148,724
	Dec	728	985	919,610	1.00	744	18,634	1.3	744	4,851	0.1													3,687	\$ 13,362	943,096	\$ 86,765	\$ 100,127
Total	7,731	23,524	19,180,956	0.85	8,760	52,725	1.3	8,760	26,051	0.07	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	7,562	\$ 934,167	19,259,733	\$ 1,771,895	\$ 2,706,062	
Total Cost	\$			1,764,648	\$		4,851	\$		2,397	\$		-	\$		-	\$			\$			7,562	\$ 934,167	19,259,733	\$ 1,771,895	\$ 2,706,062	
Summary	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps																			
	Change Hrs	0				-8,760			-8,760			0			0			0		0								
	Energy Savings	14,237				-52,725			-26,051			0			0			0		0								
Reduction in Cost	\$	1,310				\$ (4,851)			\$ (2,397)			\$ -			\$ -			\$ -		\$ (6,952)								

Table 84. M.4 Energy Model Tower C Cooling Summary

Tower C Heating		Measure:	M.4 - Tower C Heat Pump													Date: 11/14/2022	Client: Brookfield Community Heat Pump												
Calculation Method:		8,760	Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input type="checkbox"/> Actual Dynamic <input checked="" type="checkbox"/> Manufacturer Dynamic								Utility Rate Data															
Design	Tag	Heat Pump Summary				Tower C LH HHW Gen			Tower C UH HHW Gen			Tower C LH Steam Utilization			Tower C UH Steam Utilization			Tower C LH HHW Utilization			Tower C UH HHW Utilization			Utility	Blend Rate				
																								Winter Steam	\$ 34.000	\$/klb			
																							Summer Steam	\$ 13.880	\$/klb				
																							Swr	\$ -	\$/hcf				
																							Wtr	\$ -	\$/hcf				
Existing Operation	HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost	
	Jan	0	0	0	0.00	731	3,355,515	3,514	731	881,465	923	459	1,450,658	1,519	0	0	0.00										5,687,638	5,956	\$ 202,492
	Feb	0	0	0	0.00	610	2,720,925	2,849	610	599,259	627	373	1,110,049	1,162	0	0	0.00										4,430,233	4,639	\$ 157,726
	Mar	0	0	0	0.00	680	3,100,273	3,246	680	736,317	771	426	1,280,651	1,341	0	0	0.00										5,117,242	5,358	\$ 182,185
	Apr	0	0	0	0.00	540	1,845,934	1,933	422	160,950	169	282	205,220	215	0	0	0.00										2,212,104	2,316	\$ 32,151
	May	0	0	0	0.00	558	1,735,615	1,817	313	91,888	96	276	96,767	101	0	0	0.00										1,924,270	2,015	\$ 27,967
	Jun	0	0	0	0.00	540	1,494,308	1,565	59	11,796	12	79	7,684	8	0	0	0.00										1,513,788	1,585	\$ 22,001
	July	0	0	0	0.00	558	1,512,764	1,584	2	141	0	9	0	0	0	0	0.00										1,512,905	1,584	\$ 21,989
	Aug	0	0	0	0.00	558	1,518,322	1,590	10	937	1	46	0	0	0	0	0.00										1,519,259	1,591	\$ 22,081
	Sep	0	0	0	0.00	540	1,507,634	1,579	84	16,211	17	113	13,527	14	0	0	0.00										1,537,373	1,610	\$ 22,344
	Oct	0	0	0	0.00	558	1,771,157	1,855	278	107,822	113	223	167,374	175	0	0	0.00										2,046,352	2,143	\$ 72,854
	Nov	0	0	0	0.00	640	2,575,590	2,697	640	484,134	507	401	845,625	885	0	0	0.00										3,905,349	4,089	\$ 139,039
	Dec	0	0	0	0.00	737	3,600,089	3,770	737	1,036,590	1,085	460	1,658,109	1,736	0	0	0.00										6,294,788	6,591	\$ 224,108
Total	0	0	0	0.00	7,250	26,738,126	27,998	4,566	4,127,510	4,322	3,147	6,835,664	7,158	0	0	0	0	0	0	0	0	0	0	0	0	0	37,701,301	39,478	\$ 1,126,936
Total Cost	\$					\$	749,373	\$		141,008	\$		236,555	\$		-													
M.4 - Tower C Heat Pump																							Utility Rate Data						
Tag	Heat Pump Summary				Tower C LH HHW Gen			Tower C UH HHW Gen			Tower C LH Steam Utilization			Tower C UH Steam Utilization			Tower C LH HHW Utilization			Tower C UH HHW Utilization			Utility	Blend Rate					
																							Winter Steam	\$ 34.000	\$/klb				
																							Summer Steam	\$ 13.880	\$/klb				
																							0	\$ -	\$/klb				
																							Swr	\$ -	\$/hcf				
																							Wtr	\$ -	\$/hcf				
Modified Operation	HP MBH	Max HP HHW Flow	HP MBH	Max HP HHW Flow	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Hours	MBH	klbs	Total MBH	Total klbs	Sum Cost	
	Jan	0	0	172,915	439.07	731	3,108,815	3,255	731	526,507	551	459	1,450,658	1,519	0	0	0.00										5,085,980	5,326	\$ 181,072
	Feb	0	0	37,761	393.44	610	2,542,216	2,662	610	382,903	401	373	1,110,049	1,162	0	0	0.00										4,035,169	4,225	\$ 143,660
	Mar	0	0	110,289	410.70	680	2,888,614	3,025	680	448,055	469	426	1,280,651	1,341	0	0	0.00										4,617,320	4,835	\$ 164,386
	Apr	0	0	23,473	325.00	540	1,800,064	1,885	422	87,785	92	282	205,220	215	0	0	0.00										2,093,069	2,192	\$ 30,421
	May	0	0	20,941	325.00	558	1,709,860	1,790	313	43,046	45	276	96,767	101	0	0	0.00										1,849,673	1,937	\$ 26,883
	Jun	0	0	5,479	325.00	540	1,491,075	1,561	59	2,815	3	79	7,684	8	0	0	0.00										1,501,574	1,572	\$ 21,824
	July	0	0	100	19.07	558	1,512,727	1,584	2	0	0	9	0	0	0	0	0.00										1,512,727	1,584	\$ 21,986
	Aug	0	0	664	61.14	558	1,518,071	1,590	10	0	0	46	0	0	0	0	0.00										1,518,071	1,590	\$ 22,064
	Sep	0	0	6,960	325.00	540	1,503,182	1,574	84	4,428	5	113	13,527	14	0	0	0.00										1,521,137	1,593	\$ 22,108
	Oct	0	0	11,351	325.00	558	1,740,459	1,822	278	63,216	66	223	167,374	175	0	0	0.00										1,971,049	2,064	\$ 70,173
	Nov	0	0	24,799	381.04	640	2,432,684	2,547	640	310,764	325	401	845,625	885	0	0	0.00										3,589,073	3,758	\$ 127,779
	Dec	0	0	270,552	439.07	737	3,315,925	3,472	737	591,874	620	460	1,658,109	1,736	0	0	0.00										5,565,908	5,828	\$ 198,158
Total	0	0	685,284	314.04	7,250	25,563,692	2230.69	4,566	2,461,392	2,577	3,147	6,835,664	7,158	0	0	0										34,860,748	36,503	\$ 1,030,514	
Total Cost	\$					\$	709,237	\$		84,722	\$		236,555	\$		-													
Summary	Tag	Heat Pump Summary				Tower C LH HHW Gen			Tower C UH HHW Gen			Tower C LH Steam Utilization			Tower C UH Steam Utilization			Tower C LH HHW Utilization			Tower C UH HHW Utilization			Total Saving					
	Change Hrs					0			0			0			0												2,840,552	MBH	
	Energy Reduction					1,174,434			1,666,118			0			0												2,974	klbs	
Reduction in Cost	\$					\$	40,135	\$		56,287	\$		-	\$		-										\$		96,422	

9.4.7 Results

Table 85 summarizes the energy and financial results for this option.

Table 85. M.4 Energy Model Financial Summary

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.4	Tower C Heat Pump	2,974	-64,663	126	-19	107	\$ (3,400,390)	\$ -	\$ (3,400,390)	\$ 89,458	38.0	\$ 28,659	28.7	-1,106,849.5
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

9.5 M.5, Tower D: Decentralized Heat Pump

This section outlines the proposed modification and system descriptions, controls strategy, analysis of the proposed modifications, and first cost analysis for the energy conservation measure for the Tower D decentralized heat pump.

9.5.1 Proposed Modification Description

The team proposes the following modifications to the current system:

- Installing a heat pump at the building level
- Modifying the heating hot water distribution piping
- Replacing select non-100% outdoor air unit steam coils with heating hot water coils
- Replacing select 100% outdoor air unit steam coils with hot water coils
- Replacing steam-to-hot water domestic hot water heaters with water-to-water domestic hot water heaters
- Implementing associated controls.

9.5.2 Proposed System Description

The team proposes the following system configuration and integration for Tower D:

- Install a heat pump in the building penthouse using the building's chilled water as the heat source. The 275-ton heat pump will generate 130°F heating hot water supply temperature.
- Tap into the AC-35D chilled water coil and the secondary chilled water return to provide the chilled water supply to the heat pump.
- Tap the heating hot water distribution from the heat pump into the existing heating hot water return piping to preheat the heating hot water before it enters the heat exchangers for the following units:
 - HX-5D
 - HX-1D

9.5.3 Proposed Controls Strategy

The team proposes the following controls strategy and adjustments to optimize the system's performance:

- **Tower D Plant heat pump:** Install a new heat pump in building D plant. The heat pump will operate to maintain the heating hot water distribution set-point, which will adjust based on the outdoor air temperature. All building equipment must coordinate with this set-point.
- **Air handling units:** The heating hot water will feed AC-35D to provide heating to the 100% outdoor air. If the mixed air temperature exceeds the acceptable level for an individual floor. The compartment unit's chilled water will trim the mixed air temperature per floor.
- **Heating hot water:** The heat pump-generated heating hot water will blend with the heating hot water return to maintain the heating hot water set-point. If the heat pump-generated hot water is

insufficient to maintain the local heating hot water set-point, the existing steam-to-hot water heat exchanger will trim to maintain the local set-point.

9.5.4 Analysis of Proposed Modification: Base Case and Energy Conservation Measures

The team proposes the following modeling process for this energy conservation measure.

- **Base case:**
 - Model the base case using an 8760 Excel model built solely for each heating load.
 - Implement the schedules and controls strategy for each unit.
- **Energy conservation measure analysis:**
 - Incorporate a switch into the base model to toggle the heat pump on and off, with an adjustable heating hot water supply temperature. This hourly supply temperature dictates how much heating can transfer from steam to the heat pump. The model calculates a heating hot water flow profile based on the calculated load and differential temperature.
 - Use this flow to calculate the energy conservation of the new heating hot water pump.
 - Use this heating load to calculate the heat pump power draw using factory equipment data. The existing plant chiller 8,760-kW/ton profile is used to calculate the reduction in power consumption for the existing plant.

9.5.5 First Cost Analysis

Smith Engineering produced an estimate for the first cost of this option, which totals \$3,158,404.

Table 86 details these costs.

Table 86. M.5 Energy Model Tower D Estimated Heat Pump Cost

Opinion of Probable Construction Cost					Date:	Wednesday, March 1, 2023			
For M.5 - Tower D					Client:	Brookfield			
					Project:	BFPL Community Heat Pump Stg A			
Basis of Estimate		<input type="checkbox"/> No Design		<input checked="" type="checkbox"/> Conceptual Design		<input type="checkbox"/> Final Design		<input type="checkbox"/> Actual Cost	
Item #	Description	Quantity	Units	Material Cost per Unit	Total Material Cost	Labor Hour	Labor Cost per Hour	Total Labor Cost	Total Cost
Division 01000 - General									
1	Permits and Filing	1		\$ 35,000	\$ 35,000	125	\$ 250	\$ 31,250	\$ 66,250
2	Superintended	1		\$ -	\$ -	250	\$ 225	\$ 56,250	\$ 56,250
3	Demolition and Protection	1		\$ 25,000	\$ 25,000	350	\$ 195	\$ 68,250	\$ 93,250
4	Rigging	1		\$ 150,000	\$ 150,000	500	\$ 225	\$ 112,500	\$ 262,500
5	Gen Conditions	1		\$ 35,000	\$ 35,000	250	\$ 195	\$ 48,750	\$ 83,750
Subtotal					\$ 245,000	1475		\$ 317,000	\$ 562,000
Division 23000 - Mechanical									
1	Heat Pump	1		\$ 412,500	\$ 412,500	325	\$ 250	\$ 81,250	\$ 493,750
2	Plant Piping and Insulation	1		\$ 103,125	\$ 103,125	200	\$ 250	\$ 50,000	\$ 153,125
3	Piping into 100% OA CC	1		\$ 65,000	\$ 65,000	250	\$ 250	\$ 62,500	\$ 127,500
4	Tempered Loop HX-1D	1		\$ 65,000	\$ 65,000	300	\$ 250	\$ 75,000	\$ 140,000
5									
6									
7									
8									
9									
Subtotal					\$ 645,625	1075		\$ 268,750	\$ 914,375
Division 25000 - Controls									
1	Schneider Software Prog and Graphics	1		\$ 20,000	\$ 20,000	275	\$ 300	\$ 82,500	\$ 102,500
2	Hardware and Labor	1		\$ 35,000	\$ 35,000	250	\$ 300	\$ 75,000	\$ 110,000
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 55,000	525		\$ 157,500	\$ 212,500
Division 26000 - Electrical									
1	Power Conduit and Wiring	1		\$ 95,000	\$ 95,000	400	\$ 275	\$ 110,000	\$ 205,000
2	Controls Conduit and Wiring	1		\$ 35,000	\$ 35,000	250	\$ 275	\$ 68,750	\$ 103,750
3				\$ -	\$ -			\$ -	\$ -
Subtotal					\$ 130,000	650		\$ 178,750	\$ 308,750
Subtotal					\$1,075,625		Multiplier	\$ 922,000	\$ 1,997,625
Contingency					10%		10%	\$ 92,200	\$ 199,763
Subtotal					\$1,183,188			\$1,014,200	\$ 2,197,388
Construction Management Overhead					10%		10%	\$ 101,420	\$ 219,739
Profit					10%		10%	\$ 101,420	\$ 219,739
Subtotal Construction					\$1,419,825			\$ 1,217,040	\$ 2,636,865
Tax					8.875%		0%	\$ -	\$ 126,009
Design Engineering								\$ 210,949	\$ 210,949
Construction Administration								\$ 105,475	\$ 105,475
Commissioning								\$ 79,106	\$ 79,106
Total Estimated Cost					\$			3,158,404	

Table 88. M.5 Energy Model Tower D Cooling Summary

Chiller Plant Operational Data				Measure:	M.5 - Tower D Heat Pump											Date: 11/14/2022	Client: Brookfield										
Calculation Method:				Baseline - Design, Trended			Basis: <input type="checkbox"/> Installed <input checked="" type="checkbox"/> Conceptual Design <input type="checkbox"/> Final Design			Equipment Efficiency: <input checked="" type="checkbox"/> Actual Dynamic					Utility Rate Data												
Design	Tag	CHW Plant													Utility		Blend Rate										
Existing Operation		Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost		
	Jan	60	1,447	107,333	0.09																	2,696	\$ 30,597	107,333	\$ 9,875	\$ 40,472	
	Feb	360	1,394	680,413	0.49																	5,148	\$ 18,539	680,413	\$ 62,598	\$ 81,137	
	Mar	727	1,655	1,538,161	0.94																	5,283	\$ 20,771	1,538,161	\$ 141,511	\$ 162,282	
	Apr	720	1,588	1,506,940	0.96																	6,129	\$ 25,966	1,506,940	\$ 138,638	\$ 164,604	
	May	744	2,000	1,918,768	0.98																	7,386	\$ 68,073	1,918,768	\$ 176,527	\$ 244,600	
	Jun	720	2,803	2,414,972	0.91																	7,562	\$ 178,622	2,414,972	\$ 222,177	\$ 400,800	
	July	744	3,121	2,520,502	0.86																	7,500	\$ 160,134	2,520,502	\$ 231,886	\$ 392,020	
	Aug	744	3,081	2,537,487	0.88																	7,498	\$ 177,095	2,537,487	\$ 233,449	\$ 410,544	
	Sep	720	2,478	2,126,600	0.93																	7,471	\$ 154,435	2,126,600	\$ 195,647	\$ 350,083	
	Oct	744	1,717	1,564,803	1.01																	6,033	\$ 62,980	1,564,803	\$ 143,962	\$ 206,942	
	Nov	720	1,279	1,355,734	1.10																	5,070	\$ 23,236	1,355,734	\$ 124,728	\$ 147,964	
	Dec	728	999	923,479	0.99																	3,687	\$ 12,704	923,479	\$ 84,960	\$ 97,664	
Total	7,731	23,563	19,195,193	0.84	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	0	Peak 7,562	\$ 933,152	19,195,193	\$ 1,765,958	\$ 2,699,110		
Total Cost	\$			1,765,958	\$			\$			\$			\$													
M.5 - Tower D Heat Pump																	Utility Rate Data										
Design	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps									Utility		Blend Rate							
Modified Operation		Hours	kTon-Hrs	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Hours	kWh	kW/Ton	Plant Peak Demand	Demand Cost	Plant kWh	Energy (kWh) Cost	Total Cost		
	Jan	60	1,415	106,900	0.09	744	22,374	1.3	744	6,672	0.03											2,696	\$ 30,597	135,947	\$ 12,507	\$ 43,104	
	Feb	360	1,376	675,053	0.49	672	16,174	1.3	672	5,320	0.0											5,175	\$ 18,539	696,548	\$ 64,082	\$ 82,621	
	Mar	727	1,619	1,510,757	0.95	744	28,407	1.3	744	6,077	0.0											5,378	\$ 20,932	1,545,241	\$ 142,162	\$ 163,094	
	Apr	720	1,569	1,489,352	0.97	720	24,712	1.3	720	3,057	0.1											6,199	\$ 25,966	1,517,121	\$ 139,575	\$ 165,541	
	May	744	1,982	1,911,740	0.98	744	22,632	1.3	744	2,144	0.1											7,386	\$ 68,860	1,936,516	\$ 178,159	\$ 247,019	
	Jun	720	2,799	2,411,387	0.91	720	5,450	1.3	720	356	0.0											7,562	\$ 178,622	2,417,193	\$ 222,382	\$ 401,004	
	July	744	3,121	2,520,410	0.86	744	183	1.3	744	7	0.0											7,500	\$ 160,134	2,520,600	\$ 231,895	\$ 392,029	
	Aug	744	3,081	2,537,483	0.88	744	933	1.3	744	40	0.0											7,498	\$ 177,095	2,538,456	\$ 233,538	\$ 410,633	
	Sep	720	2,472	2,123,419	0.93	720	7,484	1.3	720	490	0.0											7,471	\$ 154,435	2,131,393	\$ 196,088	\$ 350,524	
	Oct	744	1,706	1,556,887	1.02	744	14,312	1.3	744	2,034	0.0											6,033	\$ 62,980	1,573,234	\$ 144,738	\$ 207,718	
	Nov	720	1,263	1,329,389	1.10	720	17,558	1.3	720	5,407	0.1											5,070	\$ 23,941	1,352,355	\$ 124,417	\$ 148,357	
	Dec	728	951	892,436	1.00	744	35,471	1.3	744	6,849	0.0											3,687	\$ 14,796	934,757	\$ 85,998	\$ 100,794	
Total	7,731	23,356	19,065,215	0.85	8,760	195,691	1.3	8,760	38,454	0.03	0	0	#DIV/0!	0	0	#DIV/0!	0	0	#DIV/0!	0	0	Peak 7,562	\$ 936,897	19,299,361	\$ 1,775,541	\$ 2,712,438	
Total Cost	\$			1,754,000	\$		18,004	\$		3,538	\$			\$													
Summary	Tag	CHW Plant				CH-1 (Heat Pump)			HHW Pumps																		
	Change Hrs	0				-8,760			-8,760			0			0			0		0							
	Energy Savings	129,977				-195,691			-38,454			0			0			0		0							
Reduction in Cost	\$	11,958				\$	(18,004)			\$	(3,538)			\$	-			\$	-			\$	-		\$	(13,328)	

9.5.7 Results

Table 89 summarizes the energy and financial results for this option.

Table 89. M.5 Energy Model Financial Summary

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.5	Tower D Heat Pump	4,578	-104,168	193	-30	163	\$ (3,158,404)	\$ -	\$ (3,158,404)	\$ 135,183	23.4	\$ 43,750	17.6	316,027.0
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

10 Conclusion and Recommendations

10.1 Subject and Purpose

This report presents the preliminary findings of a Smith Engineering study that Brookfield Properties and NYSERDA commissioned to conduct a community heat pump feasibility study at the 250 Vesey St. facility in New York, NY.

10.2 Conclusion

The complex experiences a significant amount of simultaneous heating and cooling—creating an opportunity for a heat pump application. Initially, the central heat pump option appeared more attractive since it could effectively leverage the site’s diversity. The site offers substantial carbon reduction potential. As the carbon content of electricity decreases, these measures will improve annual carbon reduction and account for a higher percentage of the total site reduction.

Table 90. Financial Summary of Modifications

Breakdown of Modification Options														
Option	Description	Estimated Annual Steam Reduction	Estimated Annual Electric Reduction	2024 Estimated Annual CO2 Reduction			First Cost - Estimated			Estimated Annual Energy Cost Savings		With CO2 Tax (2024)		Net Present Value
		Energy	Energy	Steam	Electric	Total	First Cost	Rebate	Net Capex After Rebate	Measure Savings	SPB	CO2 Tax	SPB	20 Year NPV
		Mlbs	kWh	MT CO2e	MT CO2e	MT CO2e	\$	\$	\$	\$	Years	\$	Years	\$
M.1	Central Heat Pump	113,650	-4,739,842	4,800	-1,370	3,430	\$ (35,657,579)	\$ -	\$ (35,657,579)	\$ 2,971,526	12.0	\$ 919,313	9.2	\$ 39,892,695
M.2	Tower A Heat Pump	5,954	-271,945	251	-79	173	\$ (3,741,971)	\$ -	\$ (3,741,971)	\$ 118,944	31.5	\$ 46,334	22.6	\$ (532,686)
M.3	Tower B Heat Pump	11,324	-467,814	478	-135	343	\$ (3,210,609)	\$ -	\$ (3,210,609)	\$ 267,538	12.0	\$ 91,940	8.9	\$ 3,769,543
M.4	Tower C Heat Pump	2,974	-64,663	126	-19	107	\$ (3,400,390)	\$ -	\$ (3,400,390)	\$ 89,458	38.0	\$ 28,659	28.7	-1,106,849.5
M.5	Tower D Heat Pump	4,578	-104,168	193	-30	163	\$ (3,158,404)	\$ -	\$ (3,158,404)	\$ 135,183	23.4	\$ 43,750	17.6	316,027.0
Cost Estimates														
X	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Based on some contractor estimates and more refined scope												
	Investment Grade / Proposal	Fully developed scope, subcontractor pricing, ready for design build proposal and management approval												
Savings Calculations and Paybacks														
	Placeholder	Based on past experience on similar projects. This number could go down or up substantially based on further development.												
	Estimates / Budgetary	Energy model build based on final scope.												
X	Investment Grade / Rebate Ready	Fully developed energy model calibrated with full QA/QC												

10.3 Recommendations

Smith Engineering recommends proceeding to the next task: determining the system costs.

11 Results Impact and Lessons Learned

11.1 Results Impact

The opportunity to reduce carbon emissions and operating expenses is significantly greater than initially considered with the plant option. The Central Plant option offers substantial advantages over the distributed model because it directs heating and cooling where needed. In the distributed model, operating a heat pump is less beneficial when Tower C is cooling-dominant and Tower B is heating-dominant compared to the Central Plant model.

11.2 Key Takeaways

The following lessons highlight key takeaways from the project:

- **Takeaway 1:** The large, complex site posed complex challenges in distributing large-diameter piping across the 14-acre campus. The team conducted an extensive site investigation to address these challenges.
- **Takeaway 2:** The team found the cost-benefit of individual loads to be highly beneficial. A 7,000-MBH load near the piping mains provided an attractive return on investment per ton of CO₂ saved. In contrast, a 70-MBH load, located multiple levels away and thousands of feet from the mains, resulted in a much lower return on investment. The embodied carbon for the latter may never be recouped through the carbon savings.
- **Takeaway 3:** The original design of the facility, with 160°F–140°F hot water on the non-steam heating units, significantly improved the project's feasibility.
- **Takeaway 4:** The team faced challenges fitting the larger York CYK heat pump into an existing New York City building.
- **Takeaway 5:** Understanding the existing control strategies for all heating equipment allowed the team to assess the feasibility more accurately when determining load-by-load return on investment.

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